# **Chapter 2: Kinematics of Deformation**

In this chapter, we will study how bodies/structures move/deform and how can this motion/deformation be described mathematically. (In general, bodies/structures move/deform when forces are acting on them, but we are not concerned (for now) about the causes of this motion/deformation.)

We are concerned only about describing the motion/deformation.

Motion / deformation can be:

- Fast: Dynamic effects are important: Acceleration, inertia etc.
- Slow: Dynamic effects can be neglected: (quasi-)static. Or after steady state has been achieved.

 $\mathcal{E}_{\text{ENG}} \equiv \frac{\Delta l}{l_0} = \frac{l - l_0}{l_0} = \lambda - 1$ 

 $E = \frac{1}{2} \left( \frac{l^2 - l_0^2}{l^2} \right) = \frac{1}{2} \left( \frac{\lambda^2 - 1}{\lambda^2} \right) = \frac{1}{2} \left( \lambda - 1 \right) \left( \lambda + 1 \right)$ 

 $e = \frac{1}{2} \left( \frac{l^2 - l_0^2}{l^2} \right) = \frac{1}{2} \left( 1 - \frac{1}{\lambda^2} \right) = \frac{1}{2} \left( 1 - \frac{1}{\lambda} \right) \left( 1 + \frac{1}{\lambda} \right)$ 

 $E_{\text{NAT}}/TRUE = \frac{\Delta k}{k} = \frac{l-l_0}{k} = 1 - \frac{1}{\lambda}$ 

# Stretch of a material in 1D

Consider a uniform bar of some material before and after motion/deformation.

$$stretch(\lambda)$$
  $l = \lambda l_o$  i.e.  $\lambda = \frac{l}{l_o}$ 

Conventional notions of strain in 1D

- Engineering strain
- Natural (true) strain
- Green-Lagrangian strain
- Almansi-Eulerian strain
- Logarithmic strain

Note: 
$$\sum_{i=1}^{\infty} \frac{\Delta l_i}{l_i} = \int_{0}^{l} \frac{1}{l} dl = ln(l) - ln(l_0) = ln(\frac{l}{l_0}) = ln(\lambda)$$

 $\mathcal{E}_{ln} = \ell_n(\lambda)$ 

All these are <u>average measures of strain</u> (for the entire bar) that are applicable for cases when the bar has <u>uniform stretching</u>.



Deformation maps  $\phi(z)$  and displacement vector fields u(z) in 3D

Generalizes the 1D concept of the map to 3D. Takes the position vector z of any point in the undeformed configuration and Return its position in the deformed configuration.  $\underline{x} = \underline{\phi}(\underline{z})$ 

 $\underline{\mathcal{U}}(\underline{z}) = \underline{\mathcal{I}} - \underline{z}$ 

$$x = \varphi(z)$$

$$x = z + u(z)$$
Display a notative field.

ormation

Examples of deformation maps:

imples of deformation maps:  
(i) Translation 
$$\chi = \chi + u$$
  
 $\begin{cases} \chi_1 \\ \chi_2 \\ \chi_3 \end{cases} = \begin{cases} Z_1 \\ Z_2 \\ Z_3 \end{cases} + \begin{cases} u_1 \\ u_2 \\ u_3 \end{cases} = \begin{cases} z_1 \\ z_2 \\ u_3 \end{cases}$ 

(ii) Uniform Expansion in all 3 directions

$$\mathcal{X} = \oint(\overline{Z}) = \mathcal{X}\overline{Z}$$
$$\begin{cases} \mathcal{X}_{1} \\ \mathcal{X}_{2} \\ \mathcal{X}_{3} \end{cases} = \mathcal{X} \begin{cases} \overline{Z}_{1} \\ \overline{Z}_{2} \\ \overline{Z}_{3} \end{cases}$$

(iii) Approximate bending deformation

$$\frac{\mathcal{X}}{\mathcal{X}} = \underbrace{\begin{array}{l} \not {\boldsymbol{x}} \\ \mathcal{X}_{2} \\ \mathcal{X}_{2} \\ \mathcal{X}_{3} \end{array}}_{\begin{array}{l} \not {\boldsymbol{x}}_{2} \\ \mathcal{X}_{3} \end{array}} = \begin{cases} \varphi_{1} \left( \boldsymbol{z}_{1}, \boldsymbol{z}_{2}, \boldsymbol{z}_{3} \right) \\ \varphi_{2} \left( \boldsymbol{z}_{1}, \boldsymbol{z}_{2}, \boldsymbol{z}_{3} \right) \\ \varphi_{3} \left( \boldsymbol{z}_{1}, \boldsymbol{z}_{2}, \boldsymbol{z}_{3} \right) \end{cases}$$
$$\left( \begin{array}{l} \mathcal{X}_{1} \\ \mathcal{X}_{2} \\ \mathcal{X}_{3} \\ \mathcal{X}_{3} \end{array} \right) = \begin{cases} \boldsymbol{z}_{1} - \frac{\boldsymbol{z}_{1}}{\boldsymbol{L}} \boldsymbol{\Delta}_{1} \\ \boldsymbol{z}_{2} - \boldsymbol{\Delta}_{2} \operatorname{sinc} \left( \frac{\boldsymbol{z}_{1}, \boldsymbol{\pi} \right) \\ \boldsymbol{z}_{3} \end{array} \right)$$

