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# FLOQUET SOLUTION FOR A SPINNING SYMMETRIC RIGID BODY WITH CONSTANT TRANSVERSE TORQUES

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In this paper we analyze the problem of large angular excursions of the spin axis of a rigid body using Floquet theory. This approach involves transforming the nonlinear equations into a linear periodic system and then computing solutions using Fourier series expansions. Numerical simulations confirm that the solutions are highly accurate when applied to typical spacecraft maneuvers.

## Introduction

Since Grammel<sup>1,2</sup> defined the problem of the self-excited rigid body, numerous investigators<sup>2-20</sup> have contributed approximate analytic solutions for its motion. The body is free to rotate about a point fixed in the body and inertial space under the action of a torque vector arising from internal reactions which do not appreciably alter the mass or mass distribution. The forced motion of a spacecraft due to thruster torques is a particularly relevant, modern example of the self-excited rigid body.

In the literature a number of simplifying assumptions are used to put the nonlinear differential equations involved into tractable form for analytic integration. In dealing with Euler's equations of motion most authors assume the body is axisymmetric (or nearly axisymmetric) and that the body-fixed torque components (which may act on up to three axes) are constants. To solve the associated kinematic differential equations, the usual approach is to use Eulerian angles and then to make small angle approximations (say on two of the angles) in order to obtain approximate, closed-form analytic solutions.

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Recently, interest has been stimulated in other attitude representations (see the excellent survey paper by Shuster<sup>21</sup>). A new parameterization developed by Tsiotras and Longuski<sup>22,23</sup> has been employed to find an approximate solution<sup>18</sup> for large angle motion of a symmetric or near-symmetric rigid body due to constant torque about three body axes. No exact solution is known (even for the axisymmetric case) of constant torque on three axes.

In this paper we show that the axisymmetric case of constant transverse torque (i.e. no axial torque) is amenable to Floquet theory<sup>24,25</sup> when Cayley-Klein parameters<sup>21,26</sup> are used for attitude representation. We show that the ensuing standard eigenvalue problem, solved numerically, can provide an arbitrarily accurate solution for all possible motion.

## Analytic Solutions

### Euler's Equations of Motion

The spin of a rigid body is controlled by Euler's equations:

$$\dot{\omega}_x = M_x/I_x - [(I_z - I_y)/I_x] \omega_y \omega_z \quad (1)$$

$$\dot{\omega}_y = M_y/I_y - [(I_x - I_z)/I_y] \omega_z \omega_x \quad (2)$$

$$\dot{\omega}_z = M_z/I_z - [(I_y - I_x)/I_z] \omega_x \omega_y \quad (3)$$

where  $M_x$ ,  $M_y$ , and  $M_z$  are torque components,  $\omega_x$ ,  $\omega_y$ , and  $\omega_z$  are angular velocity components, and  $I_x$ ,  $I_y$ , and  $I_z$  are principal moments of inertia. We assume that the applied torques are constant, and purely transverse ( $M_z = 0$ ). Such transverse torques

often appear in spacecraft thrusting maneuvers, due to center-of-mass offset and thruster misalignment. In addition we assume that the mass distribution is fixed (i.e., the case of the self-excited rigid body<sup>1,2</sup>), and that the body is near symmetric [ $(I_x - I_y)/I_z \ll 1$ ]. Thus the angular velocity about the  $z$ -axis will be essentially constant,

$$\dot{\omega}_z \approx 0 \quad (4)$$

which is exact if  $M_z \equiv 0$  and  $I_x \equiv I_y$ .

With this simplification, as discussed by Randall et al.,<sup>19</sup> Eqs. (1) and (2) reduce to a pair of linear, constant-coefficient, ordinary differential equations with constant forcing terms. The solution for  $(\omega_x, \omega_y)$  is a simple sine wave, which we can write in the compact form:

$$\frac{\omega}{\omega_z} = 2 \sum_{-1}^1 \omega_j E_j \quad (5)$$

where

$$\omega(t) = \omega_x(t) + i\omega_y(t) \quad (6)$$

$$E_j = e^{i j k \tau} \quad (7)$$

$$\tau = \omega_z t \quad (8)$$

The constant  $k$ , determined by the mass distribution, is the transverse mode frequency in units of  $\omega_z$ . It ranges between  $k = 0$  (a sphere) and  $k = 1$  (a flat disk). The three nondimensional constants,  $\omega_j$ , are determined by the applied torques and the initial conditions through the relations:

$$\kappa_x = \sqrt{(I_z - I_y)/I_x}, \quad \kappa_y = \sqrt{(I_z - I_x)/I_y} \quad (9)$$

$$k = \kappa_x \kappa_y \quad (10)$$

$$k_1 = (1/\kappa_x + 1/\kappa_y)/2 \quad (11)$$

$$k_2 = (1/\kappa_y - 1/\kappa_x)/2 \quad (12)$$

$$F = [i M_x/(I_x \kappa_x) - M_y/(I_y \kappa_y)]/\omega_z^2 \quad (13)$$

$$\Omega_0 = (\omega_x \kappa_y + i \omega_y \kappa_x)/\omega_z \quad \text{at } t = 0 \quad (14)$$

$$\omega_{-1} = k_2(\bar{\Omega}_0 - \bar{F})/2 \quad (15)$$

$$\omega_0 = (k_1 F + k_2 \bar{F})/2 \quad (16)$$

$$\omega_1 = k_1(\Omega_0 - F)/2 \quad (17)$$

It is important to keep in mind that this solution, in which  $\omega_x$  and  $\omega_y$  are sinusoidal, relies on the effective constancy of  $\omega_z$ . The main point of this paper is to

present a solution of the kinematic problem when this is true.

### Kinematic Equations

A classical method of expressing the attitude motion of a rigid body is to use a Type 1: 3-2-1 Euler angle sequence<sup>27</sup>. The corresponding kinematic equations are

$$\dot{\phi}_x = \omega_x + (\omega_y \sin \phi_x + \omega_z \cos \phi_x) \tan \phi_y \quad (18)$$

$$\dot{\phi}_y = \omega_y \cos \phi_x - \omega_z \sin \phi_x \quad (19)$$

$$\dot{\phi}_z = (\omega_y \sin \phi_x + \omega_z \cos \phi_x) \sec \phi_y \quad (20)$$

where  $\phi_x$ ,  $\phi_y$ , and  $\phi_z$  are the Eulerian angles. These equations are highly nonlinear, and seemingly intractable for analytical solution, although much progress has been made using linearization, e.g. by assuming  $\phi_x$  and  $\phi_y$  are small.<sup>11,14,16,19</sup>

An alternative and, for our purposes, preferable representation of the kinematics is the Cayley-Klein parameters,<sup>21,26</sup>  $[\alpha, \beta]$ , which are defined in terms of the Euler angles by

$$\alpha = e^{i\phi_z/2} [\cos(\frac{\phi_x}{2}) \cos(\frac{\phi_y}{2}) - i \sin(\frac{\phi_x}{2}) \sin(\frac{\phi_y}{2})] \quad (21)$$

$$\beta = e^{i\phi_z/2} [\cos(\frac{\phi_x}{2}) \sin(\frac{\phi_y}{2}) - i \sin(\frac{\phi_x}{2}) \cos(\frac{\phi_y}{2})] \quad (22)$$

These two complex numbers obey the normalization  $|\alpha|^2 + |\beta|^2 = 1$  as is easily confirmed from their definition.

The inverse relation, giving Euler angles in terms of Cayley-Klein parameters is:

$$\phi_x = \tan^{-1} \left[ \frac{2 \operatorname{Im}(\alpha \bar{\beta})}{|\alpha|^2 - |\beta|^2} \right] \quad (23)$$

$$\phi_y = \sin^{-1} [2 \operatorname{Re}(\alpha \bar{\beta})] \quad (24)$$

$$\phi_z = \tan^{-1} \left[ \frac{\operatorname{Im}(\alpha^2 - \beta^2)}{\operatorname{Re}(\alpha^2 - \beta^2)} \right] \quad (25)$$

The advantage of the Cayley-Klein representation is that  $[\alpha, \beta]$  obey linear differential equations, in sharp contrast to Eqs. (18)-(20):

$$\dot{\alpha} = \frac{i\omega_z}{2} \alpha - \frac{i\bar{\omega}}{2} \beta \quad (26)$$

$$\dot{\beta} = -\frac{i\omega}{2} \alpha - \frac{i\omega_z}{2} \beta \quad (27)$$

This fact allows us to use the principle of linear superposition to construct general solutions for arbitrary initial conditions. Moreover, with the approximation  $\dot{\omega}_z \approx 0$ , the coefficients  $\omega(t)$  in Eqs. (26)-(27) are periodic, so that the fundamental solutions can be developed using Floquet theory, even for very large angular displacements.

