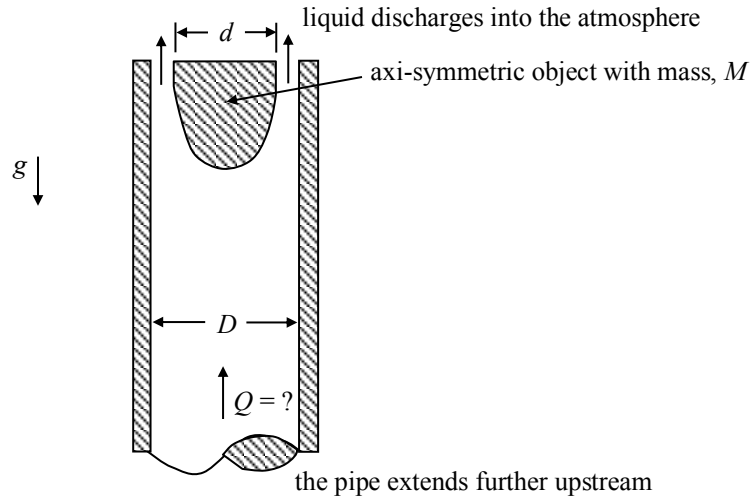
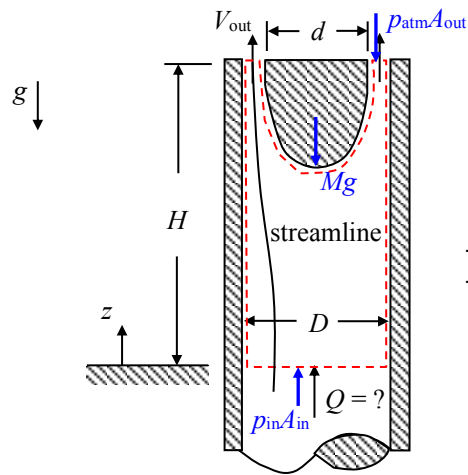


The axi-symmetric object shown below is placed in the end of a vertical circular pipe of inner diameter, D . A liquid with density, ρ , is pumped upward through the pipe and discharges to the atmosphere. Neglecting viscous effects, determine the volume flow rate, Q , of the liquid needed to support the object in the position shown in terms of d , D , g , ρ , and M .



SOLUTION:

Apply conservation of mass to the control volume shown below.



- Let: $A_{in} = \pi D^2/4$ and $A_{out} = \pi(D^2-d^2)/4$.
- Choose H such that it is much larger than the size of the object ($\Rightarrow V_{CV} \approx A_{in}H$).

$$\frac{d}{dt} \int_{CV} \rho dV + \int_{CS} \rho \mathbf{u}_{rel} \cdot d\mathbf{A} = 0 \quad (1)$$

where

$$\frac{d}{dt} \int_{CV} \rho dV = 0 \quad (\text{steady flow})$$

$$\int_{CS} \rho \mathbf{u}_{rel} \cdot d\mathbf{A} = -\rho Q + \rho V_{out} A_{out}$$

Substituting and simplifying gives:

$$-\rho Q + \rho V_{out} A_{out} = 0$$

$$V_{out} = \frac{Q}{A_{out}} \quad (2)$$

Apply the linear momentum equation in the z -direction to the same control volume. Use the fixed frame of reference shown in the figure.

$$\frac{d}{dt} \int_{CV} u_z \rho dV + \int_{CS} u_z (\rho \mathbf{u}_{rel} \cdot d\mathbf{A}) = F_{B,z} + F_{S,z} \quad (3)$$

where

$$\frac{d}{dt} \int_{CV} u_z \rho dV = 0 \quad (\text{steady flow})$$

$$\begin{aligned} \int_{CS} u_z (\rho \mathbf{u}_{rel} \cdot d\mathbf{A}) &= -m \left(\frac{Q}{A_{in}} \right) + m V_{out} \\ &= \rho Q \left(\frac{Q}{A_{out}} - \frac{Q}{A_{in}} \right) \\ &= \rho Q^2 \left(\frac{A_{in} - A_{out}}{A_{in} A_{out}} \right) \end{aligned}$$

(Note that Eqn. (2) was used in simplifying the momentum flux term.)