Verify: for a point on the mid cross-section:







(5)

(iv) Pure bending of a prismatic cantilever beam: (pages 250-255, Timoshenko & Goodier)

(iv) Pure bending of a prismatic cantilever beam:  
(pages 250-255, Timoshenko & Goodier)  

$$\underline{\mathcal{X}} = \underline{\mathcal{Z}} + \underline{\mathcal{U}}(\underline{\mathcal{Z}})$$

$$\underline{\mathcal{X}} \sim \begin{bmatrix} \mathcal{X}_{1} \\ \mathcal{H}_{2} \\ \mathcal{H}_{3} \end{bmatrix} \quad \underline{\mathcal{Z}} \sim \begin{bmatrix} \underline{\mathcal{Z}}_{1} \rightarrow \mathcal{H} \\ \underline{\mathcal{Z}}_{2} \rightarrow \underline{\mathcal{Y}} \\ \underline{\mathcal{Z}}_{2} \rightarrow \underline{\mathcal{Y}} \\ \underline{\mathcal{Z}}_{2} \rightarrow \underline{\mathcal{Z}} \end{bmatrix} \quad \underline{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ v = -\frac{vxy}{R} \\ w = \frac{xz}{R} \end{bmatrix} \quad \underline{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{1}{2R} [z^{2} + v(x^{2} - y^{2})] \\ \mathbf{\mathcal{U}} \sim \begin{bmatrix} u = -\frac{$$

Note: For the lateral surfaces of the beam:

$$\begin{aligned} \varkappa_2 &= \pm b + v = \pm b \left( 1 - \frac{\nu x}{R} \right) \\ \varkappa_1 &= \pm a + u = \pm a - \frac{1}{2R} \left[ c^2 + \nu (a^2 - y^2) \right] \end{aligned}$$

Lagrangian vs. Eulerian descriptions of motion/deformation Note: The displacement field can be expressed as:



 $u = u_{L}(z) = x(z) - z = \phi(z) - z$  (Lagrangian)  $u = u_{E}(x) = x - z(x) = x - \phi^{-1}(x)$  (Eulerian)

Stretch along a curve in 3D To generalize the ideas of stretch and strain to 3D consider a curve C embedded in a structure as it deforms: curve parameter  $\underline{Z}(S) \sim \begin{pmatrix} z_1(S) \\ z_2(S) \\ z_2(S) \\ z_2(S) \end{pmatrix} \begin{pmatrix} c_1 = \{ \underline{Z} : \underline{Z} \sim \{ \begin{smallmatrix} 0 \\ S \\ S \end{bmatrix} \}; \quad S \in [0, 1] \} \\ C_2 = \{ \underline{Z} : \underline{Z} \sim \{ \begin{smallmatrix} R \sin(S/L & 2\pi) \\ R \cos(S/L & 2\pi) \}; \quad S \in [0, L] \} \\ K \cos(S/L & 2\pi) \}; \quad S \in [0, L] \}$ Examples: ... infinitely many possible curves. The deformed locations of these curves are given by: To find the stretch along the curve: Tangent to the undeformed curve:  $rac{d z}{d z}$ Tangent to the **m**deformed curve:  $\frac{dx}{ds} = \lim_{\Delta s \to 0} \frac{x(s + \Delta s) - x(s)}{\Delta s}$  $\frac{d\underline{z}}{dz} = \lim_{\Delta s \to 0} \frac{\underline{z}(s + \Delta s) - \underline{z}(s)}{\Delta s}$  $\left\| \frac{d\underline{x}}{d\underline{s}} \right\| = \lambda(\underline{s}) \left\| \frac{d\underline{z}}{d\underline{s}} \right\|$ The 1-D stretch at a point *P* along the curve *C* is given by:

Note that the stretch at P in an arbitrary direction can be obtained by using a different curve passing though P.

Also note:  

$$\frac{d\mathcal{A}}{ds} \sim \begin{cases} \frac{d\mathcal{A}_{1}}{ds} \\ \frac{d\mathcal{A}_{2}}{ds} \\ \frac{d\mathcal{A}_{2}}{ds} \\ \frac{d\mathcal{A}_{2}}{ds} \end{cases} = \begin{cases} \frac{\partial\mathcal{A}_{1}}{\partial z_{1}} \cdot \frac{dz_{1}}{ds} + \frac{\partial\mathcal{A}_{1}}{\partial z_{2}} \cdot \frac{dz_{2}}{ds} + \frac{\partial\mathcal{A}_{1}}{\partial z_{3}} \cdot \frac{dz_{3}}{ds} \\ \frac{\partial\mathcal{A}_{2}}{\partial z_{1}} \cdot \frac{dz_{1}}{ds} + \frac{\partial\mathcal{A}_{2}}{\partial z_{2}} \cdot \frac{dz_{2}}{ds} + \frac{\partial\mathcal{A}_{1}}{\partial z_{3}} \cdot \frac{dz_{3}}{ds} \\ \frac{\partial\mathcal{A}_{2}}{\partial z_{1}} \cdot \frac{dz_{1}}{ds} + \frac{\partial\mathcal{A}_{3}}{\partial z_{2}} \cdot \frac{dz_{2}}{ds} + \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} \cdot \frac{dz_{3}}{ds} \\ \frac{\partial\mathcal{A}_{2}}{\partial z_{1}} \cdot \frac{dz_{1}}{ds} + \frac{\partial\mathcal{A}_{3}}{\partial z_{2}} \cdot \frac{dz_{2}}{ds} + \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} \cdot \frac{dz_{3}}{ds} \end{cases} \\ \end{cases}$$

