

Viscous Flow Issues for 519 Scope

416 is not a prerequisite.

Some have taken 416, some are taking 416

Don't want to duplicate 416

613 covers in more depth. Don't want to duplicate 613. Some of 416 & 519 can be covered in more depth in 613, some overlap OK.

511 Covers a lot of Viscous Flow, in more depth than in here. 519 is more of a survey course, and covers many more topics. 511 is mostly grads, 519 has more undergrads than grads. Few in 519 have taken 511 (the grads).

How many have taken 416 or 511, or are taking one of these?

21599

Laminar Compressible Viscous Flow

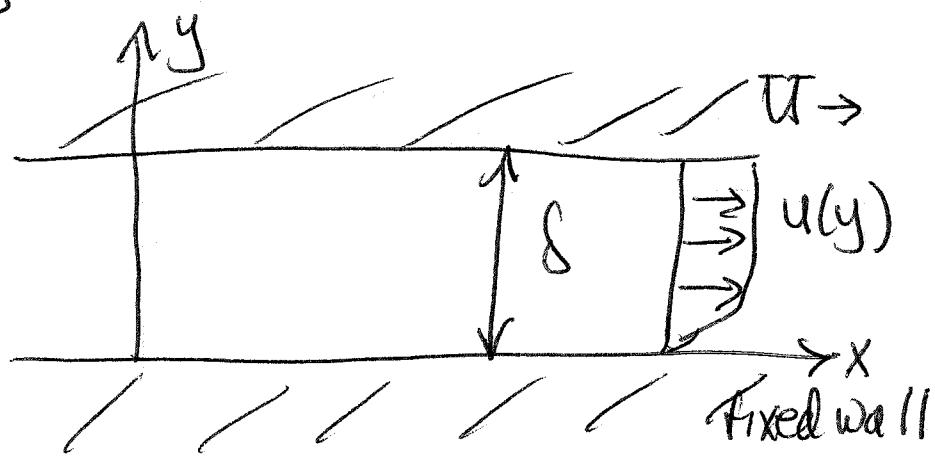
Introduce using

Couette Flow - An Exact Solution

Lepmann & Roshki
p 306-313

21599
Fall 1999
No 416/613
people

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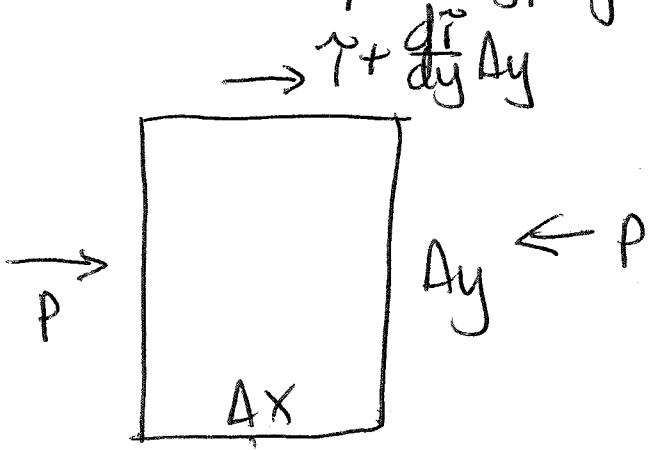


$$\frac{\partial y}{\partial t} + u \frac{\partial y}{\partial x} + v \frac{\partial y}{\partial y}$$

$$= -\frac{1}{\rho} \frac{\partial p}{\partial x} + \frac{1}{\rho} \frac{\partial \tau}{\partial y} + \frac{f_x}{\rho}$$

3-23 $T = \mu \frac{\partial u}{\partial y}$

Assume 2-D, steady, neglect gravity, flow independent of x.



Equilibrium in x-direction

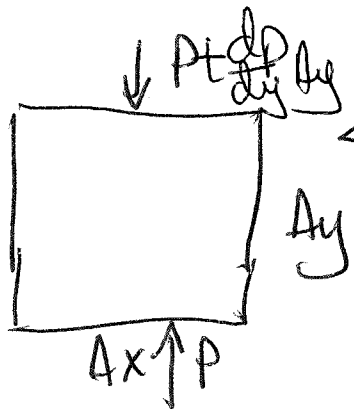
shows $\frac{d\tau}{dy} = 0$

(a bit high-handed)

$$\frac{\partial \rho u}{\partial x} + \frac{\partial \rho v}{\partial y} = 0$$

$$\Rightarrow v = 0 \text{ so no shear on vertical sides}$$

$\tau = 0$
since $v = 0$



\rightarrow since $v = 0$, $\frac{\partial p}{\partial y} = 0$ also

$\Rightarrow T = \mu \frac{\partial u}{\partial y} = \text{const} = \tau_w$ (a bit high-handed too)

~~Schettz 2, 3, 4, 5~~

$$\rho \left[\frac{\partial}{\partial t} \left(h + \frac{u^2}{2} \right) + u \frac{\partial}{\partial x} \left(h + \frac{u^2}{2} \right) + v \frac{\partial}{\partial y} \left(h + \frac{u^2}{2} \right) \right] = - \frac{\partial q_y}{\partial y} + \frac{\partial}{\partial y} (\tau u) + \frac{\partial p}{\partial t}$$

But $u \neq \frac{\tau}{\mu} y + \text{const}$, for $y \neq \text{const}$!!

μ depends on T . (molecular argument).
 (also true for heat transfer if $p = \text{const}$)

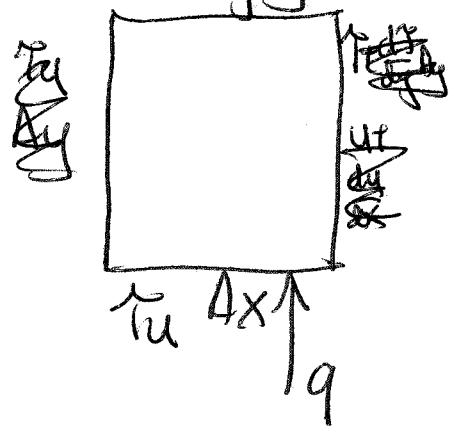
temperature varies because viscous work \rightarrow heat.

$$\frac{\mu}{\tau} du = dy \quad \text{so} \quad y = \int \frac{\mu}{\tau} du$$

Energy Equation needed
 (steady)

$$\left[q - \left(q + \frac{dq}{dy} \Delta y \right) \right] \Delta x +$$

- work done by shear



$$\left[\left(\tau + \frac{d\tau}{dy} \Delta y \right) \left(u + \frac{du}{dy} \Delta y \right) - \tau u \right] \Delta x = 0$$

$$- \frac{dq}{dy} \Delta y \Delta x + \tau \frac{du}{dy} \Delta y \Delta x + u \frac{d\tau}{dy} \Delta y \Delta x + \frac{d\tau}{dy} \frac{du}{dy} (\Delta y)^2 \Delta x = 0$$

divide by $\Delta y \Delta x$, take limit, last term drops out.

$$- \frac{dq}{dy} + \frac{d}{dy} (\tau u) = 0 \quad \text{or} \quad \tau u - q = \text{const}$$

2-496
 2-20-98 but need to do w/ free body, only did from Schettz.

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Evaluate at wall: (lower wall, $u=0$, $q=q_w$)

$$\tau u - q = \frac{q_w}{\lambda} \quad \text{heat transfer at fixed wall.}$$

-2-10-95

Note 1: $\tau u - q = \mu u \frac{\partial u}{\partial y} + k \frac{\partial T}{\partial y}$

$$= \mu \left(\frac{\partial}{\partial y} \left(\frac{u^2}{2} \right) + \frac{k}{\mu c_p} \frac{\partial h}{\partial y} \right)$$

$$Pr \equiv \frac{c_p \mu}{k} = \text{Prandtl number.}$$

ratio of diffusion of momentum to diffusion of heat.

for many substances, Pr is a const, since μ & k vary in the same way.

if we take $Pr = 1$, then $\mu \neq \text{const.}$
 $Tu + q = \mu \frac{\partial}{\partial y} (h + \frac{1}{2}u^2) \neq h + \frac{1}{2}u^2 = H = \text{const.}$
 a great simplification.

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Note 2
 $q - u \tau = \text{const}$ is the balance between heat transfer and viscous dissipation which is required in order to have steady 2-D flow.

Here Pr

using note 1,

$$-q_w = \mu \left[\frac{\partial}{\partial y} \left(\frac{u^2}{2} \right) + \frac{1}{Pr} \frac{\partial h}{\partial y} \right]$$

$$= \mu \frac{\partial}{\partial y} \left(\frac{u^2}{2} + \frac{h}{Pr} \right)$$

Assume constant Prandtl #

"near-const. even for dissociation since C_p & k peak at same place"

Integrate:

$$\frac{u^2}{2} + \frac{h}{Pr} = \int \frac{q_w}{\mu} dy + \text{const.}$$

$$\frac{u^2}{2} + \frac{h}{Pr} = - \int_0^y \frac{q_w}{\mu} dy + \frac{h_{wall}}{Pr} = -q_w \int_0^y \frac{dy}{\mu} + \frac{h_{wall}}{Pr}$$

$$= -q_w \int_0^y \frac{du}{\tau} + \frac{h_{wall}}{Pr} \quad (\text{using mom. } \tau = \mu \frac{du}{dy})$$

$$\frac{u^2}{2} + \frac{h}{Pr} = -\frac{q_w}{T_w} \int_0^y du + \frac{h_{wall}}{Pr}$$

$$= -\frac{q_w}{T_w} u + \frac{h_{wall}}{Pr}$$

So $\boxed{\frac{u^2}{2} + \frac{h-h_{wall}}{Pr} = -\frac{q_w}{T_w} u}$

Energy eqn. for
Couette flow
(const Pr.)

at upper wall, $y = \delta$,

$$\boxed{\frac{U^2}{2} + \frac{h_s - h_w}{Pr} = -\frac{q_w}{T_w} U}$$

$q_w =$ lower wall
heat transfer

Bring in a new concept: Recovery temperature or enthalpy:

Recovery temperature is the wall temperature,
if the wall is insulated, or adiabatic.

$$h_w (q_w = 0) = h_r \text{ at lower wall}$$

$$\text{So } \boxed{h_{r,w} = \frac{U^2}{2} Pr + h_s} = h_s \left(1 + Pr \frac{U^2}{2h_s} \right)$$

So recovery enthalpy at the lower wall is

$$\frac{h_r}{h_g} = 1 + \frac{Pr U^2}{2 h_g}$$

$$\frac{U^2}{h_g} = \frac{U^2}{c_p T_g \gamma R \frac{1}{\gamma R}} = \frac{U^2}{a_g^2} \frac{1}{\frac{c_p}{\gamma R} \frac{1}{c_p (\gamma - 1)}} = (\gamma - 1) M_g^2$$

so $\boxed{\frac{h_{r,w}}{h_g} = 1 + \frac{\gamma - 1}{2} Pr M_g^2}$

$h_r \sim M_g^2$
Look at numbers
for transonic, supersonic,
hypersonic.

not just U_g^2 or M_g^2 but
has Pr # factor

Note: since $\frac{h_g}{Pr} + \frac{U^2}{2} = \frac{h_r}{Pr}$, and $\frac{h_g - h_w}{Pr} = -\frac{q_w}{\tilde{\rho}_w U} - \frac{U^2}{2}$,

$$\frac{h_r}{Pr} - \frac{U^2}{2} = \frac{h_w}{Pr} - \frac{q_w}{\tilde{\rho}_w U} - \frac{U^2}{2}$$

$$\frac{q_w}{\tilde{\rho}_w U} = \frac{h_w - h_r}{Pr}, \text{ or } \boxed{q_w = \frac{\tilde{\rho}_w}{U Pr} (h_w - h_r)}$$

For positive heat transfer, $h_w > h_r$.

Wall has a higher natural temperature than the ambient stream. To make heat flow from the wall, wall temp. must be \gg ambient at high speed. Very difficult to cool a wall in high-speed flow - rather, the flow heats the wall.