### Floquet Formulation and Solution of Kinematic Equations

We seek the general solution of Eqs. (26) and (27) for the Cayley-Klein parameters. These equations are second order, linear, and homogeneous, so there are two independent solutions. Also, by inspection, they have the symmetry that if  $[\alpha, \beta]$  is a solution, then so is  $[\beta, -\bar{\alpha}]$ . Hence the general solution must have the form:

$$\alpha = C_1 \alpha_1 + C_2 \bar{\beta}_1 \quad (28)$$

$$\beta = C_1 \beta_1 - C_2 \bar{\alpha}_1 \quad (29)$$

where  $[\alpha_1, \beta_1]$  is any solution pair.

Finally, as seen in Eq. (5), the coefficients are periodic in  $\tau$  with period  $T = 2\pi/k$ , so that Floquet theory<sup>24,25</sup> applies. The essence of Floquet theory is that there will be solutions of the form

$$\alpha = e^{-is\tau} u(\tau) \quad (30)$$

$$\beta = e^{-is\tau} v(\tau) \quad (31)$$

where  $u$  and  $v$  will be periodic with period  $T$  provided that  $s$  is suitably chosen.

It follows that the general solution of Eqs. (26) and (27) can be written as:

$$\alpha = C_1 e^{-is\tau} u + C_2 e^{is\tau} \bar{v} \quad (32)$$

$$\beta = C_1 e^{-is\tau} v - C_2 e^{is\tau} \bar{u} \quad (33)$$

where  $C_1, C_2$  are determined by the initial conditions, and where  $[u, v]$  are any pair of solutions of the differential equations:

$$\frac{du}{d\tau} - i\left(s + \frac{1}{2}\right)u = -\frac{i\bar{\omega}}{2\omega_z}v \quad (34)$$

$$\frac{dv}{d\tau} - i\left(s - \frac{1}{2}\right)v = -\frac{i\omega}{2\omega_z}u \quad (35)$$

These equations have the symmetry that if  $[u, v, s]$  is a solution then so is  $[\bar{v}, -\bar{u}, -s]$ .

Because  $[u, v]$  are periodic, they can be represented by Fourier series:

$$u = \sum_{-\infty}^{\infty} u_j E_j, \quad v = \sum_{-\infty}^{\infty} v_j E_j \quad (36)$$

Substituting this expansion into Eqs. (34) and (35), we get a set of recurrence relations which determine the Fourier coefficients  $[u_j, v_j]$  and the eigenvalue  $s$ :

$$s u_j = \left(jk - \frac{1}{2}\right) u_j + (\bar{\omega}_{-1} v_{j-1} + \bar{\omega}_0 v_j + \bar{\omega}_1 v_{j+1}) \quad (37)$$

$$s v_j = \left(jk + \frac{1}{2}\right) v_j + (\omega_{-1} u_{j+1} + \omega_0 u_j + \omega_1 u_{j-1}) \quad (38)$$

These relations can be arranged in the form :

$$s U = A U \quad (39)$$

where  $U = [\dots u_j, v_j, \dots]$  and  $A$  is an infinite dimensional, pentadiagonal matrix:

$$A = \text{diag} (D_{-3}, D_{-1}, D_0, D_1, D_3) \quad (40)$$

where  $D_0$  is the main diagonal,  $D_1$  the first super-diagonal,  $D_{-1}$  the first sub-diagonal, etc. The elements of these diagonals are:

$$D_0 = [\dots (jk - 1/2, jk + 1/2) \dots] \quad (41)$$

$$D_{-1} = \bar{D}_1 = [\dots (\omega_0, \bar{\omega}_{-1}) \dots] \quad (42)$$

$$D_{-3} = \bar{D}_3 = [\dots (\omega_1, 0) \dots] \quad (43)$$

It is easily seen that  $A$  is Hermitian, so that the eigenvalues  $s$  must be real. Moreover, we can show, (most easily from Eqs. (37) and (38)) that if  $s_0$  is an eigenvalue, then so is  $\pm s_0 + Nk$  where  $N$  is any integer. So, although there are an infinite number of eigenvalues of this infinite dimensional matrix  $A$ , there is, in fact only one which is physically distinct. (The other eigenvalues and eigenvectors arise from a trivial renumbering of the Fourier modes.)