$$\frac{d\mathcal{A}}{ds} \sim \begin{bmatrix} \frac{\partial\mathcal{A}_{1}}{\partial z_{1}} & \frac{\partial\mathcal{A}_{1}}{\partial z_{2}} & \frac{\partial\mathcal{A}_{1}}{\partial z_{2}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} \\ \frac{\partial\mathcal{A}_{2}}{\partial z_{2}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} \\ \frac{\partial\mathcal{A}_{3}}{\partial z_{1}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{2}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} \\ \frac{\partial\mathcal{A}_{3}}{\partial z_{1}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{2}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} \\ \frac{\partial\mathcal{A}_{3}}{\partial z_{1}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{2}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} \\ \frac{\partial\mathcal{A}_{3}}{\partial z_{3}} & \frac{\partial\mathcal{A}_{3}}{\partial z_{2}} & \frac{\partial\mathcal{A}_{2}}{\partial z_{3}} \\ \frac{\partial\mathcal{A}_{3}}{\partial z_{3}} & \frac{\partial\mathcal{A}_{3}}{\partial z_{3}} & \frac{\partial\mathcal{A}_{3}}{\partial z_{3}} \\ \frac{\partial\mathcal{A}_{3}}{\partial z_{3}} & \frac{\partial\mathcal{A}_{3}}{\partial z_{3}} & \frac{\partial\mathcal{A}_{3}}{\partial z_{3}} \\ \frac{\partial\mathcal{A}_{3}}{$$

Deformation gradient tensor F

The relationship for stretches in arbitrary directions in 3D can be expressed more compactly as:

$$\frac{dx}{ds} = \frac{dx_i}{ds} \underbrace{g_i}_{z_i} = \frac{\partial x_i}{\partial z_j} \cdot \frac{dz_j}{ds} \underbrace{g_i}_{z_i} = \begin{bmatrix} \frac{\partial x_i}{\partial z_j} (\underbrace{e_i} \otimes \underbrace{g_i}) \\ \frac{\partial z_i}{\partial z_j} \end{bmatrix} \begin{pmatrix} \frac{dz_k}{ds} \underbrace{g_k} \\ \frac{dx}{ds} \underbrace{g_k} \end{pmatrix}$$

$$\Rightarrow \frac{dx}{ds} = \underbrace{F}_{ds} \frac{dz}{ds} \quad \text{where} \quad F = \underbrace{\nabla_z}_{z_i} \underbrace{x}_{z_i} = (\underbrace{x} \otimes \underbrace{\nabla_z}_{z_i}) = \frac{\partial \underline{x}_i}{\partial z_i}$$

$$\text{Since} \quad x = \oint(z) \Rightarrow F_{z_i}(z) = \underbrace{\nabla_z}_{z_i} \underbrace{\phi(z)}_{z_i} = (\underbrace{p(z)}_{z_i} \otimes \underbrace{\nabla_z}_{z_i}) = \frac{\partial \underline{p(z)}}{\partial z_i}$$

In components:

$$F_{n}(z) = \varphi_{i,j}(\underline{e}; \otimes \underline{g}_{j}) = \frac{\partial \varphi_{i}(z_{1}, z_{2}, z_{3})}{\partial z_{j}}(\underline{e}; \otimes \underline{g}_{j})$$

A useful interpretation of *F* 





Examples:

(i) Translation 
$$\mathcal{Z} = \mathcal{Q}(\mathbf{z}) = \mathbf{z} + \mathbf{u}$$
  

$$\begin{cases} \mathcal{H}_{1} \\ \mathcal{H}_{2} \\ \mathcal{H}_{3} \end{cases} = \begin{cases} \mathcal{Z}_{1} + \mathcal{U}_{1} \\ \mathcal{Z}_{2} + \mathcal{U}_{2} \\ \mathcal{Z}_{3} + \mathcal{U}_{3} \end{cases} \qquad \mathbf{F}(\mathbf{z}) \sim \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$



(ii) Uniform Expansion in all 3 directions  $\begin{cases} \alpha l_1' \\ \alpha l_2 \\ \alpha l_3 \\$ 





€1,9,

(iii) Approximate bending deformation  $\chi = \oint (\Xi)$ 

 $\begin{cases} \chi_{1} \\ \chi_{2} \\ \chi_{3} \end{cases} = \begin{cases} \Xi_{1} - \frac{\Xi_{1}}{L} \Delta_{1} \\ \Xi_{2} - \Delta_{2} \sin\left(\frac{\Xi_{1}}{L} \Pi\right) \\ \end{array}$ 

D

Stretch and Strain in arbitrary directions in 3D

Using the interpretation of F as: dx = f, dzwe can calculate the stretch in any arbitrary direction nof the undeformed configuration.