The stagnation enthalpy $H = h + \frac{1}{2} u^2$. This differs from the recovery enthalpy by a Pr # factor. Why? - heat transfer.

$$[H_s \text{ could be } h_s + \frac{1}{2} U^2]$$

2-16-96.
10-4-99

Note that

$$\frac{h_r - h_g}{C_p} = T_r - T_g = \frac{U^2}{2} Pr \frac{1}{C_p}$$

Note also that $\frac{T_0}{T_g} = 1 + \frac{\gamma-1}{2} M_g^2$ from isentropic relations.

$$\frac{T_r - T_g}{T_0 - T_g} = r = \text{recovery factor} = \frac{\frac{U^2}{2 C_p} Pr}{T_g \frac{\gamma-1}{2} M_g^2}$$

$$\frac{U^2}{2 h_g} = \frac{U^2}{2 C_p T_g} = \frac{U^2}{2 C_p \frac{1}{\gamma R} (\gamma R T_g)} = M_g^2 \frac{1}{2} \frac{C_p \gamma}{C_p \gamma} = M_g^2 \frac{\gamma-1}{2 \gamma}$$

$$\text{so } r = \frac{M_g^2 \frac{\gamma-1}{2} Pr}{\frac{\gamma-1}{2} M_g^2} = \boxed{Pr = r}$$

For flat plate
lam B.L. Text
 $r \approx \sqrt{Pr}$ eqn
(6.90)
p. 219

if $Pr = 1$, stagnation enthalpy temp = recovery temp.
Different because $Pr \neq 1$, momentum &
heat don't diffuse at same rate.

(p. 310 L&R).

So what is the velocity profile?

$$\tau = \tau_w = \mu \frac{du}{dy}$$

$$dy = \frac{\mu}{\tau_w} du \Rightarrow y = \frac{1}{\tau_w} \int_0^u \mu du \quad (\text{velocity profile})$$

What is $\mu(T)$ or $\mu(h)$? (see white paper)

can get a good approx. for dilute gases using a power-law:

$$\frac{\mu}{\mu_s} = \left(\frac{T}{T_s}\right)^w \quad \frac{1}{2} \leq w < 1$$

so
$$y = \frac{\mu_s T}{\tau_w} \int_0^u \frac{\mu}{\mu_s} d\left(\frac{u}{T}\right)$$

evaluate at upper wall:

$$\delta = \frac{\mu_s}{\tau_w} T \int_0^1 \frac{\mu}{\mu_s} d\left(\frac{u}{T}\right)$$

$$\Rightarrow \tau_w = \frac{\mu_s}{\delta} T \int_0^1 \frac{\mu}{\mu_s} d\left(\frac{u}{T}\right)$$

(u depends on T, h, μ_s , etc, known; $\mu/\mu_s = f(\mu/\mu_s)$ too. So this can be found, maybe numerically)

Now remember

$$h = h_w - \frac{q_w U}{\gamma_w} Pr - \frac{Pr}{2} u^2$$

and

$$\frac{U^2}{2} + \frac{h_s - h_w}{Pr} = -\frac{q_w U}{\gamma_w}$$

$$\text{So } h = h_w + Pr \left(\frac{U^2}{2} + \frac{h_s - h_w}{Pr} \right) \frac{u}{U} - \frac{Pr}{2} u^2$$

$$h = h_w - \frac{Pr}{2} u^2 + \frac{Pr}{2} u U + (h_s - h_w) \frac{u}{U}$$

$$\text{or } h = h_w \left(1 - \frac{u}{U} \right) + \frac{Pr}{2} u (U - u) + \frac{u}{U} h_s$$

at $u=0$, $h=h_w$ OK; at $u=U$, $h=h_s$ OK

$$\frac{h}{h_s} = \frac{h_w}{h_s} \left(1 - \frac{u}{U} \right) + \frac{Pr u}{2 h_s} (U - u) + \frac{u}{U}$$

2-25-98 but reduced to m/s form
-2-19-97

use this and power law for μ/μ_s in integral eqn also

for γ_w to get $\tilde{\tau}_w$: ($\mu/\mu_s = f(h/h_s)$) ($\xi \equiv \frac{u}{U}$)

$$\tilde{\tau}_w = \frac{\mu_s}{s} U \int_0^1 f \left[\frac{h_w}{h_s} (1-\xi) + \frac{Pr U^2}{2 h_s} (\xi)(1-\xi) + \xi \right] d\xi$$

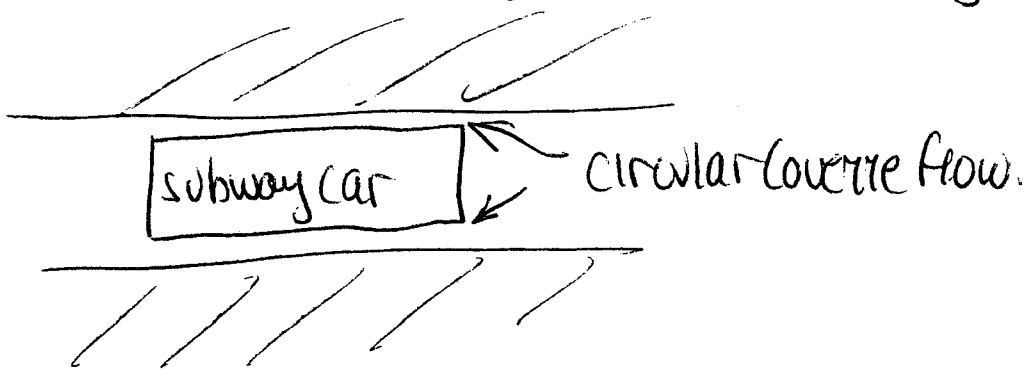
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u-s

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1st term is heat transfer from wall, rest is mach # effects.

M is measure of energy in velocity vs. energy in enthalpy.



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for a perfect gas, $h = c_p T = \frac{c_p}{\gamma R} \gamma R T = \frac{c_p}{\gamma (c_p - c_v)} a^2 = \frac{a^2}{\gamma - 1}$

eqn for M/h_s becomes

$$h_s = \frac{a_s^2}{\gamma - 1}$$

$$\frac{T}{T_s} = \frac{T_w}{T_s} \left(1 - \frac{u}{a}\right) + \frac{Pr}{2} \frac{u^2}{(\gamma - 1) a_s^2} \frac{u}{a} \left(1 - \frac{u}{a}\right) + \frac{u}{a}$$

$$= \frac{T_w}{T_s} \left(1 - \frac{u}{a}\right) + \frac{Pr}{2} (\gamma - 1) M^2 \left(\frac{u}{a}\right) \left(1 - \frac{u}{a}\right) + \frac{u}{a}$$

but integral of power-law still hard (not analytic)

"In most cases where M large, have gases, and $Pr \approx 1$ then $\eta(\tau)$ is the most important tricky part"

Mul

for a perfect gas, then

$$\tau_w = \frac{\mu_s}{\delta} U \int_0^1 f \left[\frac{T_w}{T_s} (1-\xi) + \frac{Pr}{2} (\gamma-1) M^2 \xi(1-\xi) + \xi \right] d\xi$$

terms in the

if $f(x) = x$, or linear, then can decouple the integrals

where ~~μ~~ , $\frac{\mu}{\mu_0} \approx \left(\frac{T}{T_0}\right)^n$

~~$$\frac{\mu}{\mu_s} = \frac{\mu_0}{\mu_0} \frac{\mu_0}{\mu_s} = \frac{\mu_0}{\mu_s} \left(\frac{T}{T_0}\right)^n = \frac{\mu_0}{\mu_s} \left(\frac{T_s}{T_0}\right)^n \left(\frac{T}{T_s}\right)^n$$~~

~~so $f(x) = \frac{\mu_0}{\mu_s} \left(\frac{T_s}{T_0}\right)^n x^n$~~

$n \approx 0.7$, $T_0 = 492^\circ R$
 $\mu_0 = .1716 \text{ mP}$

~~$f(x) = C_1 x^n$~~

~~$f(x) = x^n$~~

~~$C_1 = \left(\frac{T_0}{T_s}\right)^n \left(\frac{T_s}{T_0}\right)^n = 1$~~

so

$$\tau_w = \frac{\mu_s}{\delta} U \int_0^1 \left[\frac{T_w}{T_s} (1-\xi) + \frac{Pr}{2} (\gamma-1) M^2 \xi(1-\xi) + \xi \right]^n d\xi$$

$$[\tau_w] = \left[\mu \frac{du}{dy} \right] = \left[\mu \frac{U}{y} \right]$$

$$\bar{\tau}_w = \frac{\tau_w}{\mu_s U} = \int_0^1 \left[\frac{T_w}{T_s} (1-\xi) + \frac{Pr}{2} (\gamma-1) M^2 \xi(1-\xi) + \xi \right]^n d\xi$$

2-15-95

Example 1: Does the Prandtl # affect result much? (often "no")

What are T_w & T_s ? say, heat or cool to keep $T_w = T_s$.

let $\delta = 1.4, M = 3, Pr = ?, n = 0.7$

if $Pr = 0.7, \bar{T}_w = 1.142$ } not big
= 1.0, $\bar{T}_w = 1.20$ } effect

Obviously, as $M \rightarrow \infty, \bar{T}_w \sim (Pr)^n$ with Pr , nearly linear.

for $M \rightarrow 0, Pr$ doesn't matter at all.

at $M = 10, \delta = 1.4, n = 0.7$ { $Pr = 0.7, \bar{T}_w = 2.29; Pr = 1, 2.75$ }

$\frac{T_w}{T_s}$ - reg 1
~~reg 2~~
~~reg 3~~
n - reg 4
 ξ - reg 0
~~reg 1~~
~~reg 2~~

$\frac{Pr}{2} (M-1)^2$ - reg 2

ok prgm:
all i's
(reg 1, 2, 4)
7/6

Ex 2: Does the power law exponent affect the result much?

$$\bar{T}_w = \int_0^1 (f(\xi))^n d\xi = \int_0^1 e^{n \ln f(\xi)} d\xi$$

$$\bar{T}_w(n=n_0) = \bar{T}_w(n_0) + \frac{d\bar{T}_w}{dn_0} (n-n_0) + \dots$$

$$\frac{\partial \bar{T}_w}{\partial n} = \int_0^1 f(\xi) e^{n \ln f(\xi)} d\xi \quad \text{hard.}$$

if $\delta = 1.4, M = 3, Pr = 0.7, n = 0.7, \bar{T}_w = 1.14$ } not a big
 $n = 1.0, \bar{T}_w = 1.21$ } effect either.

From Anderson, Fundamentals of Aerodynamics, McGraw-Hill, 2nd ed.

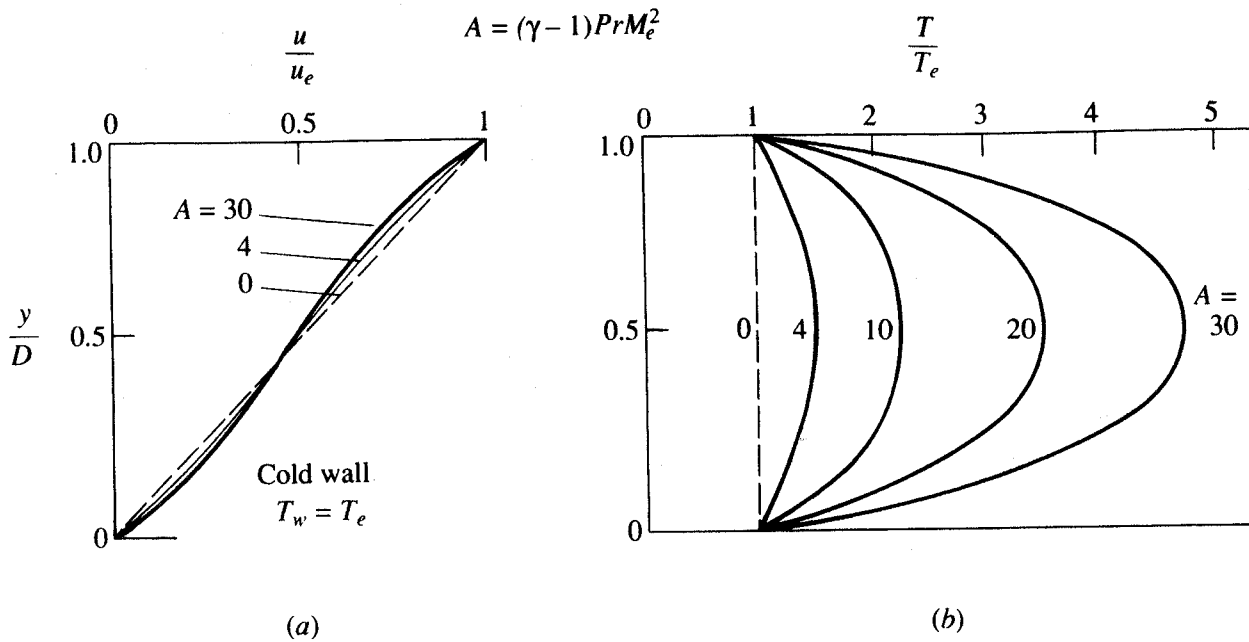


FIGURE 16.9

Velocity and temperature profiles for compressible Couette flow. Cold wall cases. (From White, Ref. 43.)

