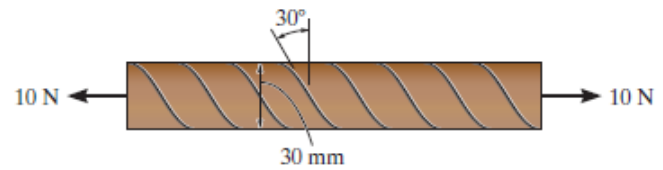


9-38. A paper tube is formed by rolling a paper strip in a spiral and then gluing the edges together as shown. Determine the shear stress acting along the seam, which is at 30° from the vertical, when the tube is subjected to an axial force of 10 N. The paper is 1 mm thick and the tube has an outer diameter of 30 mm.



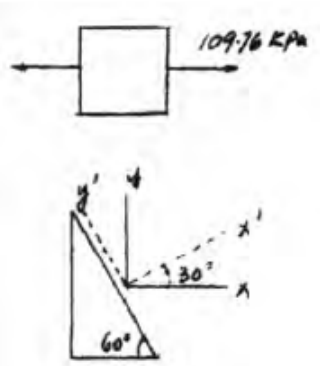
$$\sigma = \frac{P}{A} = \frac{10}{\frac{\pi}{4}(0.03^2 - 0.028^2)} = 109.76 \text{ kPa}$$

$$\sigma_x = 109.76 \text{ kPa} \quad \sigma_y = 0 \quad \tau_{xy} = 0 \quad \theta = 30^\circ$$

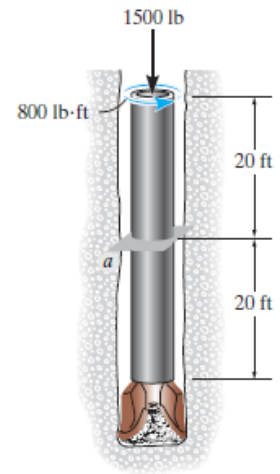
$$\tau_{x'y'} = -\frac{\sigma_x - \sigma_y}{2} \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$= -\frac{109.76 - 0}{2} \sin 60^\circ + 0 = -47.5 \text{ kPa}$$

Ans.



9-42. The drill pipe has an outer diameter of 3 in., a wall thickness of 0.25 in., and a weight of 50 lb/ft. If it is subjected to a torque and axial load as shown, determine (a) the principal stress and (b) the maximum in-plane shear stress at a point on its surface at section *a*.



Internal Forces and Torque: As shown on FBD(a).

Section Properties:

$$A = \frac{\pi}{4} (3^2 - 2.5^2) = 0.6875\pi \text{ in}^2$$

$$J = \frac{\pi}{2} (1.5^4 - 1.25^4) = 4.1172 \text{ in}^4$$

Normal Stress:

$$\sigma = \frac{N}{A} = \frac{-2500}{0.6875\pi} = -1157.5 \text{ psi}$$

Shear Stress: Applying the torsion formula.

$$\tau = \frac{Tc}{J} = \frac{800(12)(1.5)}{4.1172} = 3497.5 \text{ psi}$$

a) **In - Plane Principal Stresses:** $\sigma_x = 0$, $\sigma_y = -1157.5 \text{ psi}$ and $\tau_{xy} = 3497.5 \text{ psi}$ for any point on the shaft's surface. Applying Eq. 9-5.

$$\begin{aligned} \sigma_{1,2} &= \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} \\ &= \frac{0 + (-1157.5)}{2} \pm \sqrt{\left(\frac{0 - (-1157.5)}{2}\right)^2 + (3497.5)^2} \\ &= -578.75 \pm 3545.08 \end{aligned}$$

$$\sigma_1 = 2966 \text{ psi} = 2.97 \text{ ksi}$$

Ans.

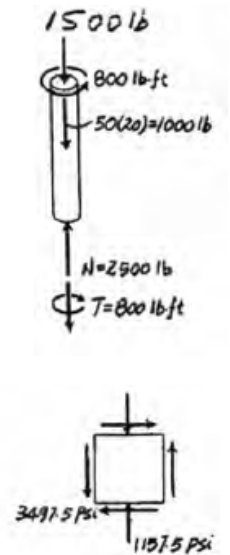
$$\sigma_2 = -4124 \text{ psi} = -4.12 \text{ ksi}$$

Ans.

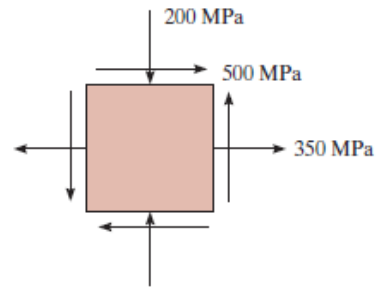
b) **Maximum In - Plane Shear Stress:** Applying Eq. 9-7

$$\begin{aligned} \tau_{\text{in-plane}}^{\text{max}} &= \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} \\ &= \sqrt{\left(\frac{0 - (-1157.5)}{2}\right)^2 + (3497.5)^2} \\ &= 3545 \text{ psi} = 3.55 \text{ ksi} \end{aligned}$$

Ans.



9-67. Determine the principal stress, the maximum in-plane shear stress, and average normal stress. Specify the orientation of the element in each case.



Construction of the Circle: In accordance with the sign convention, $\sigma_x = 350$ MPa, $\sigma_y = -200$ MPa, and $\tau_{xy} = 500$ MPa. Hence,

$$\sigma_{\text{avg}} = \frac{\sigma_x + \sigma_y}{2} = \frac{350 + (-200)}{2} = 75.0 \text{ MPa} \quad \text{Ans.}$$

The coordinates for reference point *A* and *C* are

$$A(350, 500) \quad C(75.0, 0)$$

The radius of the circle is

$$R = \sqrt{(350 - 75.0)^2 + 500^2} = 570.64 \text{ MPa}$$

a)

In - Plane Principal Stresses: The coordinate of points *B* and *D* represent σ_1 and σ_2 respectively.

$$\sigma_1 = 75.0 + 570.64 = 646 \text{ MPa} \quad \text{Ans.}$$

$$\sigma_2 = 75.0 - 570.64 = -496 \text{ MPa} \quad \text{Ans.}$$

Orientation of Principal Plane: From the circle

$$\tan 2\theta_{P1} = \frac{500}{350 - 75.0} = 1.82$$

$$\theta_{P1} = 30.6^\circ \text{ (Counterclockwise)} \quad \text{Ans.}$$

b)

Maximum In - Plane Shear Stress: Represented by the coordinates of point *E* on the circle.

$$\tau_{\text{in-plane}}^{\text{max}} = R = 571 \text{ MPa} \quad \text{Ans.}$$

Orientation of the Plane for Maximum In - Plane Shear Stress: From the circle

$$\tan 2\theta_s = \frac{350 - 75.0}{500} = 0.55$$

$$\theta_s = 14.4^\circ \text{ (Clockwise)} \quad \text{Ans.}$$