Using the interpretation of F as: 
$$d\underline{x} = \underline{F}, d\underline{z}$$
  
we can calculate the stretch in any arbitrary direction **n**  
of the undeformed configuration.  
 $\leq 4 + vet oh$   
 $\lambda(\underline{n}) = \frac{\|\underline{F}, \underline{n}\|}{\|\underline{F}, \underline{n}\|} = \sqrt{(\underline{F}, \underline{n}) \cdot (\underline{F}, \underline{n})}$   
Atternatively,  $\frac{\lambda^2}{(\underline{n})} = \underline{n} \cdot \underline{E}^T \underline{F}, \underline{n}$   
 $\frac{\lambda^2}{(\underline{n})} = \underline{n} \cdot \underline{S}, \underline{n}$   
 $\frac{\lambda}{(\underline{n})} = \underline{h} \cdot \underline{S}, \underline{n}$   
 $\frac{\lambda}{(\underline{n})} = \underline{h} \cdot \underline{S}, \underline{n} = \underline{h$ 

$$\Rightarrow \text{ for any } d\underline{z} \text{ vector}:$$

$$d\underline{z} \cdot d\underline{z} = d\underline{x} \cdot d\underline{x} \quad \forall d\underline{z}$$

$$d\underline{z} \cdot d\underline{z} = d\underline{x} \cdot d\underline{x} \quad \forall d\underline{z}$$

$$d\underline{z} \cdot d\underline{z} = (\underline{F}, d\underline{z}) \cdot (\underline{F}, d\underline{z}) = d\underline{z} \cdot (\underline{F}^T \underline{F}, d\underline{z})$$

$$\Rightarrow d\underline{z} \cdot (\underline{F}^T \underline{F}, -\underline{I}) d\underline{z} = 0 \quad \Rightarrow \quad \text{Strain} \propto (\underline{C}, -\underline{I}) \quad \underline{F}_{1} = 1/2 (\underline{C}, -\underline{I})$$

Alternatively:  

$$(\underline{F}^{-1}d\underline{x}) \cdot (\underline{F}^{-1}d\underline{x}) = d\underline{x} \cdot d\underline{x}$$

$$\Rightarrow d\underline{x} \cdot ((\underline{F}^{-1})^T \underline{F}^{-1}d\underline{x}) = d\underline{x} \cdot \underline{I} d\underline{x}$$

$$\Rightarrow d\underline{x} \cdot ((\underline{F}, \underline{F}^T)^{-1} d\underline{x}) = d\underline{x} \cdot \underline{I} d\underline{x}$$

$$\Rightarrow d\underline{x} \cdot ((\underline{F}, \underline{F}^T)^{-1} d\underline{x}) = d\underline{x} \cdot \underline{I} d\underline{x}$$

$$\Rightarrow d\underline{x} \cdot (\underline{I} - \underline{B}^{-1}) d\underline{x} = 0 \quad \text{where} \quad \underline{B} = \underline{F} \underline{F}^T$$

$$\underline{B}: Left \quad Cauchy - Green \quad deformation \quad Tensor$$

$$(or \quad Almansi \quad tensor ; \quad Finger \quad Tensor)$$
Similarly
$$\underbrace{Strain \quad \mathcal{O} \quad \underline{I} - \underline{B}^{-1}}_{Euler - Almansi \quad Strain \quad Tensor \quad (\underline{e})}$$

Examples:

$$e_{2}$$

$$(c) = 2\pi$$

$$z_{1}$$

$$e_{2}$$

$$z_{1}$$

$$z_{2}$$

$$z_{2}$$

 $\mathbf{F} \sim \begin{bmatrix} (1-z_2)\sin z_1 / c_1 + (1-(1-z_2)\cos z_1) / c_2 + (1-z_2)\sin z_1 & \cos z_1 & 0 \\ (1-z_2)\sin z_1 & \cos z_1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$   $\mathbf{F} \sim \begin{bmatrix} (1-z_2)\sin z_1 & \cos z_1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$   $\mathbf{F} \sim \begin{bmatrix} (1-z_2)^2 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$ 

Figure 36 Shearing of base vectors for the example deformation maps (a) simple shear, (b) compound shearing and extension, and (c) pure bending

Physical significance of components of C and E

Recall: 
$$\lambda^{2}(\underline{n}) = \underline{n} \cdot \underline{\zeta} \, \underline{n}$$
  
and  $\frac{1}{2}(\lambda^{2}(\underline{n})-1) = \underline{n} \cdot \underline{\xi}, \underline{m}$ 

Note:  $C_{ij} = g_i \cdot c_{jj}$  $E_{ij} = g_i \cdot c_{jj}$ 



Thus C11, C22, C33 represent "normal" stretches in g1, g2, g3 directions. ( E11, E22, E33 represent "normal" strains in g1 g2 g3 directions.)