For $\gamma=1/4$, $Pr=0.7$, this is $M=0, 3.8, 6.0, 8.5$ & 10.4

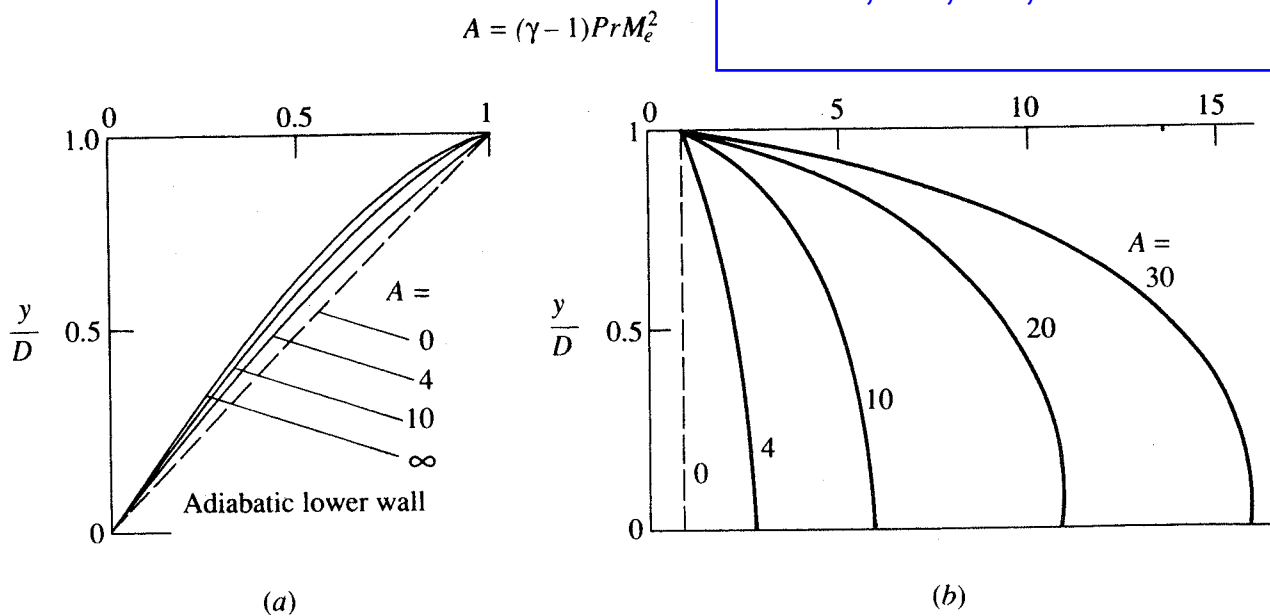


FIGURE 16.10

Velocity and temperature profiles for compressible Couette flow. Adiabatic lower wall. (From Ref. 43.)

Power ~~to the~~ law for μ .

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2597

1-30-95

82B

The ~~the~~ equations can be solved for Blasius flow,
flat plate, $\frac{\partial T}{\partial x} = 0$.

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = \nu \frac{\partial^2 u}{\partial y^2} \quad ; \quad \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

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Dim. analysis: u, v, x, y, ν . 6 var, 2 dimensions, 4 π 's

$\frac{u}{U}, \frac{v}{U}$, 2 more. $\frac{x}{y}$? no change w/ Re. $\left[\frac{\nu}{U} \right] = m$

Try a similarity solution.

Let $\eta = \frac{y}{\sqrt{\frac{x\nu}{U}}}$. $\eta = \frac{y}{x}$ clearly won't work

Like $\frac{y}{\sqrt{xt}}$, $x' = \frac{x}{U}$

Let $u = U f(\eta)$ $U = \text{const}$ for Blasius, ^{but} not F-S

$$\frac{\partial u}{\partial x} = f'(\eta) U y \sqrt{\frac{U}{\nu}} \left(-\frac{1}{2}\right) x^{-3/2} = U f' \left(-\frac{1}{2}\right) \frac{y}{\sqrt{\frac{x\nu}{U}}} \frac{1}{x}$$

$$\frac{\partial u}{\partial x} = -\frac{1}{2} \eta f' U \frac{1}{x}$$

29-98

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$$\frac{\partial y}{\partial y} = U f' \frac{1}{\sqrt{\frac{v x}{U}}}$$

Note $\frac{\partial y}{\partial y} \sim \frac{1}{\sqrt{v}} \rightarrow \infty$ as $v \rightarrow 0$.
 $\frac{\partial y}{\partial x}$ stays finite, smaller

$$\frac{\partial^2 y}{\partial y^2} = U f'' \frac{1}{\frac{v x}{U}} = \frac{U^2}{v x} f''$$

← 'partial integ' - note - a long const x

$$v = - \int \frac{\partial y}{\partial x} dy \text{ from continuity.}$$

$$= - \int (-\frac{1}{2}) \eta f' U \frac{1}{x} dy \quad d\eta = \frac{dy}{\sqrt{\frac{v x}{U}}}$$

$$= \frac{1}{2} U \frac{1}{x} \sqrt{\frac{v x}{U}} \int \eta f' d\eta \quad [+ f(x) \text{ only}]$$

$$= \frac{1}{2} \sqrt{\frac{U v}{x}} \int \eta f' d\eta$$

integrate by parts $\int u dv = uv - \int v du$

$$\begin{aligned} dv &= f' d\eta & u &= \eta \\ v &= f & du &= d\eta \end{aligned}$$

$$v = \frac{1}{2} \sqrt{\frac{U v}{x}} \left[\eta f - \int f d\eta \right]$$

Note that $\frac{v}{U} \sim \sqrt{\frac{y}{U x}} \sim \frac{1}{Re_x^{1/2}}$ as $Re_x \rightarrow \infty$,
 $v \rightarrow 0$. OK.

Assemble:

$$U f' \left(-\frac{1}{2} \right) \eta f' \frac{U}{X} + \frac{1}{2} \sqrt{\frac{U^2}{X}} \left[\eta f - \int f d\eta \right] U f' \frac{1}{\sqrt{\frac{U^2}{X}}}$$

$$= \frac{U^2}{X^2} f''$$

Simplify.

$$-\frac{1}{2} \eta f' \frac{U^2}{X} + \frac{1}{2} \sqrt{\frac{U^2}{X^2}} U f' \left[\eta f - \int f d\eta \right] = \frac{U^2}{X} f''$$

here
2+95

divide out U^2/X .

$$-\frac{1}{2} \cancel{\eta f'} + \frac{1}{2} \cancel{f' \eta f} - \frac{1}{2} \left[f' \int f d\eta \right] = f''$$

$$f'' + \frac{1}{2} f' \int f d\eta = 0$$

$$\text{let } g = \int f d\eta. \quad g' = f, \quad g'' = f', \quad g''' = f''$$

$$\boxed{g''' + \frac{1}{2} g'' g = 0}$$

Blasius Equation.

3 BC needed.

B.C. For flat plate, $u=v=0$ at $\eta=0$ ($=y=0$)
(except where $x=0$)

$$u(y=0) = 0$$

$$\Rightarrow f(0) = 0 \Rightarrow \boxed{g'(0) = 0}$$

$$v = \frac{1}{2} \sqrt{\frac{Uv'}{x}} (\eta g' - g)$$

$$v(0) = 0 \Rightarrow \boxed{g(0) = 0}$$

as $y \rightarrow \infty$, $\eta \rightarrow \infty$, $u \rightarrow U \Rightarrow f(\infty) = 1$ or
 $\boxed{g'(\infty) = 1}$

(See solution plot)

Note: $v(\infty) \neq 0$.

as $\eta \rightarrow \infty$, $g' \rightarrow 1$, $\eta \rightarrow \infty$, $g \rightarrow \infty$.

$$\eta g' - g \rightarrow 1.7208$$

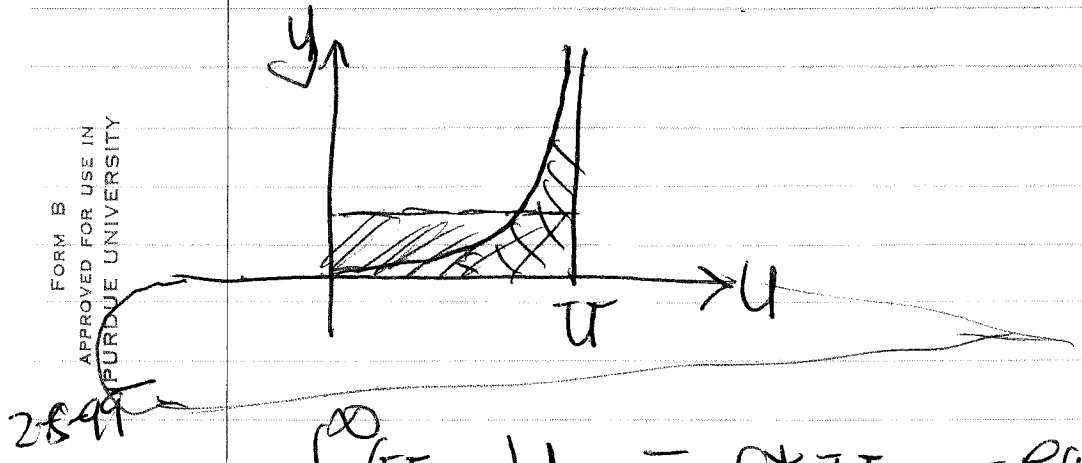
$$v(y \rightarrow \infty) = \frac{1}{2} \sqrt{\frac{Uv'}{x}} (1.7208)$$

$$\frac{v}{U} = \frac{1.7208}{2} \sqrt{\frac{v}{Ux}} = \frac{1.7208}{2} \frac{1}{\text{Re}_x^{1/2}}$$

finite movement out.

1-7-97
show solns.
2-11-98 showed solns
already.

This can be accounted for by the displacement thickness:



$$\int_0^{\infty} (U-u) dy \equiv \delta^* U \quad \text{— equivalent displacement of outer flow.}$$

$$\delta^* = \int_0^{\infty} \left(1 - \frac{u}{U}\right) dy = \sqrt{\frac{x\nu}{U}} \int_0^{\infty} (1-g') d\eta$$

$$\frac{\delta^*}{x} = \frac{1}{Re_x^{1/2}} \int_0^{\infty} (1-g') d\eta = \frac{1}{\sqrt{Re_x}} \left[\eta - g \right]_0^{\infty}$$

since $g' \rightarrow 1$ as $\eta \rightarrow \infty$, this is

$$\frac{\delta^*}{x} = \frac{1}{\sqrt{Re_x}} 1.7208$$

Leads to higher order correction to potential flow — look at inviscid flow over body + displacement thickness.

2-5-96

SPS
30 Jan 01
①

Heat Transfer in Falkner-Skan Flows

Incompressible 2-D B.L.

Need the energy equation. Here, assume low speed,

so $\frac{1}{2}u^2 \ll h$, and neglect ^{viscous} dissipation into heat.

Anderson HHTG (6.30) gives (6.30 in 2e and 3e also)

$$\rho u \frac{\partial h}{\partial x} + \rho v \frac{\partial h}{\partial y} = \frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y} \right) + u \frac{\partial p_e}{\partial x} + \mu \left(\frac{\partial u}{\partial y} \right)^2$$

derivation of this equation is tricky, and given in 6.13.

here, note that $\mu \left(\frac{\partial u}{\partial y} \right)^2$ is the viscous dissipation into heat (neglect here), and that

$$u \frac{\partial p_e}{\partial x} = u \frac{\partial}{\partial x} \left(-\frac{1}{2} \rho U^2 \right) = -u U \frac{\partial U}{\partial x} \rho$$

$$= -\rho u \frac{\partial}{\partial x} \left(\frac{1}{2} U^2 \right) \ll \rho u \frac{\partial h}{\partial x} \quad (\text{in magnitude})$$

for low speed, so neglect it. (see HW, Aelolb Final, 1984)

we now have, assuming $h = c_p T$, $c_p = \text{const}$,

(2)

$$\rho c_p u \frac{\partial T}{\partial x} + \rho c_p v \frac{\partial T}{\partial y} = \frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y} \right) = k \frac{\partial^2 T}{\partial y^2}$$

which is consistent with White (4-35c)

From the momentum-equation solution, we have u & v .

Here, we assume that the changes in T do not affect

u & v - a poor assumption in many cases, since $\mu = \mu(T)$. Here we assume $\mu = \text{const}$, etc.