In practice, only a finite number of terms,  $j = [-M, M]$ , can be retained in the series, Eq. (36). When this is done, the  $U$  vector will be of length  $4M + 2$  and the matrix  $A$  will be square of the same size. For example, the smallest such truncation,  $M = 1$ , yields the  $6 \times 6$  matrix:

$$A = \begin{bmatrix} -k - 1/2 & \bar{\omega}_0 & 0 & \bar{\omega}_1 & 0 & 0 \\ \omega_0 & -k + 1/2 & \omega_{-1} & 0 & 0 & 0 \\ 0 & \bar{\omega}_{-1} & -1/2 & \bar{\omega}_0 & 0 & \bar{\omega}_1 \\ \omega_1 & 0 & \omega_0 & +1/2 & \omega_{-1} & 0 \\ 0 & 0 & 0 & \bar{\omega}_{-1} & k - 1/2 & \bar{\omega}_0 \\ 0 & 0 & \omega_1 & 0 & \omega_0 & k + 1/2 \end{bmatrix}$$

The truncated matrix will have  $2M+1$  equal and opposite pairs of real eigenvalues, but because the truncation breaks the translational symmetry, the eigenvalues will not be precisely related by  $\pm s_0 + Nk$ . For any given  $M$ , some of the  $4M+2$  eigenvalues will be more accurate than others.

The key questions are: how big must  $M$  be to achieve a given accuracy, and how can the most accurate eigenvalue be selected? These questions will be answered in more detail in a later section, after we have looked at some numerical results. However, we can now give a rough estimate of the how big  $M$  needs to be.

For very large  $j$ , the Fourier coefficients must decrease. Assuming that  $|v_{j+1}| \ll |v_j| \ll |v_{j-1}|$ , we can easily deduce from the recurrence relations, Eqs. (37) and (38), that the ratio of alternating  $v$  coefficients, as  $j \rightarrow \infty$ , is:

$$\frac{v_j}{v_{j-2}} \approx \frac{\omega_1 \bar{\omega}_{-1}}{(jk)^2} \quad (44)$$

with similar expressions for  $u$  and for  $j \rightarrow -\infty$ . This demonstrates two important properties:

- 1) The Fourier coefficients decay superexponentially for large  $j$ .
- 2) We must have  $j > \nu/k$  before the coefficients begin to decay, where  $\nu = (\sum |\omega_j|^2)^{1/2}$  or some other norm of the Fourier coefficients  $\omega_j$ .

The first property says the series will converge rapidly, so that not many terms will be needed. The second property gives us a lower bound on a reasonable truncation level:

$$M \geq 1 + \frac{\nu}{k} \quad (45)$$

Evidently, when  $\nu \ll 1$  only a very few terms will be needed.

### Small Torque Approximation

When the applied torque is small enough so that  $\nu \ll 1$ , the matrix  $A$  is essentially diagonal and we can derive a simple approximate solution of Eq. (39). The result is that the Fourier coefficients form an asymptotic sequence in powers of  $\nu$ . The coefficients for  $|j| > 1$  are  $\mathcal{O}(\nu^3)$  or smaller, so that to get  $\mathcal{O}(\nu^2)$  accuracy we only need to compute the terms for  $j = [-1, 0, 1]$ . This can be done recursively starting with the scaling assumption  $v_0 = 1$ , with the result:

$$j = [-1, 0, 1] \quad (46)$$

$$u_j \approx \left[ -\frac{\bar{\omega}_1}{1+k}, \bar{\omega}_0, -\frac{\bar{\omega}_{-1}}{1-k} \right] \quad (47)$$

$$v_j \approx \left[ \frac{1}{k} \left( \frac{\omega_0 \bar{\omega}_1}{1+k} + \bar{\omega}_0 \omega_{-1} \right), 1, -\frac{1}{k} \left( \bar{\omega}_0 \omega_1 + \frac{\omega_0 \bar{\omega}_{-1}}{1-k} \right) \right] \quad (48)$$

At this level of approximation  $[u, v]$  are simple harmonic. The eigenvalue,  $s$ , to the same order, is given by:

$$s \approx \frac{1}{2} \left[ 1 + 4 \sum_{-1}^1 \frac{|\omega_j|^2}{1+jk} \right]^{1/2} \quad (49)$$

It is evident that this solution fails when  $k = 0$  (sphere) and  $k = 1$  (plate), regardless of how small  $\nu$  is. When  $k$  is close to either extreme, the ordering of the coefficients changes, so that the  $j = 2$  terms may be as large as the  $j = 1$  terms.