Shearing components of C and E  
Shear is usually measured as change  
in angles between tangent vectors.  

$$\begin{array}{c} p_{2,1}p_{1} \\ g_{3} \\$$

If we choose 
$$\underline{m}_1$$
 and  $\underline{m}_2$  to be  $\underline{g}_i$  and  $\underline{g}_j$  basis vectors:  
Shear =  $\overline{m}_2 - \cos^{-1}\left(\frac{\underline{g}_i \cdot \underline{c} \cdot \underline{g}_j}{\lambda \underline{g}_i \lambda \underline{g}_2}\right) = \frac{\overline{m}}{2} - \cos^{-1}\left(\frac{\underline{C}_{ij}}{\sqrt{c_{ii}} \sqrt{c_{ij}}}\right)$  (ij : no sum)

Example:

(simple shear)



Example: (simple extension)







Note: The zero off-diagonal components of C in this case only mean that there is no shearing between the basis vectors  $g_1$ ,  $g_2$  and  $g_3$  of this particular coordinate system.

Clearly there are other pairs of vectors  $n_1$  and  $n_2$  for which there is definite shearing, even for this simple extension problem.

# Principal Deformations and Strains

From the preceding discussion one can see that for any deformation, a small neighborhood of a point deforms in a way that there is both stretching and shearing.

*i.e.* a sphere of arbitrary infinitesimal undeformed tangent vectors dz is mapped to an ellipsoid of deformed tangent vectors dx.

Thus there would be some directions in which the stretching is extremum (maximum / minimum).

However, unlike the effect of a symmetric tensor (where these extremal are not rotated), in this case, the extremal tangent vectors will in general have both stretching and rotation.

To find these directions of extremal stretch note:





 $\lambda^2(\underline{m}) = \underline{m} \cdot \overset{C}{\sim} \underline{m}$ 

To maximize / minimize 
$$\lambda^{2}(\underline{n})$$
 subject to  $||\underline{n}|| = 1$  i.e.  $\underline{n} \cdot \underline{n} = 1$   
Consider the function  $L(\underline{n}, \mathcal{U}) \equiv \lambda^{2}(\underline{n}) - \mathcal{U}(\underline{n} \cdot \underline{n} - 1)$   
Now, to extremize :  $\frac{\partial L}{\partial \underline{n}} = \frac{C}{n} - \mathcal{U}\underline{m} = \underline{O} \Rightarrow \frac{C}{n} - \mathcal{U}\underline{n}$   
Eigenvalue Problem  
 $\frac{\partial L}{\partial \underline{u}} = \underline{n} \cdot \underline{n} - 1 = \underline{O} \Rightarrow ||\underline{n}|| = 1$ 

Eigenvalues and Eigenvectors of *C* are found the same way as any symmetric tensor and have the same physical interpretations.

Note: stretches in the Eigenvector directions:  

$$\lambda^{2}(\underline{n}_{i}) = \underline{n}_{i} \underbrace{\mathcal{C}}_{n} \underline{n}_{i} = u(\underline{n}_{i} \cdot \underline{n}_{i}) = \mathcal{M} \Rightarrow \underbrace{\lambda = \sqrt{u}}_{(\underline{n}_{i} \cdot \underline{n}_{i})}_{(\underline{n}_{i} \cdot \underline{n}_{i})}_{(\underline{n}_{i} \cdot \underline{n}_{i})}_{(\underline{n}_{i} \cdot \underline{n}_{i})} = \underbrace{\Pi}_{2} - \cos^{-1}\left(\frac{\underline{n}_{i} \cdot \underline{\mathcal{C}}_{n_{j}}}{\lambda(\underline{n}_{i}) \times (\underline{n}_{j})}\right) = \underbrace{\Pi}_{2} - \cos^{-1}\left(\frac{u_{j}(\underline{n}_{i} \cdot \underline{n}_{j})}{\sqrt{u}_{i} \sqrt{u}_{j}}\right)_{\sqrt{u}_{i} \sqrt{u}_{j}}$$

$$\Rightarrow \text{ Shear between } \underline{n}_{i} \text{ and } \underline{n}_{j} \quad (\text{for } i \neq j) = 0$$

Similarly principal values of the Lagrangian strain tensor:

En=Yn

$$\Rightarrow \underbrace{1}_{2} (\underline{C} - \underline{I}) \underline{m} = \widehat{\Upsilon} \underline{m} \Rightarrow \underbrace{1}_{2} \underbrace{(\underline{M} - \underline{I})}_{\mathcal{M}} \underline{m} = \widehat{\Upsilon} \underline{m} \Rightarrow \underbrace{1 - 2\widehat{\Upsilon}}_{\mathcal{M}} \underline{m} = \underbrace{(\underline{1 - 2\widehat{\Upsilon}})}_{\mathcal{M}} \underline{m}$$

$$\Rightarrow \underbrace{1 + 2\widehat{\Upsilon}}_{i} = \underbrace{1 + 2\widehat{\Upsilon}}_{i} = \underbrace{1 - 2\widehat{\Upsilon}}_{i} \underline{m} = \underbrace{(\underline{1 - 2\widehat{\Upsilon}})}_{\mathcal{M}} \underline{m}$$

$$\Rightarrow \underbrace{\widehat{\Upsilon}_{i} = \underbrace{1 - 2\widehat{\Upsilon}}_{i} \underbrace{(\underline{M}_{i} - 1)}_{\mathcal{M}}$$

$$\Rightarrow \underbrace{\widehat{\Upsilon}_{i} = \underbrace{1 - 2\widehat{\Upsilon}}_{i} \underbrace{(\underline{M}_{i} - 1)}_{i}$$

$$and same Eigenvectors$$



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Example: (simple shear)



