The pressure waves created by a rapid change of flow in a water line are referred to as water-hammers. To analyze the behavior of this phenomenon, consider a fluid flowing at speed U in a *rigid* pipe. The flow is stopped by a sudden closure of a valve. The pressure and the density of the fluid near the valve are suddenly increased by an amount  $\Delta p$  and  $\Delta \rho$ , respectively, and a pressure wave propagates upstream of the valve with speed, a.

a. Show that the increase in pressure,  $\Delta p$ , and the wave speed, *a*, are related by:

$$\Delta p = \rho U (U + a)$$
$$a(U + a) = \frac{\Delta p}{\Delta \rho}$$

b. The bulk modulus  $K = \rho (dp/d\rho)$  is  $43 \times 10^6 \text{ lb}_f/\text{ft}^2$  for water. Compute the wave speed *a* in a rigid pipe and  $\Delta p$  due to a sudden stoppage of water flowing with a speed of 1 ft/s. You may assume that the pressure change across the wave is sufficiently weak to be considered an acoustic wave for the given conditions.



## SOLUTION:

Apply conservation of mass and the linear momentum equation to a control volume surrounding the pressure wave.



Change the frame of reference so that wave appears stationary.

$$u = U + a$$

$$\rho$$

$$p$$

$$p$$

$$\mu = a$$

$$\rho + \Delta \rho$$

$$p + \Delta p$$

$$\rightarrow X$$

Apply conservation of mass to the control volume.

$$\frac{d}{dt} \int_{CV} \rho \, dV + \int_{CS} \rho \mathbf{u}_{rel} \cdot d\mathbf{A} = 0 \tag{1}$$

where

$$\frac{d}{dt} \int_{CV} \rho \, dV = 0 \quad \text{(steady in the given frame of reference)} \tag{2}$$

$$\int_{CS} \rho \mathbf{u}_{rel} \cdot d\mathbf{A} = -\rho (U+a)A + (\rho + \Delta \rho)aA$$
(3)

Combine and simplify.

$$-\rho(U+a)A + (\rho + \Delta\rho)aA = 0 \tag{4}$$

$$\rho(U+a) = (\rho + \Delta \rho)a \tag{5}$$

Apply the linear momentum in the *x*-direction using an inertial frame of reference.

$$\frac{d}{dt} \int_{CV} u_X \rho \, dV + \int_{CS} u_X \left( \rho \mathbf{u}_{rel} \cdot d\mathbf{A} \right) = F_{B,X} + F_{S,X} \tag{6}$$

where

$$\frac{d}{dt} \int_{CV} u_X \rho \, dV = 0 \quad \text{(steady in the given frame of reference)} \tag{7}$$

$$\int u_X (\rho \mathbf{u}_{\text{rel}} \cdot d\mathbf{A}) = -\rho (U+a)^2 A + (\rho + \Delta \rho) a^2 A \tag{8}$$

$$\int_{CS} u_X(p\mathbf{u}_{Rl}, u\mathbf{A}) = -p(0 + u) A + (p + \Delta p)u A$$
(0)

$$F_{B,X} = 0 \tag{9}$$

$$F_{s,x} = pA - (p + \Delta p)A \tag{10}$$

Combine and simplify.

$$-\rho(U+a)^{2}A + (\rho + \Delta\rho)a^{2}A = pA - (p + \Delta p)A$$
<sup>(11)</sup>

$$-\rho(U+a)^{2} + (\rho + \Delta\rho)a^{2} = -\Delta p \tag{12}$$

$$-\rho(U+a)^2 + \rho(U+a)a = -\Delta p \quad (\text{making use of Eq. (5)})$$
(13)

$$\rho(U+a)[(U+a)-a] = \Delta p \tag{14}$$

$$\Delta p = \rho U(U+a) \tag{15}$$

Note that if  $U \ll a$ , which is typically the case, then Eq. (15) becomes,  $\Delta p = \rho U a$ (16) Re-arranging Eq. (15) to solve for  $\rho$  gives,

$$\rho = \frac{\Delta p}{\rho U(U+a)} \tag{17}$$

Substitute this relation into Eq. (5) and simplify.

$$(U+a) = \left(1 + \frac{\Delta\rho}{\rho}\right)a\tag{18}$$

$$\left(U+a\right) = \left[1 + \frac{U(U+a)\Delta\rho}{\Delta p}\right]a\tag{19}$$

$$\frac{(U+a)}{a} = 1 + U(U+a)\frac{\Delta\rho}{\Delta p}$$
(20)

$$\frac{\Delta\rho}{\Delta p} = \frac{1}{U(U+a)} \left[ \frac{(U+a)}{a} - 1 \right]$$
(21)

$$\frac{\Delta p}{\Delta \rho} = U(U+a) \left[ \frac{a}{(U+a)-a} \right]$$
(22)

$$\left|\frac{\Delta p}{\Delta \rho} = a(U+a)\right| \tag{23}$$

Again, if  $U \ll a$ , then this relation becomes,

$$\frac{\Delta p}{\Delta \rho} = a^2 \tag{24}$$

In addition, if the wave is weak, meaning that the change in pressure and density across the wave are infinitesimally small, i.e., a sound wave, then Eq. (24) becomes,

$$\frac{dp}{d\rho} = a^2 \tag{25}$$

The bulk modulus is defined as,

$$K \equiv \rho \frac{dp}{d\rho} \,, \tag{26}$$

Since the wave is assumed to be an acoustic wave for the given conditions (refer to Eq. (25)),

$$a^{2} = \frac{dp}{d\rho} \quad \Rightarrow \quad a = \sqrt{\frac{K}{\rho}} \tag{27}$$

The pressure change across the wave is found from Eq. (15). Using the given data,

 $K = 43*10^6 \text{ lb}_{\text{f}}/\text{ft}^2$   $\rho = 1.94 \text{ slug}/\text{ft}^3$ U = 1 ft/s

 $\Rightarrow$  a = 4710 ft/s and  $\Delta p = 9.14 \times 10^3$  psf = 63.4 psi

Note that  $U \ll a$  and  $d\rho / \rho \ll 1$ , consistent with the assumption of an acoustic wave.