We have $u = U f'(\eta)$, $\eta = y \sqrt{\frac{m+1}{2} \frac{U}{\nu x}}$, and

$$v = -\sqrt{\frac{2\nu k}{m+1}} \left[\frac{m+1}{2} x^{\frac{m-1}{2}} f + x^{\frac{m+1}{2}} \frac{\partial \eta}{\partial x} f' \right]$$

following White, let $\theta = \frac{T - T_e}{T_w - T_e}$, so $\frac{\partial \theta}{\partial x} = \frac{1}{T_w - T_e} \frac{\partial T}{\partial x}$

ρ & c_p are constant. So

$$\rho c_p U f' \frac{\partial \theta}{\partial x}$$

$$+ \rho c_p \frac{\partial \theta}{\partial y} \left[(-) \sqrt{\frac{2\nu k}{m+1}} \left[\frac{m+1}{2} x^{\frac{m-1}{2}} f + x^{\frac{m+1}{2}} \frac{\partial \eta}{\partial x} f' \right] \right]$$

$$= (T_w - T_e) \frac{\partial^2 \theta}{\partial y^2} k$$

now $\frac{\partial \theta}{\partial x} = \frac{\partial}{\partial x} (\theta(\eta(x,y))) = \frac{d\theta}{d\eta} \frac{\partial \eta}{\partial x}$, etc,

so we have

$$\rho_f U f' \theta' \frac{\partial \eta}{\partial x}$$

$$- \rho_f \theta' \frac{\partial \eta}{\partial y} \left[\sqrt{\frac{2\nu k'}{m+1}} \right] \left[\frac{m+1}{2} x^{\frac{m+1}{2}} f + x^{\frac{m+1}{2}} \frac{\partial \eta}{\partial x} f' \right]$$

$$= \frac{d^2 \theta}{d\eta^2} \left(\frac{\partial \eta}{\partial y} \right)^2 k = \theta'' \left(\frac{m+1}{2} \frac{U}{\nu x} \right) k$$

3-501 -

$$U f' \theta' \frac{\partial \eta}{\partial x} - \theta' \left[\frac{m+1}{2} \frac{U}{\nu x} \sqrt{\frac{2\nu k'}{m+1}} \right] \frac{m+1}{2} x^{\frac{m+1}{2}} f$$

$$- \theta' \left[\frac{m+1}{2} \frac{U}{\nu x} \sqrt{\frac{2\nu k'}{m+1}} \right] x^{\frac{m+1}{2}} \frac{\partial \eta}{\partial x} f' = \theta'' \frac{m+1}{2} \frac{U}{\nu x} \frac{k}{\rho_f}$$

$$U f' \theta' \frac{\partial \eta}{\partial x} - \theta' f \sqrt{\frac{U}{x}} \sqrt{k'} x^{m/2} \frac{m+1}{2} \frac{1}{\sqrt{x}} f$$

$$- \theta' \sqrt{\frac{U}{x}} \sqrt{k'} x^{m/2} \frac{\partial \eta}{\partial x} f' = \theta'' \frac{m+1}{2} \frac{U}{\nu x} \frac{k}{\rho_f}$$

(4)

$$\frac{U f' \theta' \frac{\partial \eta}{\partial x} - f \theta' \sqrt{\frac{U}{x}} \sqrt{U} \frac{1}{\sqrt{x}} \frac{m+1}{2} f}{- f' \theta' \sqrt{U} \sqrt{U} \frac{\partial \eta}{\partial x}} = \frac{k}{\rho c_p} \frac{m+1}{2} \frac{U}{\nu x} \theta''$$

$$0 = \frac{k}{\rho c_p} \frac{m+1}{2} \frac{U}{\nu x} \theta'' + \theta' f \frac{U}{x} \frac{m+1}{2}$$

$$0 = \frac{k}{\rho c_p} \frac{\rho}{\mu} \theta'' + f' \theta'$$

$$0 = \theta'' + \frac{\mu c_p}{k} f' \theta'; \quad Pr \equiv \frac{\mu c_p}{k}$$

$$\boxed{0 = \theta'' + Pr f' \theta'} \quad \text{agrees with White (4-76)}$$

Note that $u = U(x) f'(\eta) \equiv \frac{\partial \psi}{\partial y}$, $\psi = \text{stream fn.}$
(e.g. White 4-44)

$$\begin{aligned} \text{so } \psi &= \int U(x) f'(\eta) dy \Big|_{x \text{ held const.}} \\ &= U(x) \frac{1}{\sqrt{\frac{m+1}{2} \frac{U}{\nu x}}} \int f'(\eta) d\eta \end{aligned}$$

(5)

$$\text{or } \psi = \sqrt{\frac{2}{m+1}} v x U(x) f \quad (\text{agrees with White 4-77})$$

Now f is assumed constant, known from F-S soln.

so

$$\frac{d^2\theta}{dn^2} + Pr f(n) \frac{d\theta}{dn} = 0 \quad ; \quad \text{let } \alpha = \frac{d\theta}{dn}$$

$$\frac{d\alpha}{dn} = -Pr f \alpha \Rightarrow \frac{d\alpha}{\alpha} = -Pr f dn$$

$$\ln \alpha = -Pr \int f dn + \text{const}$$

$$\alpha = \text{const } e^{-Pr \int f dn} = \frac{d\theta}{dn}$$

$$\theta = \text{const} \int e^{-Pr \int f dn} dn + \text{const.}$$

at $n=0$, $\theta=1$, since $T=T_w$

$n=\infty$, $\theta=0$, since $T=T_e$

(6)

let

$$\Theta = \text{CONST} \int e^{-Pr \int_0^\eta f d\eta} d\eta + \text{CONST}$$

by inspection, write (4-58) first bc,

$$\Theta = \frac{\int_0^\infty \exp(-Pr \int_0^\eta f d\eta) d\eta}{\int_0^\infty \exp(-Pr \int_0^\eta f d\eta) d\eta}$$

$$q_w = -k \frac{\partial T}{\partial y} \Big|_{y=0} = -k (T_w - T_e) \frac{d\theta}{d\eta} \frac{d\eta}{dy} \Big|_{y=0}$$

$$q_w = -k (T_w - T_e) \sqrt{\frac{m+1}{2} \frac{U}{\nu x}} \frac{d\theta}{d\eta} \Big|_{\eta=0} \sim \frac{1}{\sqrt{x}}$$

$$No_x \equiv \frac{q_w x}{k_x (T_w - T_e)} = -\theta'(0) \sqrt{\frac{m+1}{2}} \sqrt{\frac{U x}{\nu}}$$

$$No_x = -\theta'(0) \sqrt{\frac{m+1}{2}} Re_x^{1/2}$$

$$\frac{d\theta}{d\eta} \Big|_{\eta=0} = \theta'(0) = \frac{1}{\int_0^\infty \omega d\eta} \frac{d}{d\eta} \left[\int_0^\infty \exp(-Pr \int_0^\eta f(s) ds) d\eta \right]$$

$$\left. \frac{d\theta}{dn} \right|_{n=0} = \frac{1}{\int_0^{\infty} e^{-Pr \int_0^n f(s) ds} dn} \circ$$

$$\left[(-) e^{-Pr \int_0^n f(s) ds} \right]_{n=0} = \frac{-1}{\int_0^{\infty} e^{-Pr \int_0^n f(s) ds} dn}$$

$$No_x = \frac{\sqrt{\frac{m+1}{2}} Re_x^{1/2}}{\int_0^{\infty} e^{-Pr \int_0^n f(s) ds} dn}$$

(cp. White)
4-78

workout Numerically.

Stanton number

(White)
4-22

$$Ch = \frac{q_w}{\rho U C_p (T_w - T_e)} = \frac{No_x k (T_w - T_e) |}{X \rho U C_p (T_w - T_e)}$$

$$= \frac{No_x k v}{X \rho U C_p \mu / \rho} = \frac{No_x}{Re_x Pr}$$

$$C_h = N \nu_x Re_x^{-1} Pr^{-1}$$

$$C_h = \frac{\sqrt{\frac{m+1}{2}} Re_x^{-1/2} Pr^{-1}}{\int_0^\infty \exp(-Pr \int_0^n f(s) ds) dn} \equiv \frac{\sqrt{\frac{m+1}{2}} Re_x^{-1/2} Pr^{-1}}{I}$$

both C_h & C_f scale with $Re_x^{-1/2}$

$$\frac{C_f}{C_h} = \frac{f''(0) \sqrt{2(m+1)} Re_x^{-1/2}}{\sqrt{\frac{m+1}{2}} Re_x^{-1/2} Pr^{-1}} \quad I$$

$$\frac{C_f}{C_h} = f''(0) Pr = 2I$$

$$\frac{C_f}{C_h} = 2 f''(0) Pr \int_0^\infty \exp(-Pr \int_0^n f(s) ds) dn$$

again, evaluate numerically, (for PS5)

Crocco-Busemann Integrals

The 2-D boundary layer equations for compressible flow can be shown to be (steady flow, Anderson p. 226)

(p. 276 in 2e. eqn no. unchanged)

Continuity $\frac{\partial}{\partial x}(\rho u) + \frac{\partial}{\partial y}(\rho v) = 0$ (A.6.27)

x-momentum $\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} = -\frac{\partial p}{\partial x} + \frac{\partial}{\partial y}(\mu \frac{\partial u}{\partial y})$ (A.6.28)

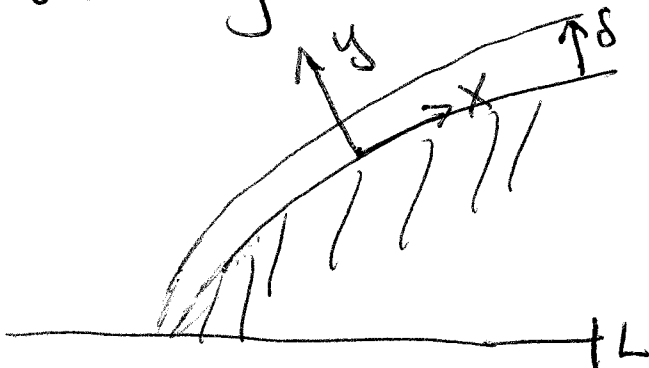
y-momentum $\frac{\partial p}{\partial y} = 0$ (A.6.29) *also always true for hypersonic*

Energy: $\rho u \frac{\partial h}{\partial x} + \rho v \frac{\partial h}{\partial y} = u \frac{\partial p}{\partial x} + \mu \left(\frac{\partial u}{\partial y}\right)^2 + \frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y}\right)$ (A.6.30)

x-momentum is similar to the incompressible equations. 4.6 covers B.L. theory, incl. some compressible flow, and cannot be reproduced here. B.L. theory assumes

$$\frac{\partial}{\partial x} \ll \frac{\partial}{\partial y}$$

$$\delta \ll L, \quad \frac{\partial}{\partial x} \ll \frac{\partial}{\partial y}$$



get a form of the energy equation that replaces h with $H = h + \frac{1}{2}u^2$, the total enthalpy.

take momentum, multiply by u , and add to energy

$$\rho u \left(\frac{\partial h}{\partial x} + u \frac{\partial u}{\partial x} \right) + \rho v \left(\frac{\partial h}{\partial y} + u \frac{\partial u}{\partial y} \right) = u \frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right) + \mu \left(\frac{\partial u}{\partial y} \right)^2 + \frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y} \right)$$

note $u \frac{\partial u}{\partial x} = \frac{\partial}{\partial x} (u^2/2)$

$$\frac{\partial}{\partial y} \left(\mu u \frac{\partial u}{\partial y} \right) = u \frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right) + \mu \frac{\partial u}{\partial y} \frac{\partial u}{\partial y}, \text{ so}$$

$$\rho u \frac{\partial H}{\partial x} + \rho v \frac{\partial H}{\partial y} = \frac{\partial}{\partial y} \left(\mu u \frac{\partial u}{\partial y} \right) + \frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y} \right)$$

$Pr = \frac{\text{delt. mom}}{\text{heat}}$

note $Pr \equiv \frac{\mu c_p}{k}$, $dh = c_p dT$ perfect gas, c_p not const.

$$\begin{aligned} \rho u \frac{\partial H}{\partial x} + \rho v \frac{\partial H}{\partial y} &= \frac{\partial}{\partial y} \left[\mu u \frac{\partial u}{\partial y} + \frac{\mu c_p}{Pr} \frac{\partial T}{\partial y} \right] \\ &= \frac{\partial}{\partial y} \left[\mu u \frac{\partial u}{\partial y} + \frac{\mu}{Pr} \frac{\partial h}{\partial y} \right] \end{aligned}$$