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 $\underline{\mathbb{II}}_{\mathcal{L}} = \left(\lambda_1 \lambda_2 \lambda_3\right)^2$ 

Deformation Gradient (F) and Displacement Gradient (Vu)

Recall:  $p(\underline{z}) = \underline{z} + \underline{u}(\underline{z})$ Deformation gradient:  $F_{\underline{z}} = \nabla_{\underline{z}} p(\underline{z}) = \nabla_{\underline{z}}(\underline{z}) + \nabla_{\underline{z}} \underline{u}(\underline{z}) \Rightarrow F_{\underline{z}} = \overline{1} + \nabla_{\underline{u}}$ Æn. A Du ni £ Vy nz Right Cauchy-Green Deformation Tensor dx = F dz  $dz = dz + \nabla y dz$  $\mathcal{G} = \mathcal{F}^{\mathsf{T}} \mathcal{F} = (\mathbf{I}^{\mathsf{T}} + \nabla \mathbf{u}^{\mathsf{T}}) (\mathbf{I} + \nabla \mathbf{u})$  $\Rightarrow \qquad C = I + \nabla u + (\nabla u + (\nabla u))$  $\underline{\nabla}\underline{\mu} = \frac{\partial u_i}{\partial z_i} \cdot (\underline{e}_i \otimes \underline{e}_j)$ Green Lagrange Strain Tensor  $E = \frac{1}{2} \left( \frac{c}{2} - \frac{c}{2} \right) = \frac{1}{2} \left( \frac{\nabla u}{2} + \frac{\nabla u}{2} + \left( \frac{\nabla u}{2} - \frac{\nabla u}{2} \right) \right)$ strain (E) is a non-linear function of  $\mathcal{U}(\mathbf{E})$ *Linearized* Strain:  $\mathcal{E}_{\mathcal{I}} = \mathcal{I}_{\mathcal{I}} \left( \nabla \mathcal{U}^{\mathsf{T}} + \nabla \mathcal{U}_{\mathcal{I}} \right)$  $\epsilon_{ij} = \frac{1}{2} (u_{i,j} + u_{j,i})$ Example (Ref: Pg 76, Hjelmstad)  $\phi(\mathbf{z}) = (u + z_1 \cos \theta - z_2 \sin \theta) \mathbf{e}_1 + (v + z_1 \sin \theta + z_2 \cos \theta) \mathbf{e}_2 + z_3 \mathbf{e}_3$  $\nabla \mathcal{U} \sim \begin{bmatrix} (\cos \theta - 1) & -\sin \theta & 0 \\ \sin \theta & (\cos \theta - 1) & 0 \end{bmatrix} \Rightarrow \mathcal{F} = \mathcal{I} + \nabla \mathcal{U} \sim \begin{bmatrix} \cos \theta & -\sin \theta & 0 \\ \sin \theta & \cos \theta & 0 \\ \sin \theta & \cos \theta & 0 \end{bmatrix}$  $\Rightarrow C_{r} = F_{r}^{T} F_{r} \sim \begin{vmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \end{vmatrix} \sim I_{r} \Rightarrow E_{r} = O_{r} \quad (Rigid Body Motion)$ However: 

Compatibility of Strains

Meaning of compatibility  
• Given 
$$\underline{\beta}(\underline{z}) = \underline{z} + \underline{u}(\underline{z})$$
  $\longrightarrow \quad \underline{\beta}(\underline{u})$  (Automatically satisfied.)  
• Given  $\underline{\beta}(\underline{z}) = \underline{z} + \underline{u}(\underline{z})$   $\longrightarrow \quad \underline{\beta}(\underline{z})$   $\xrightarrow{2}_{1}$   $\xrightarrow$ 

For linearized (Small strain):

$$\begin{array}{lll} \begin{array}{l} \overline{\partial z_{1}} \\ \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} \\ \begin{array}{l} \overline{\partial z_{1}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} \\ \begin{array}{l} \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} \\ \begin{array}{l} \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{1}} \end{array} & \overline{\partial z_{2}} \end{array} & \overline{\partial z_{1}} \end{array} }$$
 & \overline{\partial z\_{1}} \end{array} & \overline{\partial z

In addition:

(similarly 2 more equations)

=> Total <u>6 equations of compatibility:</u>

$$\nabla \times \mathcal{E} \times \overline{\mathcal{Y}} = \mathcal{O}$$

## Local and Global Changes in Area and Volume



Ratio of local area change:

$$\frac{da}{dA} = \frac{\|\underline{Em_1 \times \underline{Em_2}}\|}{\|\underline{m_1} \times \underline{m_2}\|} = \frac{\|\underline{n} da\|}{\|\underline{m} dA\|} = \frac{det(\underline{F}) \|\underline{Em_1}\|}{det(\underline{F}) \|\underline{Em_1}\|}$$
Original Total Area:  

$$A = \iint_{S} dA \qquad a = \iint_{S} da = \iint_{S} det(\underline{F}) \|\underline{Em_1}\| dA$$

Local change in Volume:

Volume is given by scalar triple product of 3 tangent vectors:

Original Local Volume:

$$dV = (\underline{n}_1 \times \underline{n}_2) \cdot \underline{n}_3 \, ds_1 \, ds_2 \, ds_3$$

Deformed Local Volume:



Original total volume:

$$V = \iiint dV$$

Deformed Total Volume:

$$v = \iiint dv = \iiint det(E) dV$$
  
 $\varphi(v) \qquad v$ 

Example: Look at examples 16 and 17 in the textbook.