$$F_{B,z} = -\rho V_{CV} g \approx -\rho A_{in} H g \quad (H \text{ is chosen to be much larger than the object size.})$$

$$F_{S,z} = p_{in} A_{in} - Mg \quad (\text{use gage pressures so } p_{out} = p_{atm} = 0)$$

Substitute and simplify.

$$\rho Q^2 \left(\frac{A_{in} - A_{out}}{A_{in} A_{out}} \right) = -\rho A_{in} H g + p_{in} A_{in} - Mg \quad (4)$$

To determine p_{in} , apply Bernoulli's equation along a streamline from the inlet to the outlet.

$$\left(p + \frac{1}{2} \rho V^2 + \rho g z \right)_{in} = \left(p + \frac{1}{2} \rho V^2 + \rho g z \right)_{out} \quad (5)$$

where

$$p_{in} = ? \quad p_{out} = 0 \quad (\text{gage pressure})$$

$$V_{in} = \frac{Q}{A_{in}} \quad V_{out} = \frac{Q}{A_{out}} \quad (\text{from Eqn. (2)})$$

$$z_{in} = 0 \quad z_{out} = H$$

Substitute and simplify.

$$p_{in} = \frac{1}{2} \rho Q^2 \left(\frac{1}{A_{out}^2} - \frac{1}{A_{in}^2} \right) + \rho g H \quad (6)$$

Substitute Eqn. (6) into Eqn. (4) and simplify.

$$\begin{aligned}\rho Q^2 \left(\frac{A_{\text{in}} - A_{\text{out}}}{A_{\text{in}} A_{\text{out}}} \right) &= -\rho A_{\text{in}} H g + \left[\frac{1}{2} \rho Q^2 \left(\frac{1}{A_{\text{out}}^2} - \frac{1}{A_{\text{in}}^2} \right) + \rho g H \right] A_{\text{in}} - M g \\ &= \frac{1}{2} \rho Q^2 \left(\frac{1}{A_{\text{out}}^2} - \frac{1}{A_{\text{in}}^2} \right) A_{\text{in}} - M g\end{aligned}$$

$$\rho Q^2 \left[\left(\frac{A_{\text{in}} - A_{\text{out}}}{A_{\text{in}} A_{\text{out}}} \right) - \frac{1}{2} \left(\frac{A_{\text{in}}^2 - A_{\text{out}}^2}{A_{\text{in}}^2 A_{\text{out}}^2} \right) A_{\text{in}} \right] = -M g$$

$$\rho Q^2 \left[\left(\frac{A_{\text{in}} - A_{\text{out}}}{A_{\text{in}} A_{\text{out}}} \right) - \frac{1}{2} \left(\frac{A_{\text{in}}^2 - A_{\text{out}}^2}{A_{\text{in}} A_{\text{out}}^2} \right) \right] = -M g$$

$$\rho Q^2 \left(\frac{2 A_{\text{in}} A_{\text{out}} - 2 A_{\text{out}}^2 - A_{\text{in}}^2 + A_{\text{out}}^2}{2 A_{\text{in}} A_{\text{out}}^2} \right) = -M g$$

$$\rho Q^2 \left(\frac{-A_{\text{in}}^2 + 2 A_{\text{in}} A_{\text{out}} - A_{\text{out}}^2}{2 A_{\text{in}} A_{\text{out}}^2} \right) = -M g$$

$$\rho Q^2 \left[\frac{(A_{\text{in}} - A_{\text{out}})^2}{2 A_{\text{in}} A_{\text{out}}^2} \right] = M g$$

$$\boxed{\therefore Q = \sqrt{\frac{M g}{\rho \frac{(A_{\text{in}} - A_{\text{out}})^2}{2 A_{\text{in}} A_{\text{out}}^2}}} = \frac{1}{\left(\frac{A_{\text{in}}}{A_{\text{out}}} - 1 \right)} \sqrt{\frac{2 M g A_{\text{in}}}{\rho}}} \quad \text{where } A_{\text{in}} = \frac{\pi D^2}{4} \quad \text{and} \quad A_{\text{out}} = \frac{\pi (D^2 - d^2)}{4}} \quad (7)$$