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Now $\frac{\partial H}{\partial y} = \frac{\partial}{\partial y} \left(h + \frac{1}{2} u^2 \right) = \frac{\partial h}{\partial y} + u \frac{\partial u}{\partial y}$, so

$$\rho u \frac{\partial H}{\partial x} + \rho v \frac{\partial H}{\partial y} = \frac{\partial}{\partial y} \left[\frac{\mu}{Pr} \left(\frac{\partial H}{\partial y} - u \frac{\partial u}{\partial y} \right) + \mu u \frac{\partial u}{\partial y} \right]$$

$$= \frac{\partial}{\partial y} \left[\frac{\mu}{Pr} \frac{\partial H}{\partial y} + u \frac{\partial u}{\partial y} \left(\mu \right) \left(1 - \frac{1}{Pr} \right) \right]$$

$$\rho u \frac{\partial H}{\partial x} + \rho v \frac{\partial H}{\partial y} = \frac{\partial}{\partial y} \left(\frac{\mu}{Pr} \frac{\partial H}{\partial y} \right) + \frac{\partial}{\partial y} \left[\left(1 - \frac{1}{Pr} \right) \mu u \frac{\partial u}{\partial y} \right]$$

White, Viscous Fluid Flow, 2e, (7-8)

If $Pr=1$, 1st Crocco-Busemann integral:

$H = \text{const}$ is a solution of this equation.

$$\Rightarrow \frac{\partial}{\partial y} (H) = 0 = \frac{\partial}{\partial y} \left(h + \frac{1}{2} u^2 \right) = \frac{\partial h}{\partial y} + u \frac{\partial u}{\partial y}$$

at wall, $u=0$, so $\frac{\partial h}{\partial y}|_w = 0$, \Rightarrow adiabatic wall.

$$\text{so } Pr=1, \text{ adiabatic wall, } \Rightarrow H = h + \frac{1}{2} u^2 = \text{const.}$$

For gases, $Pr \approx 0.7$, this is a fair approx.

Note that $k \neq 0$! $Pr=1$ means heat & momentum have same diffusivity.

Sp. to
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(4)

For $\Gamma = 1$, $\frac{\partial p}{\partial x} = 0$, not adiabatic wall, 2nd Crocco-Rosenam
integral: (steady)

mom: $\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} = \frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right)$

energy: $\rho u \frac{\partial h}{\partial x} + \rho v \frac{\partial h}{\partial y} = \frac{\partial}{\partial y} \left(\frac{\mu}{\Gamma} \frac{\partial h}{\partial y} \right) + \mu \left(\frac{\partial u}{\partial y} \right)^2$

Same equation almost, except for dissipation term.

Try $h = h(u)$ as a solution: $\frac{\partial h}{\partial y} = \frac{dh}{du} \frac{\partial u}{\partial y}$

~~energy~~ energy becomes:

$$\rho u \frac{dh}{du} \frac{\partial u}{\partial x} + \rho v \frac{dh}{du} \frac{\partial u}{\partial y} = \frac{\partial}{\partial y} \left(\mu \frac{dh}{du} \frac{\partial u}{\partial y} \right) + \mu \left(\frac{\partial u}{\partial y} \right)^2$$

Use momentum on l.h.s.

$$\frac{dh}{du} \left[\frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right) \right] = \frac{dh}{du} \left[\frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right) \right] + \mu \frac{\partial u}{\partial y} \frac{\partial}{\partial y} \left(\frac{dh}{du} \right) + \mu \left(\frac{\partial u}{\partial y} \right)^2$$

$$0 = \mu \frac{\partial u}{\partial y} \frac{\partial}{\partial y} \left(\frac{dh}{du} \right) + \mu \left(\frac{\partial u}{\partial y} \right)^2$$

Note: $\frac{\partial}{\partial y} \left(\frac{dh}{du} (u(x,y)) \right) = \frac{d^2 h}{du^2} \frac{\partial u}{\partial y}$

(5)

$$0 = \mu \frac{\partial u}{\partial y} \frac{d^2 h}{du^2} \frac{\partial u}{\partial y} + \mu \left(\frac{\partial u}{\partial y} \right)^2$$

$$0 = \mu \left(\frac{\partial u}{\partial y} \right)^2 \left[\frac{d^2 h}{du^2} + 1 \right]$$

Satisfied iff $\frac{d^2 h}{du^2} = -1 \Rightarrow \frac{dh}{du} = -u + C_1$

$$h = -\frac{u^2}{2} + C_1 u + C_2$$

Since $u=0$ at wall, $h=h_w=C_2$

Since $h=h_{ext} \equiv h_e$ at $u=U_e$, $h_e = -\frac{U_e^2}{2} + C_1 U_e + h_w$

$$C_1 = \frac{h_e - h_w + \frac{U_e^2}{2}}{U_e} \quad \int h + \frac{u^2}{2} = C_1 u + C_2$$

So 2nd Crocco-Busemann integral is

$$Pr=1, \frac{dp}{dx}=0, H = h + \frac{u^2}{2} = h_w + \left(\frac{u}{U_e} \right) \left(h_e - h_w + \frac{U_e^2}{2} \right)$$

$$\text{or } H = h_w + \frac{u}{U_e} (H_e - h_w)$$

So H varies linearly with u across B.L.

if $U_e = h_w = H = \text{const}$, recover 1st Crocco-Busemann.

(6)

Note: recovery factor (empirical)

if $h = c_p T$, $c_p = \text{const}$, $Pr = 1$, $dp/dx = 0$,

$$T = T_w + \left(T_e + \frac{U_e^2}{2c_p} - T_w \right) \frac{u}{U_e} - \frac{u^2}{2c_p}$$

if wall is adiabatic, $H = \text{const} \Rightarrow H_w = H_e \Rightarrow$

$$T_e + \frac{U_e^2}{2c_p} = T_{aw}, \text{ so}$$

$$T = T_w + (T_{aw} - T_w) \frac{u}{U_e} - \frac{u^2}{2c_p}$$

Semi-empirical correction factor, for $Pr \neq 1$, etc, is r ,

$$T = T_w + (T_{aw} - T_w) \frac{u}{U_e} - r \frac{u^2}{2c_p}$$

if $T_w < T_{aw}$, this term cause T to increase with u away from wall.

if $T_w > T_{aw}$, this term causes T to decrease with u away from wall.

note $\# q_w = k_w \frac{\partial T}{\partial y}|_w = \left(-\frac{T_{aw} - T_w}{U_e} \frac{\partial u}{\partial y}|_w + \frac{r}{c_p} 2u \frac{\partial u}{\partial y}|_w \right) k$

$$q_w = - \frac{T_{aw} - T_w}{\mu_e \mu_w} \approx T_w k$$

q_w is $\sim T_w$, Reynolds analogy.

also q_w proportional to $T_w - T_{aw}$ - T_w has to be more than T_{aw} to ~~radio~~ conduct heat into the fluid. Very hard at hypersonic speeds.

Next, do B.L. similarity equations, leading to stagnation-point solution for laminar perfect-gas flow.

Jan 01 - First, do Falkner-Skan (Do Blasius as warmup.)
Do main solution for flow, similar to 4/6
Do heat-transfer solution in class

3701

cf White sec. 7.2, Schlichting,
Anderson sec. 6.5

4 Feb 99
SPS

Illingworth-Stewartson ... Transformations
for Compressible B.L. (steady flow) 2D

Continuity: $\frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} = 0$

let $\rho u = \frac{\partial \psi}{\partial y}$ and $\rho v = -\frac{\partial \psi}{\partial x}$, to satisfy identically.

Try similarity variables on PDE's —
one for streamfunction and one for compressibility effects.

let $\psi(\xi, \eta) = \int \rho u dy \equiv G(\xi) f(\eta) = \psi(x, y)$

use Illingworth transformation, \uparrow differs from Anderson, who allows $f(\xi, \eta)$

$\xi \equiv \int_0^x \rho_e(x) u_e(x) \mu_e(x) dx = \xi(x)$ only (simplifies)

$\eta \equiv \frac{u_e}{\sqrt{2\xi}} \int_0^y \rho dy = \eta(x, y)$ (typo in Anderson 6.34!)

First, get $u(\xi, \eta) = \frac{1}{\rho} \frac{\partial \psi}{\partial y}$

$\frac{\partial \psi}{\partial y} = G(\xi) \frac{df}{d\eta} \frac{\partial \eta}{\partial y} = G(\xi) \frac{df}{d\eta} \frac{u_e}{\sqrt{2\xi}} \rho$

modified from 1990 613 notes, to get early G for $u=f(\eta)$ statement brominated earlier.

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$$\text{so } u(\xi, \eta) = \frac{1}{\rho} G(\xi) F'(\eta) \frac{u_e}{\sqrt{2\xi}} \quad \cancel{\rho}$$

let $G(\xi) = \sqrt{2\xi}$ so this simplifies, and because it works. (see Anderson 'Step III')

the $u(\xi, \eta) = u_e(\xi) f'(\eta)$

Now substitute into steady-flow momentum eqn. (2D)

$$\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} = -\frac{\partial p_e}{\partial x} + \frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right) \quad \left(P = P_e \text{ because } \frac{\partial P}{\partial y} = 0 \right)$$

$$\frac{\partial u}{\partial x} = \frac{\partial u}{\partial \eta} \frac{\partial \eta}{\partial x} + \frac{\partial u}{\partial \xi} \frac{\partial \xi}{\partial x} \quad \text{since } u = u(\eta(x, y), \xi(x))$$

$$\frac{\partial u}{\partial y} = \frac{\partial u}{\partial \eta} \frac{\partial \eta}{\partial y} \quad \text{since } \frac{\partial \xi}{\partial y} = 0$$

$\frac{\partial \eta}{\partial x}$ will drop out (see Anderson, and watch), so don't evaluate, messy.

$$\frac{\partial \eta}{\partial y} = \frac{u_e}{\sqrt{2\xi}} \rho, \quad \frac{\partial \xi}{\partial x} = \rho e^{u_e \eta_e}, \quad \text{and}$$

$$\frac{\partial u}{\partial \eta} = u_e f''(\eta), \quad \frac{\partial u}{\partial \xi} = u_e'(\xi) f'(\eta)$$

Substitute

$$\rho U_e f' \left[U_e f'' \frac{\partial \eta}{\partial x} + U_e' f' \rho_e U_e \eta_e \right]$$

$$- \frac{\partial \psi}{\partial x} \left[U_e f'' \frac{U_e}{\sqrt{2\xi}} \rho \right] = - \frac{\partial p_e}{\partial x} + \frac{\partial}{\partial y} \left(\mu U_e f'' \frac{U_e}{\sqrt{2\xi}} \rho \right)$$

Now $\rho_e + \frac{1}{2} \rho_e U_e^2 = \text{const}$, so

homog. forced shock
 $\rho_e f' \text{ const}$

$$\frac{\partial p_e}{\partial x} = - \frac{1}{2} \rho_e U_e \frac{dU_e}{dx}$$

(from x-mom with $\frac{\partial}{\partial y} = 0$ & $\mu = 0$)

Euler along streamline

(A., Fund. sec 3.2)

$$\text{Now } \frac{dU_e(x)}{dx} = \frac{d}{dx} (U_e(\xi(x))) = \frac{dU_e}{d\xi} \rho_e U_e \eta_e$$

$$\frac{\partial \psi}{\partial x} = \frac{\partial \psi}{\partial \eta} \frac{\partial \eta}{\partial x} + \frac{\partial \psi}{\partial \xi} \frac{\partial \xi}{\partial x}, \quad \psi = G(\xi) f(\eta) = \sqrt{2\xi} f(\eta)$$

$$\frac{\partial \psi}{\partial x} = \sqrt{2\xi} f'(\eta) \frac{\partial \eta}{\partial x} + \frac{1}{2} (2\xi)^{-1/2} 2 f(\eta) \rho_e U_e \eta_e$$

$$\frac{\partial \psi}{\partial x} = \sqrt{2\xi} f' \frac{\partial \eta}{\partial x} + \frac{\rho_e U_e \eta_e}{\sqrt{2\xi}} f$$

plug these in to momentum

$$\cancel{\rho u_e^2 f' f'' \frac{\partial \eta}{\partial x}} + \rho u_e (f')^2 u_e' e u_e M_e$$

$$\cancel{-\sqrt{2\xi} f' \frac{\partial \eta}{\partial x} \frac{\rho u_e^2}{\sqrt{2\xi}} f''} - \frac{\rho u_e M_e}{\sqrt{2\xi}} f \frac{\rho u_e^2}{\sqrt{2\xi}} f''$$

$$= \rho u_e \frac{du_e}{dx} + \frac{u_e^2}{\sqrt{2\xi}} \frac{\partial}{\partial y} \left(\frac{\rho u_e}{\rho M_e} f'' \right)$$