78. A circular cylinder of length  $\ell$  and radius R experiences the deformation characterized by the following map:

 $\phi(\mathbf{z}) = \alpha z_1 \mathbf{e}_1 + \beta z_2 \mathbf{e}_2 + \gamma z_3 \mathbf{e}_3$ 

where  $\alpha$ ,  $\beta$ , and  $\gamma$  are constants of the motion. Find the volume of the deformed cylinder. Find the total surface area of the deformed cylinder. Find the principal stretches of the motion. What are the limits on the constants  $\alpha$ ,  $\beta$ , and  $\gamma$ ?



a) Volume of deformed cylinder:  

$$u = \iiint det(\underline{F}) dV$$

$$u = \iiint det(\underline{F}) dV = \alpha \beta^{q} \iiint dV = \alpha \beta^{q} (\Pi R^{2}L)$$
b) Swriface Arrea:  

$$a = \iint da = \iint det(\underline{F}) ||\underline{F}^{T}||dA \qquad \left\{ \begin{array}{c} \underline{m} \sim \begin{bmatrix} \cos \theta \\ \sin \theta \\ 0 \end{bmatrix} \right\}$$
Note 
$$\underline{F}^{T} \sim \begin{bmatrix} V_{\alpha} & 0 & 0 \\ 0 & V_{\beta} \end{bmatrix} \Rightarrow \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) \sim \pm \begin{bmatrix} 0 \\ 0 \\ V \end{bmatrix} & \underbrace{F}^{T} (\frac{\cos \theta}{\alpha}) = \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) \sim \pm \begin{bmatrix} 0 \\ 0 \\ V \end{bmatrix} & \underbrace{F}^{T} (\frac{\cos \theta}{\alpha}) = \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) \sim \pm \begin{bmatrix} 0 \\ 0 \\ V \end{bmatrix} & \underbrace{F}^{T} (\frac{\cos \theta}{\alpha}) = \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) \sim \pm \begin{bmatrix} 0 \\ 0 \\ V \end{bmatrix} & \underbrace{F}^{T} (\frac{\cos \theta}{\alpha}) = \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) \sim \pm \begin{bmatrix} 0 \\ 0 \\ V \end{bmatrix} & \underbrace{F}^{T} (\frac{\cos \theta}{\alpha}) = \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) \sim \pm \begin{bmatrix} 0 \\ 0 \\ V \end{bmatrix} & \underbrace{F}^{T} (\frac{\cos \theta}{\alpha}) = \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) \sim \pm \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) = \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) \sim \pm \underbrace{F}^{T} (\underline{t} \underline{e}_{\beta}) = \underbrace{F}^{T} (\underline{t} \underline{e}) = \underbrace{F}^{T} (\underline{$$

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Time dependent motion

All motion (and deformation) is time-dependent. All the quantities we have defined thus far are for a particular instant of time *t*.

Recall:  

$$u = u_{L}(z,t) = x(z,t) - z = \phi(z,t) - z$$
 (Lagrangian)  $e_{3}$   
 $u = u_{E}(x,t) = x - z(x,t) = x - \phi^{-1}(x,t)$  (Eulerian)  
(REFERENCE/  
MATERIAL)  
 $z = \cancel{(z,t)}$ 

₽2 ↑

Velocity L

(SPATIAL)

Eulerian / Spatial

Lagrangian / Reference / Material

Velocity **P** 

$$\underline{\Psi}_{L}(\underline{z},t) \equiv \underline{\mathcal{X}}(\underline{z},t) = \underbrace{\partial \underline{\mathcal{X}}}_{\partial t}(\underline{z},t) = \underbrace{\partial \underline{\mathcal{X}}}_{\partial t} + \underbrace{\partial \underline{\mathcal{X}}}_{\partial \underline{z}} \underbrace{\partial \underline{z}}_{\partial t}^{E}$$

Acceleration *o* 

$$\underline{Q}_{L}(\underline{z},t) \equiv \underline{\psi}_{L}(\underline{z},t) = \frac{d^{2}\underline{x}}{dt^{2}}(\underline{z},t) = \frac{\partial^{2}\underline{x}}{\partial t^{2}}$$

Example:

e2

Material time derivatives (for Eulerian descriptions)

For any scalar, vector or tensor:  

$$\frac{d \int (\alpha, t)}{dt} = \frac{\partial \int }{\partial t} + \frac{\partial \int }{\partial \alpha} \cdot \frac{\partial \alpha}{\partial t} \qquad \begin{cases} \frac{\partial f}{\partial \alpha} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} \cdot \frac{g}{g} \\ \frac{\partial f}{\partial \alpha_i} = \frac{\partial f}{\partial \alpha_i} - \frac{g}{\partial \alpha_i} - \frac{g}{\partial \alpha_i} -$$

$$\vec{x} = \overset{\neq}{\underline{\beta}}(\underline{z}, t)$$
(spatial)
(spat

