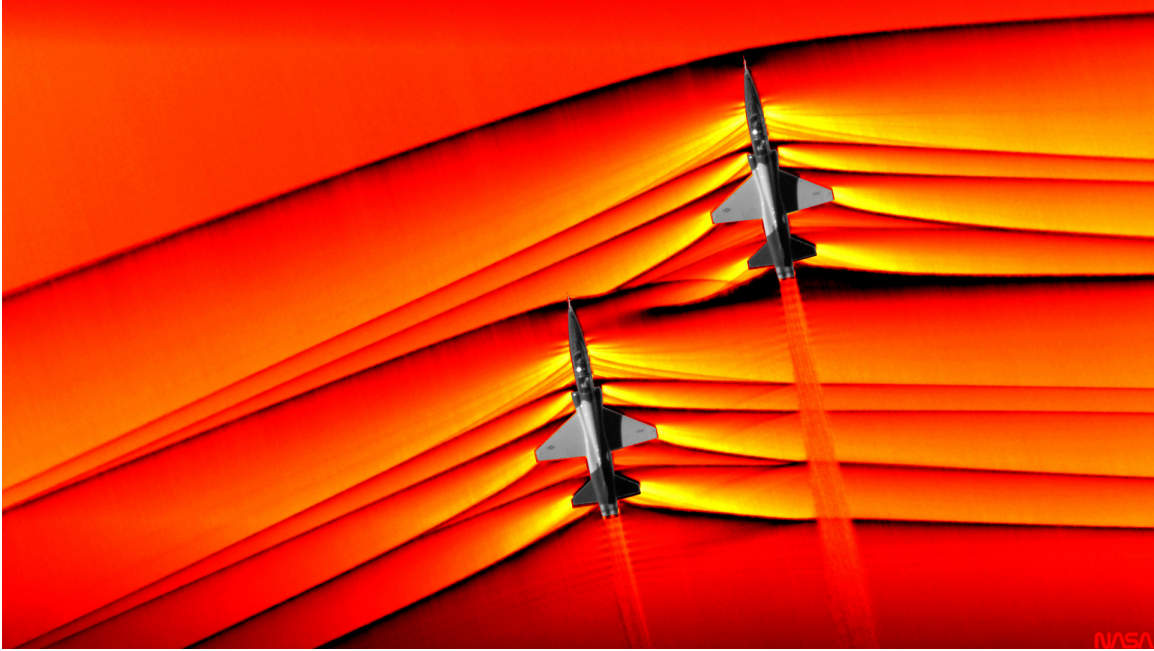


Compressible Flow – Introduction



<https://imgur.com/r/wallpapers/ufMF4X2>

Compressible Flow – Introduction

Thermodynamics Review

Some Definitions

u := specific internal energy

h := specific enthalpy = $u + p/\rho$

c_v := specific heat at constant volume

c_p := specific heat at constant pressure

k := specific heat ratio = c_p/c_v ($k = 1.4$ for air)

R := gas constant ($R_{\text{air}} = 287 \text{ J}/(\text{kg}\cdot\text{K}) = 53.3 \text{ (ft}\cdot\text{lb}_f)/(\text{lb}_m\cdot\text{degR})$)

adiabatic := no heat transfer

reversible := system and surroundings can be restored exactly to their initial states

isentropic := no change in entropy, i.e., $\Delta s = 0$

adiabatic and reversible \Rightarrow isentropic

T - s diagrams are frequently used to illustrate processes

Ideal Gas Relations

$p = \rho RT$ (use absolute pressure and temperature; R is the gas constant, not the universal gas constant))

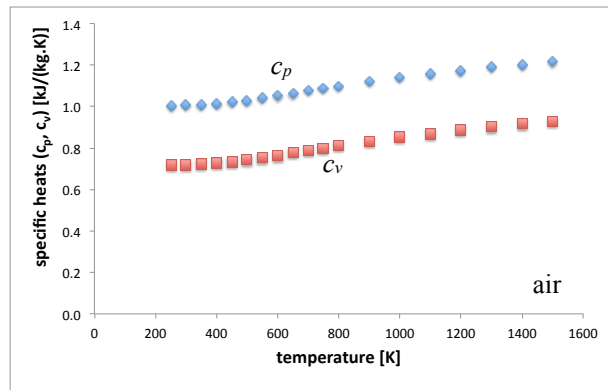
$$u_2 - u_1 = \int_{T_1}^{T_2} c_v(T) dT$$

$$h_2 - h_1 = \int_{T_1}^{T_2} c_p(T) dT$$

$$c_p = c_v + R$$

$$c_p = \frac{kR}{k-1} \quad \text{and} \quad c_v = \frac{R}{k-1}$$

$$ds = c_v(T) \frac{dT}{T} - R \frac{d\rho}{\rho} = c_p(T) \frac{dT}{T} - R \frac{dp}{p}$$



Perfect Gas Relations

perfect gas = ideal gas with constant specific heats

$$u_2 - u_1 = c_v(T_2 - T_1) \quad (c_{v,\text{air}} = 718 \text{ J}/(\text{kg}\cdot\text{K}) = 133 \text{ (ft}\cdot\text{lb}_f)/(\text{lb}_m\cdot\text{degR}))$$

$$h_2 - h_1 = c_p(T_2 - T_1) \quad (c_{p,\text{air}} = 1005 \text{ J}/(\text{kg}\cdot\text{K}) = 187 \text{ (ft}\cdot\text{lb}_f)/(\text{lb}_m\cdot\text{degR}))$$

$$s_2 - s_1 = c_v \ln \frac{T_2}{T_1} - R \ln \frac{\rho_2}{\rho_1} = c_p \ln \frac{T_2}{T_1} - R \ln \frac{p_2}{p_1}$$

isentropic process involving a perfect gas: $\frac{T_2}{T_1} = \left(\frac{\rho_2}{\rho_1}\right)^{k-1}$, $\frac{p_2}{p_1} = \left(\frac{T_2}{T_1}\right)^{\frac{k}{k-1}}$, $\frac{p_2}{p_1} = \left(\frac{\rho_2}{\rho_1}\right)^k$

Compressible Flow – Introduction

1st Law for steady flow of a gas with one inlet and one outlet and uniform inlet and outlet properties

$$\underbrace{\frac{d}{dt} \int_{CV} e \rho dV}_{=0 \text{ (steady)}} + \int_{CS} \left(h + \frac{1}{2} V^2 + \underbrace{gz}_{\substack{\text{negligible compared} \\ \text{to other terms since} \\ \text{its a gas}}} \right) \underbrace{(\rho \mathbf{u}_{rel} \cdot d\mathbf{A})}_{=\dot{m}} = \dot{Q}_{\text{into CV}} + \dot{W}_{\text{on CV}}$$

From COM: $\dot{m} = \dot{m}_{\text{in}} = \dot{m}_{\text{out}}$. Combining with 1st Law:

$$\Delta \left(h + \frac{1}{2} V^2 \right) = q_{\text{into CV}} + w_{\text{on CV}}$$

For an adiabatic ($q_{\text{into}} = 0$) flow with no work other than pressure work ($w_{\text{on}} = 0$):

$$\Delta \left(h + \frac{1}{2} V^2 \right) = 0 \quad \text{or} \quad h + \frac{1}{2} V^2 = \text{constant}$$

For a perfect gas ($\Delta h = c_p \Delta T$):

$$c_p T + \frac{1}{2} V^2 = \text{constant}$$