$\frac{\partial \eta}{\partial x}$ terms cancel! Define $\frac{\rho u_e}{\rho M_e} \equiv C$, Chapman-Rubensin parameter

$$\rho u_e u_e' e u_e M_e (f')^2 - \frac{\rho u_e^2 \rho u_e M_e}{2\xi} f f''$$

$$= \rho u_e^2 \frac{1}{u_e} \frac{du_e}{dx} + \rho u_e \frac{u_e^2}{\sqrt{2\xi}} \frac{\partial}{\partial y} \left(\frac{\rho u_e}{\rho M_e} f'' \right)$$

for last term, note

$$\frac{\partial J(\eta, \xi)}{\partial y} = \frac{\partial J}{\partial \eta} \frac{\partial \eta}{\partial y} + \frac{\partial J}{\partial \xi} \frac{\partial \xi}{\partial y} \stackrel{\rightarrow 0}{=} \frac{\partial J}{\partial \eta} \frac{\rho u_e}{\sqrt{2\xi}}, \text{ so}$$

$$\frac{\partial}{\partial y} \left(\frac{\rho u_e}{\rho M_e} f'' \right) = \frac{\partial}{\partial \eta} (C f'') \frac{\rho u_e}{\sqrt{2\xi}}$$

Substitute & divide through by $e^{-u} e^2$

$$\rho u' e^{-u} (f')^2 - \frac{\rho u e^{-u}}{2\xi} f f''$$

$$= \frac{\rho}{e^{-u}} \frac{1}{u} \frac{du}{d\xi} e^{-u} u e^{-u} + \frac{\rho u e^{-u}}{\sqrt{2\xi}} \frac{\rho u e^{-u}}{\sqrt{2\xi}} \frac{\partial}{\partial n} (C f'')$$

(divide by e^{-u}), so

$$e^{-u} (f')^2 - \frac{\rho u e^{-u}}{2\xi} f f'' = \rho \frac{du}{d\xi} + \frac{\rho u e^{-u}}{2\xi} (C f'')$$

multiply by $\frac{2\xi}{\rho u e^{-u}}$ & switch sides to put leading term on left.

$$(C f'')' + \frac{2\xi}{\rho u e^{-u}} \frac{du}{d\xi} - \frac{2\xi}{\rho u e^{-u}} (f')^2 + \frac{\rho u e^{-u}}{2\xi} f f'' \cdot \frac{2\xi}{\rho u e^{-u}} = 0$$

$$(C f'')' + \frac{2\xi \rho}{u e^{-u}} \frac{du}{d\xi} - \frac{2\xi}{u e^{-u}} (f')^2 + f f'' = 0$$

$$(C f'')' + \frac{2\xi}{u e^{-u}} \frac{du}{d\xi} \left(\frac{e^{-u}}{e^{-u}} - (f')^2 \right) + f f'' = 0$$

ck. White^{2e} eqn. 7-20, Anderson 6.55 (here $f=f(\eta)$ only is already assumed)

B.C. (solid wall) $v(y=0)=0$

$$\rho v = -\frac{\partial \psi}{\partial x} = \frac{\partial}{\partial x} \left[\sqrt{2\xi} f' \frac{\partial \eta}{\partial x} + \frac{\rho u_e \eta e}{\sqrt{2\xi}} f \right]$$

recall $\eta = \frac{u_e}{\sqrt{2\xi}} \int_0^y \rho dy$ (typo in Anderson 6.34!)

so at $y=0, \eta=0$

in general, for $v(\eta=0)=0, \boxed{f'(0) = f(0) = 0}$

no-slip: $u(\eta=0)=0 = u_e f'(\eta=0)$, also works with the above.

Freestream match:

$$u(\xi, \eta) \rightarrow u_e(\xi) \text{ as } \eta \rightarrow \infty, u = u_e f'$$

$$\Rightarrow \boxed{f'(\eta \rightarrow \infty) \rightarrow 1}$$

Note: if $M \rightarrow 0, \mu, \rho, u_e$ const, $C=1$, Blasius, get

(white 445)

$$f'''' + f f'' = 0 \quad \left(\frac{d u_e}{d \xi} = 0 \right)$$

$$\xi = \rho u_e x, \quad \eta = \frac{u_e}{\sqrt{2\rho u_e x}} \rho y = y \sqrt{\frac{\rho u_e}{2 u_e x}} = y \sqrt{\frac{\rho}{2 x}}$$

reduces to Blasius (white 442)

Energy Equation: try $h(x,y) = h_e(\xi)g(\eta)$

again differs from Anderson, $g = g(\eta)$ only.

steady, 2D,

$$\rho u \frac{\partial h}{\partial x} + \rho v \frac{\partial h}{\partial y} = u \frac{\partial p_e}{\partial x} + \mu \left(\frac{\partial u}{\partial y} \right)^2 + \frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y} \right)$$

but $dh = c_p dT$ for perfect gas, and $\frac{k}{c_p \mu} = \frac{1}{Pr}$, so $\frac{k}{c_p} = \frac{\mu}{Pr}$

$$\rho u \frac{\partial h}{\partial x} + \rho v \frac{\partial h}{\partial y} = u \frac{\partial p_e}{\partial x} + \mu \left(\frac{\partial u}{\partial y} \right)^2 + \frac{\partial}{\partial y} \left(\frac{\mu}{Pr} \frac{\partial h}{\partial y} \right)$$

again, $\frac{\partial p_e}{\partial x} = -\rho_e u_e \frac{du_e}{dx} = -\rho_e u_e \frac{du_e}{d\xi} \rho_e u_e M_e$

and $\frac{\partial h}{\partial x} = \frac{\partial h}{\partial \eta} \frac{\partial \eta}{\partial x} + \frac{\partial h}{\partial \xi} \frac{\partial \xi}{\partial x} = h_e g'(\eta) \frac{\partial \eta}{\partial x} + h_e' g \rho_e u_e M_e$

$u = u_e f'$ from before

$$\rho v = -\frac{\partial \psi}{\partial x} = - \left[\sqrt{2\xi} f' \frac{\partial \eta}{\partial x} + \frac{\rho_e u_e M_e}{\sqrt{2\xi}} f \right]$$

Anderson drops this term. Not clear why. 5th printing from before

$$\frac{\partial h}{\partial y} = \frac{\partial h}{\partial \eta} \frac{\partial \eta}{\partial y} + \frac{\partial h}{\partial \xi} \frac{\partial \xi}{\partial y} = h_e g' \frac{u_e}{\sqrt{2\xi}} \rho_e$$

$\frac{\partial \eta}{\partial y}$ from before

$$\frac{\partial u}{\partial y} = \frac{\partial u}{\partial \eta} \frac{\partial \eta}{\partial y} + \frac{\partial u}{\partial \xi} \frac{\partial \xi}{\partial y} = u_e f'' \frac{u_e}{\sqrt{2\xi}} \rho_e$$

-3-19-01

plug in

$$\begin{aligned} & \rho u_e f' \left[h e g' \frac{\partial n}{\partial x} + h e' g' \rho e u_e M_e \right] \\ & - \left[\sqrt{2\xi} f' \frac{\partial n}{\partial x} + \frac{\rho e u_e M_e}{\sqrt{2\xi}} f \right] h e g' \frac{u_e}{\sqrt{2\xi}} \rho \\ & = u_e f' \left[-\rho^2 u_e^2 M_e \frac{du_e}{d\xi} \right] + M u_e^2 (f''')^2 \frac{u_e^2}{2\xi} \rho^2 \\ & \quad + \frac{\partial}{\partial y} \left(\frac{M}{Pr} h e g' \frac{u_e}{\sqrt{2\xi}} \rho \right) \end{aligned}$$

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2999
w/ $\frac{\partial n}{\partial x}$ dropped
(canceled)

remember $\frac{\partial J}{\partial y} = \frac{\partial J}{\partial n} \frac{\rho u_e}{\sqrt{2\xi}}$, so

$$\begin{aligned} & \cancel{\rho u_e f' h e g' \frac{\partial n}{\partial x}} + \rho u_e^2 f' h e' g' \rho e M_e \\ & - \cancel{f' \frac{\partial n}{\partial x} h e g' u_e \rho} - \frac{\rho e u_e^2 M_e}{2\xi} f h e g' \rho \\ & = -\rho^2 u_e^3 M_e f' \frac{du_e}{d\xi} + \frac{M u_e^4 \rho^2 (f''')^2}{2\xi} \\ & \quad + \frac{\partial}{\partial n} \left(\frac{M \rho}{Pr} g' \right) \frac{h e u_e}{\sqrt{2\xi}} \frac{\rho u_e}{\sqrt{2\xi}} \end{aligned}$$

(9)

$\frac{\partial \eta}{\partial X}$ terms again cancel out (part of reason for $\sqrt{2\xi}$ defn.)

again define $C \equiv \frac{e\mu}{e\eta e}$

$$e\eta^2 f' h' g e\eta e - e\eta^2 e\eta e e \frac{fg'}{2\xi}$$

$$= -e^2 \eta e^{\xi'} \eta f' \frac{d\eta}{d\xi} + \frac{\mu \eta e^{\xi'} e^2 (f'')^2}{2\xi} + \left(\frac{C}{Pr} g'\right)' \frac{e\eta^2 h' e\eta e}{2\xi}$$

divide by η^2 . multiply by 2ξ

$$e h' e\eta e (f' g) 2\xi - e e\eta e e fg'$$

$$= -e^2 \eta e\eta e \frac{d\eta}{d\xi} f' 2\xi + \mu \eta e^2 e^2 (f'')^2 + \left(\frac{C}{Pr} g'\right)' e\eta e\eta e$$

switch sides & move terms

$$\left(\frac{C}{Pr} g'\right)' e e e\eta e + e e\eta e e fg'$$

$$= 2\xi \left[e e e h' \eta e f' g + e^2 \eta e\eta e \frac{d\eta}{d\xi} f' \right] - e^2 \mu \eta e^2 (f'')^2$$

divide by $e e e\eta e$

$$\left(\frac{C}{Pr} g'\right)' + fg' = 2\epsilon \left[\frac{\eta_e' \mu_e f' g}{\eta_e \mu_e} + \frac{\rho_e \mu_e \frac{d\mu_e}{d\epsilon} f'}{\rho_e \eta_e} \right] - \frac{\rho_e \mu_e \mu_e^2}{\rho_e \mu_e \eta_e} (f'')^2$$

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$$\left(\frac{C}{Pr} g'\right)' + fg' = 2\epsilon \left[\frac{\eta_e' f' g}{\eta_e} + \frac{\rho_e \mu_e \frac{d\mu_e}{d\epsilon} f'}{\rho_e \eta_e} \right] - C \frac{\mu_e^2}{\eta_e} (f'')^2$$

~~Anderson 6.58 is missing~~ (2nd printing) ~~(this term)~~ (present in 5th printing)
 he has $\frac{\partial g}{\partial \epsilon}$ & $\frac{\partial f}{\partial \epsilon}$ terms that are both zero by definition/assumption here.

in general, there is no reason why $\frac{d\eta_e}{d\epsilon} = 0$.

Lees (1956) assumes $\frac{d\eta_e}{d\epsilon} = 0$, which is reasonable, although not true where heat conduction in the entropy layer is significant.

also differs from White (7-25)

(6.54), we finally obtain

$$(Cf'')' + ff'' = \frac{2\xi}{u_e} \left[(f')^2 - \frac{\rho_e}{\rho} \right] \frac{du_e}{d\xi} + 2\xi \left(f' \frac{\partial f'}{\partial \xi} - \frac{\partial f}{\partial \xi} f'' \right) \quad (6.55)$$

Eq. (6.55) is the transformed boundary layer x-momentum equation for a two-dimensional, compressible flow.