Note: 
$$\mathcal{D} = \frac{1}{2} \left( \dot{\mathcal{E}} \mathcal{E}^{-1} + \mathcal{E}^{-T} \dot{\mathcal{E}}^{T} \right)$$
  

$$= \frac{1}{2} \left( (\dot{\mathcal{R}} \mathcal{U} + \mathcal{R} \dot{\mathcal{U}}) (\mathcal{U}^{-1} \mathcal{R}^{-1}) + (\mathcal{R} \mathcal{U}^{-1}) (\mathcal{U} \dot{\mathcal{R}}^{T} + \dot{\mathcal{U}} \mathcal{R}^{T}) \right)$$

$$= \frac{1}{2} \left( \dot{\mathcal{R}} \mathcal{R}^{T} + \mathcal{R} \dot{\mathcal{U}} \mathcal{U}^{-1} \mathcal{R}^{-1} + \mathcal{R} \dot{\mathcal{R}}^{T} + \mathcal{R} \mathcal{U}^{-1} \dot{\mathcal{U}} \mathcal{R}^{T} \right)$$

$$\Rightarrow \mathcal{D} = \frac{1}{2} \mathcal{R} \left( \dot{\mathcal{U}} \mathcal{U}^{-1} + \mathcal{U}^{-1} \dot{\mathcal{U}} \right) \mathcal{R}^{T}$$
and  $\mathcal{W} = \dot{\mathcal{R}} \mathcal{R}^{T} + \frac{1}{2} \mathcal{R} \left( \dot{\mathcal{U}} \mathcal{U}^{-1} - \mathcal{U}^{-1} \dot{\mathcal{U}} \right) \mathcal{R}^{T}$ 

$$\left( \dot{\mathcal{R}} \mathcal{R}^{T} = - \mathcal{R} \mathcal{R}^{T} \right)$$

Note: For any skew symmetric tensor 
$$W$$
:  $\begin{bmatrix} \varrho_{1} & \varrho_{2} & \varrho_{3} \end{bmatrix} \begin{bmatrix} \circ & w_{12} & w_{13} \\ w_{21} & \circ & w_{22} \\ w_{31} & w_{322} & \circ \end{bmatrix} \begin{bmatrix} u_{1} \\ u_{2} \\ u_{3} \end{bmatrix} = det \begin{bmatrix} \varrho_{1} & \varrho_{2} & \varrho_{3} \\ -w_{23} & -w_{31} & -w_{12} \\ u_{1} & u_{2} & u_{3} \end{bmatrix}$   
Article vector of  $W^{-1}$   
Example: Rigid Body Motion:  
 $\psi = \overline{\chi}; \quad \overline{\chi} = \overline{\chi}$   
 $\Rightarrow \quad D_{i} = \Omega; \quad \overline{\chi} = \frac{\partial \psi}{\partial \underline{\chi}} = W = \widehat{g} \, \widehat{g}^{T}$   
 $\Rightarrow \quad d\Psi = \overline{W} \, d\underline{\chi}$   
 $= \omega \times d\underline{\chi}$   
 $(\underline{U}_{B} - \underline{U}_{A}) = \omega \times (\underline{\chi}_{B} - \underline{\chi}_{A})$ 

Example Consider a time-dependent map:



. spin Tensor:  $W = \frac{1}{2} \left( \underline{L} - \underline{L}^T \right) ; \quad W = \underline{W} \times$ 



### **Objectivity** / Frame-indifference

It is important that the physical quantities that we use to characterize material behavior and the laws of physics must not change with a change in the frame of reference *i.e.* they must be <u>objective</u>. While scalar quantities are objective, unfortunately, a lot of vector and tensor quantities (especially those that measure time-rates of changes) are not objective - they are different in different frames of reference.

## Frame of reference:

It is a "point of view / way of viewing" the processes occurring in the world / universe. Think of: Observations (video) from a camera (with full 3D depth perception) and time stamp (so that you can record the distances, orientations and time-instants precisely).

(It is NOT the same as a choice of a coordinate-system.)





$$\Psi(\mathfrak{A},t) = \frac{d}{dt}\mathfrak{A} = \mathfrak{A}$$

$$\Psi^{*}(\mathfrak{A},t) = \frac{d}{dt}\mathfrak{A} = \mathfrak{A}$$

$$\Psi^{*}(\mathfrak{A},t) = \mathfrak{A} = \mathfrak{A} = \mathfrak{A} = \mathfrak{A} = \mathfrak{A} = \mathfrak{A} = \mathfrak{A}$$

$$\Psi^{*}(\mathfrak{A},t) = \mathfrak{A} = \mathfrak$$

Thus velocity and acceleration are NOT objective! Rates of deformation and strain are not objective:  $\dot{\mathcal{L}}, \ddot{\mathcal{L}}$  But  $\mathcal{D} = \frac{1}{2}(\mathcal{L} + \mathcal{L})$  is objective.

Objective Rates: (using Material / Reference frame)

- Co-votational rate : u = u Wu ; I = I WI + IW (Jaumann)
  Convected rate : u = u + L<sup>T</sup>U ; Î = I + L<sup>T</sup>I + IL (cotter-Rivin)
  Oldroyd rate ; Truesdell rate ; Green-Naghdi rate ...

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