The boundary layer y-momentum equation, namely Eq. (6.29) stating that $\partial p / \partial y = 0$ becomes in the transformed space

$$\frac{\partial p}{\partial \eta} = 0 \quad (6.56)$$

The boundary layer energy equation given by Eq. (6.30) can also be transformed. Defining a nondimensional static enthalpy as

$$g = g(\xi, \eta) = \frac{h}{h_e} \quad (6.57)$$

where h_e is the static enthalpy at the boundary layer edge, and utilizing the same transformation as before, Eq. (6.30) becomes

$$\left(\frac{C}{Pr} g' \right)' + fg' = 2\xi \left[f' \frac{\partial g}{\partial \xi} + \frac{f'g}{h_e} \frac{\partial h_e}{\partial \xi} - g' \frac{\partial f}{\partial \xi} + \frac{\rho_e u_e}{\rho h_e} f' \frac{du_e}{d\xi} \right] - C \frac{u_e^2}{h_e} (f'')^2 \quad (6.58)$$

— here for 3-30-90 —

(148)

$$\frac{\partial}{\partial \eta} \left(\frac{Cg'}{Pr} \right) + fg' = \frac{2\epsilon f'}{h_e} \left[\frac{c_e u_e}{c} \frac{du_e}{d\xi} + g \frac{dh_e}{d\xi} \right] - \frac{C}{h_e} u_e^2 (f'')^2$$

energy equation (does not check with 7-25)

61. Flat Plate in Compressible Flow

for flat plate (or cone) $\frac{dh_e}{d\xi} = 0$, $\frac{du_e}{d\xi} = 0$ (constant
outer
conditions)

eqns become:

$$\left[(Cf'')' + ff'' = 0 \right] \text{ (momentum BEqn)}$$

$$\left(\frac{Cg'}{Pr} \right)' + fg' = -\frac{C}{h_e} u_e^2 (f'')^2 \text{ (energy BEqn)}$$

for a perfect gas with constant specific heats,

$$\begin{aligned} \frac{u_e^2}{h_e} &= \frac{u_e^2}{c_p T_e} = \frac{u_e^2}{c_p} \frac{\gamma R}{\gamma R T_e} = \frac{u_e^2 \gamma (c_p \bar{c})}{c_p a_e^2} = M_e^2 \frac{\gamma(\gamma-1)}{\gamma} \\ &= M_e^2 (\gamma-1) \end{aligned}$$

$$\text{so } \left[\left(\frac{Cg'}{Pr} \right)' + fg' = -C M_e^2 (\gamma-1) (f'')^2 \right]$$

- here 11-3

B.C. $f(0) = f'(0) = 0, f'(\infty) = 1;$
 $g(\infty) = 1$ (so $h = h_e$) \oplus

heat transfer: $-q_w = k_w \frac{\partial T}{\partial y}|_w = \frac{k_w}{c_p} \frac{\partial h}{\partial y}|_w$

$-q_w = \frac{k_w h_e c_p}{c_p \sqrt{2\xi}} g'|_{\eta=0}$

for adiabatic wall, $q_w = 0 \Rightarrow g'(0) = 0$ (adiabatic)

or $g(0) = 1$ (cold wall, $h_w = h_e$)

+ need an eqn for $C(h)$ (if $c = \text{const}$, change variables)
 Use Blasius soln

Compressible displacement thickness:

~~$\delta^* = \int_0^\infty (1 - \frac{\rho y}{\rho_e h_e}) dy = \int_0^\infty (1 - \frac{\rho}{\rho_e} f') dy$~~

$C = \frac{c_p}{c_p h_e} ; \frac{\mu}{\mu_0} = \left(\frac{T}{T_0}\right)^{3/2} \frac{T_0 + S}{T + S}$

Air: $T_0 = 491.6 R, S = 199^\circ R$

$\mu_0 = 0.1716 \text{ mP}$

OK for 300 - 3420 R (white)

$P = CRT \Rightarrow \frac{P}{RT} \frac{R}{c_p} = \frac{c_p}{c_p} = \frac{T_e}{T}$

Perf gas w/ const c_p

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$$\frac{\mu}{\mu_e} = \frac{\mu/\mu_0}{\mu_e/\mu_0} = \frac{\left(\frac{T}{T_0}\right)^{3/2} \frac{T_0 S}{T+S}}{\left(\frac{T_e}{T_0}\right)^{3/2} \frac{T_0 S}{T_e+S}} = \left(\frac{T}{T_e}\right)^{3/2} \frac{T_e+S}{T+S}$$

so $C \approx \frac{T_e}{T} \left(\frac{T}{T_e}\right)^{3/2} \frac{T_e+S}{T+S} \approx \left(\frac{T}{T_e}\right)^{1/2} \frac{T_e+S}{T+S}$ for range given.

$$C \approx \left(\frac{h}{h_e}\right)^{1/2} \left(\frac{h_e + c_p S}{h + c_p S}\right) \quad \text{but } g = h/\mu_e$$

$$C \approx \sqrt{g} \left(\frac{h_e + c_p S}{g h_e + c_p S}\right) \quad \text{he known, } c_p, S \text{ known}$$

have to solve numerically. ~~see~~ plots -

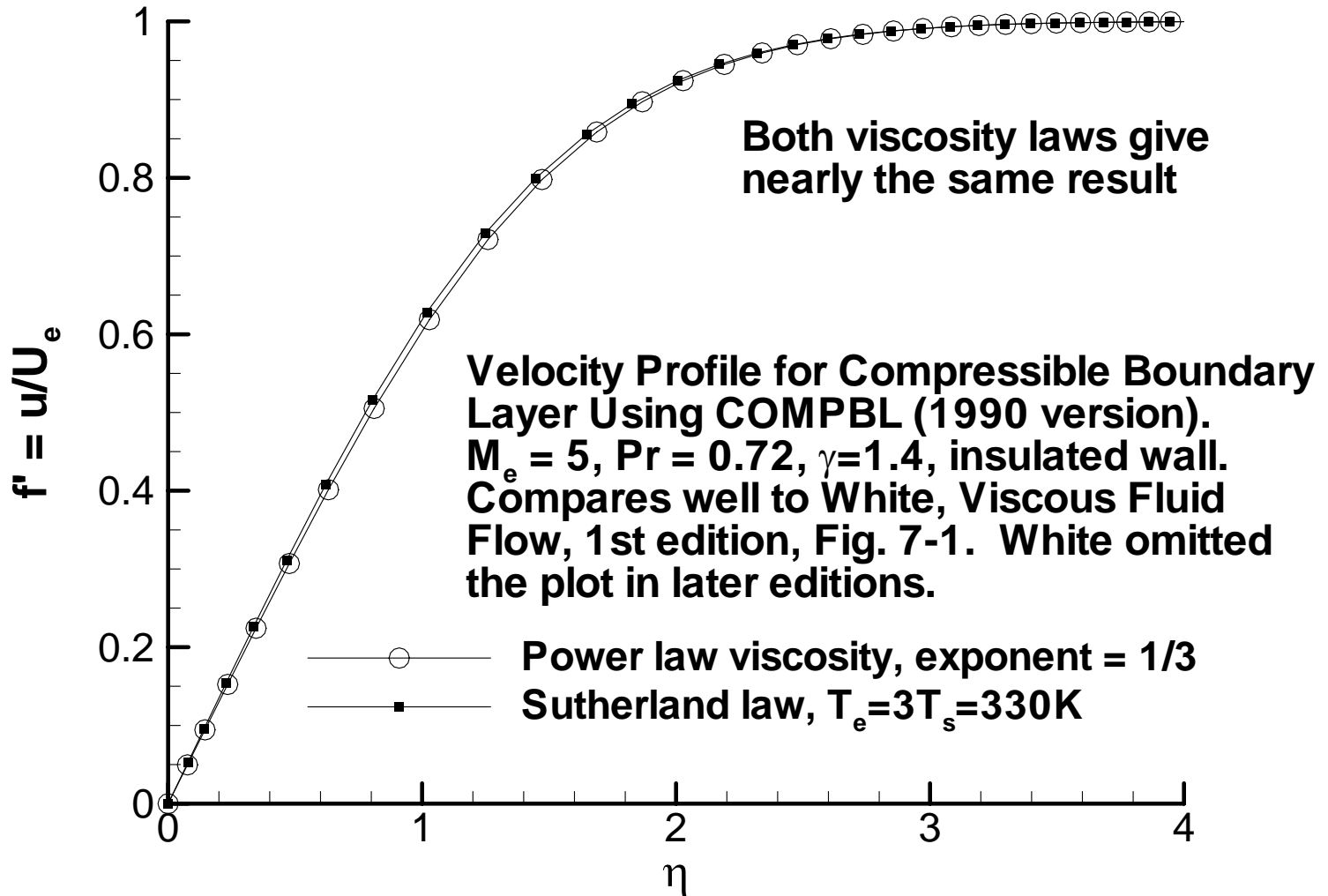
Work
heat transfer
of wall
temp. for
x/S

(Beware of dissociation effects, nonequilibrium chemistry effects)

H. Approximations for Flat Plate in Compressible Flow

- Reference Temperature

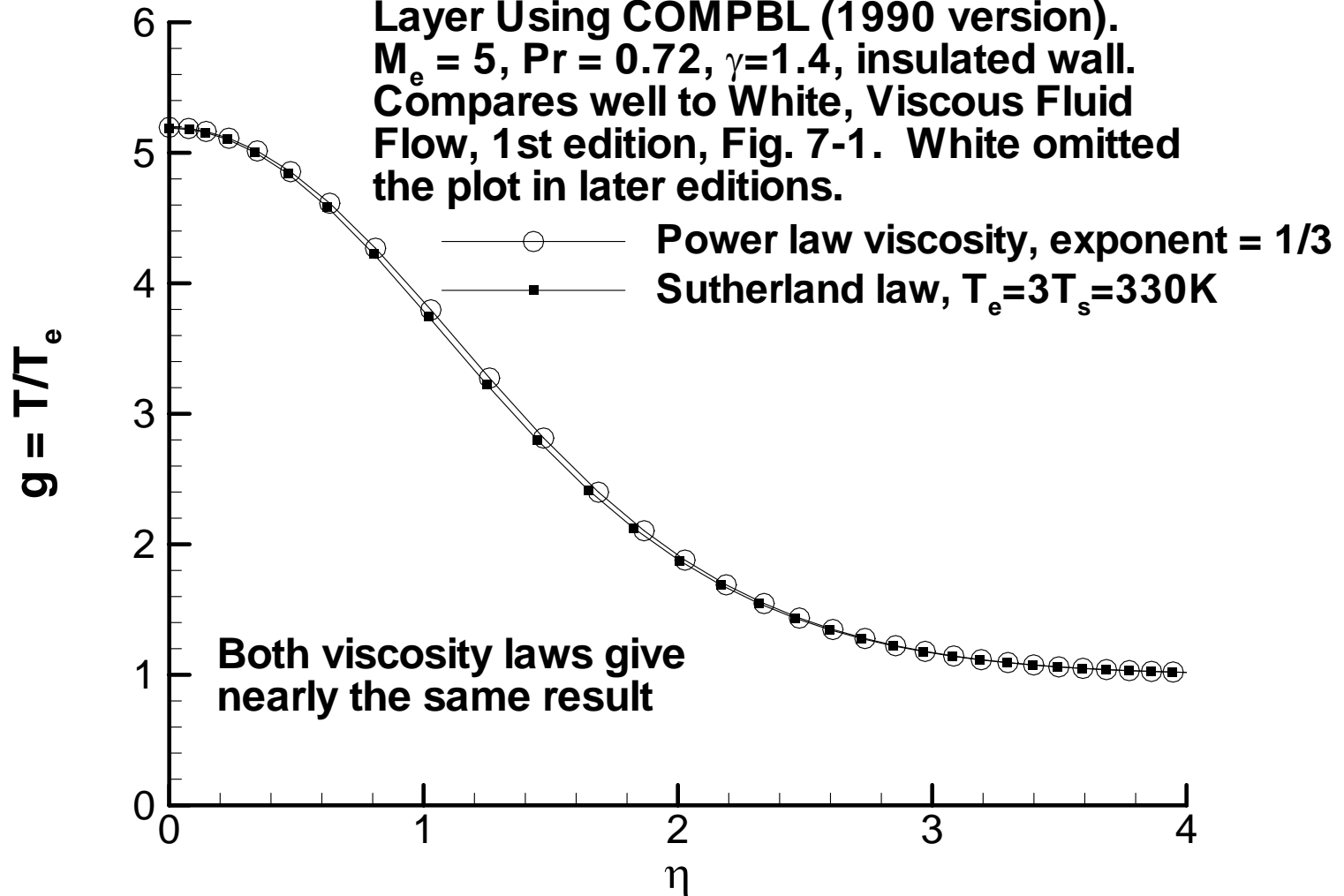
Assume that C is a constant, so that can solve the momentum equation using the Blasius solns. Pick a "reference temperature" T^* to be an appropriate characteristic temp. at which to evaluate C (and

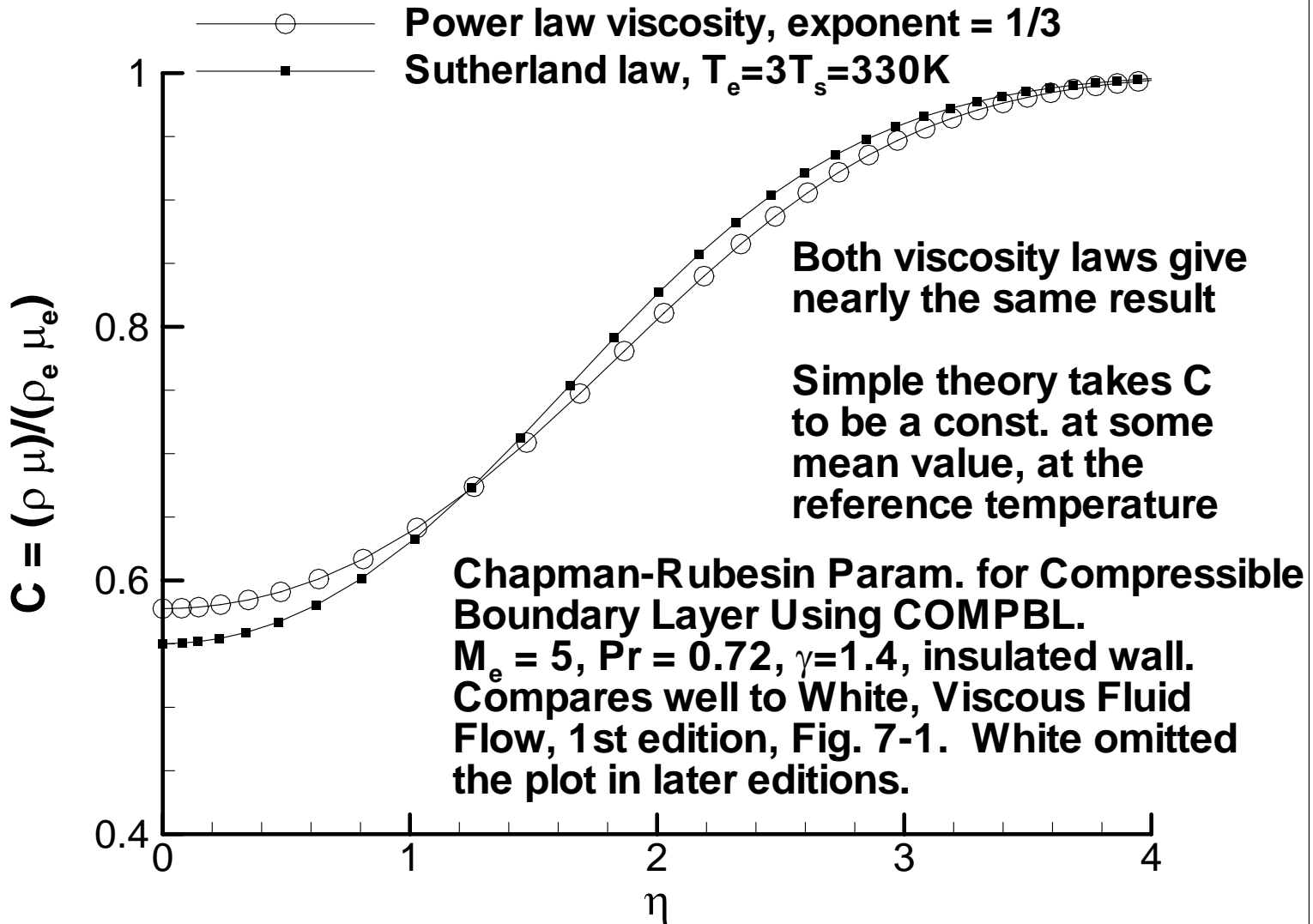


Temperature Profile for Compressible Boundary Layer Using COMPBL (1990 version).

$M_e = 5$, $Pr = 0.72$, $\gamma = 1.4$, insulated wall.

Compares well to White, Viscous Fluid Flow, 1st edition, Fig. 7-1. White omitted the plot in later editions.





and so on). Then

$$\frac{d}{d\eta} \left(c \frac{d^2 f}{d\eta^2} \right) + f \frac{d^2 f}{d\eta^2} = 0 \quad f(0) = f'(0) = 0, f'(\infty) = 1$$

let $\eta = c_0 x, f = c_1 \tilde{f}, c_1 \& c_0 = \text{const.}$

$$\frac{1}{c_0^3} c_1 \frac{d}{dx} \left(c \frac{d^2 \tilde{f}}{dx^2} \right) + \frac{c_1^2}{c_0^2} \tilde{f} \frac{d^2 \tilde{f}}{dx^2} = 0$$

$$\text{want } \tilde{f}(x=0) = 0, \tilde{f}'(x=0) = 0, \tilde{f}'(x=\infty) = 1$$

so \tilde{f} is Blasius

1st two come automatically from B.C. on f.

$$\text{3rd means } \frac{d\tilde{f}}{dx} = \frac{c_0}{c_1} \frac{df}{d\eta} = \frac{c_0}{c_1} \text{ at } x \rightarrow \infty = 1$$

need $c_1 = c_0$.

eqn becomes

$$\frac{1}{c_0 c_1} c \frac{d^3 \tilde{f}}{dx^3} + \tilde{f} \frac{d^2 \tilde{f}}{dx^2} = 0$$

$$\text{need } c = c_0 c_1 = c_0^2 \Rightarrow c_0 = (c)^{1/2} = c_1$$

$$\text{then } \tilde{f}''' + \tilde{f} \tilde{f}'' = 0 \quad \tilde{f}(x) = f_{\text{Blasius}}(x) = f_B(x)$$

$$\text{So } f(\eta) = c_1 \tilde{f}(\eta) = c_1 \tilde{f}(\cos \alpha) = c_1 f_B(\cos \alpha) \text{ (Blasius)}$$

$$(f(\eta=1) = c_1 \tilde{f}(\eta=1) = c_1 \tilde{f}(\cos \alpha=1) = \text{etc})$$

$$\frac{df}{d\eta} = f' = \frac{u}{U_e} = \frac{c_1}{c_0} \frac{d\tilde{f}}{d\alpha} = \tilde{f}' = f'_B(\cos \alpha)$$

$$\frac{d^2f}{d\eta^2} = f'' = \frac{c_1}{c_0^2} \frac{d^2\tilde{f}}{d\alpha^2} = \frac{1}{\sqrt{C^*}} \tilde{f}'' = \frac{f''_B(\cos \alpha)}{\sqrt{C^*}}$$

$$\text{So } f''(0) = \frac{f''_B(0)}{\sqrt{C^*}} = \frac{0.4696}{\sqrt{C^*}}$$

$$\tau_w = \mu \left. \frac{\partial u}{\partial y} \right|_0 = \mu \left. \frac{\partial}{\partial y} (U_e f') \right|_0 = \frac{\mu U_e^2}{\sqrt{2} \xi} f''(0)$$

$$\xi = \int^x \rho_e U_e \mu_e dx = \rho_e \mu_e U_e x$$

$$\tau_w = \frac{\rho \mu U_e^2}{\sqrt{2} \sqrt{\rho_e \mu_e U_e x}} f''(0) = \frac{\rho \mu \sqrt{\rho_e \mu_e}}{\rho_e \mu_e \sqrt{U_e x}} \frac{U_e^2}{\sqrt{2}} f''(0)$$

$$\tau_w = C \rho_e \sqrt{\frac{\mu_e}{U_e \rho_e x}} \frac{U_e^2}{\sqrt{2}} f''(0) = C \rho_e \left(\text{Re}_{\text{ext}, x} \right)^{-1/2} \frac{U_e^2}{\sqrt{2}} f''(0)$$

$$\tau_w = C \rho_e \left(\text{Re}_{\text{ext}, x} \right)^{-1/2} \frac{U_e^2}{\sqrt{2}} \frac{0.4696}{\sqrt{C^*}} = \sqrt{\frac{C^*}{2}} \left(\text{Re}_{\text{ext}, x} \right)^{-1/2} \cdot 0.4696 \rho_e U_e^2$$

FORM 3
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Reference temp. Idea: Use this approx, and pick C as some average, so that profiles are correct. Since C depends on temp. alone, to good approx, evaluate C at some "reference temp" T^* . Can do empirically or using integral form of eqns. (White p-590-91).

$$\left[\frac{T^*}{T_e} \approx 0.5 + 0.5 \frac{T_w}{T_e} + 0.039 Ma_e^2 \right] \text{ White (7-46b)}$$

Is one good approx. Numerical solns show $r \approx \sqrt{Pr} = \tau_w^{1/2} \rho^{1/2} c_p^{1/2}$ here Mon 2 April (skipped deriv.) White 7-3

I. B.L. Displacement Thickness for Similarity Solns

flat plate

$$\delta^* \equiv \int_0^{\infty} \left(1 - \frac{\rho u}{\rho_e u_e}\right) dy = \int_0^{\infty} \left(1 - \frac{f'}{f'_e}\right) dy$$

but $\frac{\rho}{\rho_e} = \frac{T_e}{T} = \frac{h_e}{h}$ since $\frac{dp}{dy} = 0$, and if $h \propto \rho T$

so $\frac{\rho}{\rho_e} = \frac{1}{g}$ where $g \equiv \frac{h}{h_e}$

$$\Rightarrow \delta^* = \int_0^{\infty} \left(1 - \frac{f'}{g}\right) dy.$$

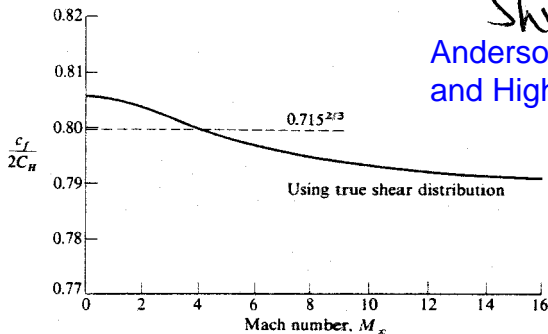
5th printing, from
Skin. 2-99Anderson, Hypersonic
and High-Temp. Gasd.

FIGURE 6.13

Comparison of exact and approximate Reynolds analogy factor for laminar flow over a flat plate. (Ref. 90.)

is expressed by Eq. (6.83); numerical solutions are given in Fig. 6.13, obtained from Ref. 90. Note that the ratio c_f/C_H decreases as M_e increases across the hypersonic regime. However, again note that the ordinate is an expanded scale, and c_f/C_H decreases by only 2 percent from $M_e = 0$ to 16. Thus, the incompressible result given by Eq. (6.82) is a reasonable approximation at hypersonic speeds, namely

$$\frac{C_H}{c_f} = \frac{1}{2} \text{Pr}^{-2/3} \quad (6.91)$$

But need $f^*(\eta)$ to integrate similarity soln:

$$\eta = \frac{Ue}{\sqrt{2\xi}} \int_0^y e^{\frac{y}{\xi}} dy \quad ; \quad \xi = \int_0^x e^{\frac{x}{\xi}} Ue^{\frac{x}{\xi}} dx$$

$$\frac{d\eta}{dy} = \frac{Ue}{\sqrt{2\xi}} e^{\frac{y}{\xi}} \quad ; \quad dy = \frac{d\eta}{\frac{Ue}{\sqrt{2\xi}} e^{\frac{y}{\xi}}} + \frac{dy}{d\xi} d\xi$$

y indep of $\xi(x)$

10-2299

so $dy = \frac{\sqrt{2\xi}}{Ue} d\eta$ but for flat plate $\xi = Ue^{\frac{x}{Ue}}$

$$dy = \sqrt{\frac{2Ue^{\frac{x}{Ue}}}{Ue^2}} d\eta = \frac{d\eta}{Ue} \sqrt{\frac{2Ue^{\frac{x}{Ue}}}{Ue}}$$

$$f^* = \int_0^\infty \left(1 - \frac{f'}{g}\right) \frac{1}{Ue} \sqrt{\frac{2Ue^{\frac{x}{Ue}}}{Ue^2}} d\eta(Ue)$$

$$\frac{f^*}{x} = \int_0^\infty \left(1 - \frac{f'}{g}\right) g \sqrt{\frac{2Ue}{Ue^2 x}} d\eta$$

$$\frac{f^*}{x} = \left[\sqrt{2} \int_0^\infty (g - f') d\eta \right] (Re_x)^{-1/2}$$

↑ integral depends only on Me, Te, Pr

given a viscosity law, Pr , and γ (perfect gas).

for Blasius, $M=0$, T doesn't matter, $I = 1.7208$
($g=1$)

for adiabatic wall conditions, Sutherland law,
 $Pr = 0.72$ (air), see following plots.