The approximate solution given here is asymptotically equivalent to an  $M = 1$  truncation of Eq. (39), but is algebraically simpler.

## Numerical Results

### Test Case

In order to test the Floquet solution we consider a Galileo-like spacecraft maneuver. We will use an axial thrusting maneuver discussed by Longuski et al.<sup>13</sup>. We consider the symmetric case where  $I_x = I_y$  with the following mass properties (similar to the Galileo):

$$I_x = I_y = 3012 \text{ kg-m}^2, \quad I_z = 4627 \text{ kg-m}^2 \quad (50)$$

and initial conditions:

$$\omega_x(0) = \omega_y(0) = 0, \quad \omega_z(0) = 0.33 \text{ rad/s} \quad (51)$$

$$\phi_x(0) = \phi_y(0) = \phi_z(0) = 0 \quad (52)$$

Transverse torque can arise from a center-of-mass offset of the main engine. Here we select a very large transverse torque (about 150 times that of the Galileo) in order to demonstrate the theory for an extreme case where the Euler angles  $(\phi_x, \phi_y)$  approach  $90^\circ$ :

$$M_x = 225 \text{ Nm}, \quad M_y = 0, \quad M_z = 0 \quad (53)$$

For this test case we note that the inertia parameter,  $k = 0.5362$  and Eqs. (15)-(17) yield the following constant values

$$\omega_{-1} = 0, \quad \omega_0 = 0.6397i, \quad \omega_1 = -0.6397i \quad (54)$$

### Baseline Numerical Integration

Since we expect the Floquet solution to be highly accurate, we need a very precise method to test it. We employ an adaptive Runge-Kutta fourth/fifth order integration method using double precision accuracy for the following simulations. In each case the accuracy is controlled by a relative tolerance of  $1 \times 10^{-12}$  and an absolute tolerance of  $1 \times 10^{-14}$ . For our baseline numerical integration we integrate Eqs. (1)-(3) and (18)-(20). The errors in the baseline numerical integration are on the order of  $10^{-12}$  rad for the Euler angles.

### Discussion

The discussion which follows pertains to the test case, Eqs. (50)-(52), with

$$M = 7 \quad (55)$$

From Floquet theory we know that the parameters  $u$  and  $v$  are periodic with period  $T = 2\pi/k$ . In Fig. 1 we plot the real (solid line) and the imaginary (dashed line) parts of the solution for Eqs. (34) and (35) for one period.

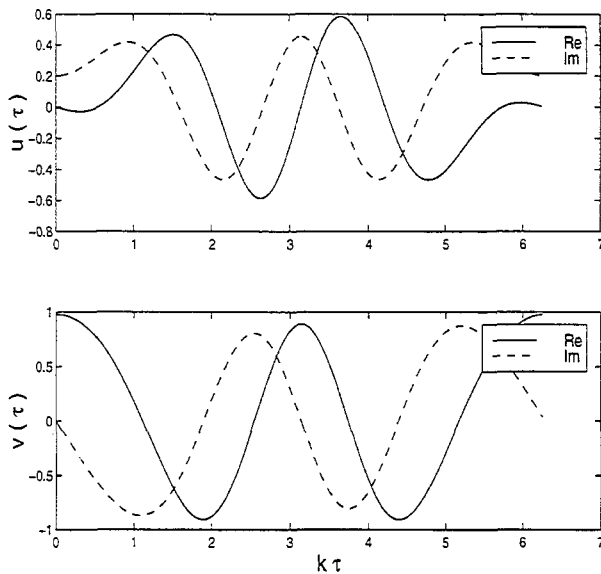


Fig. 1: One period of  $u$  and  $v$  for test case.

The Cayley-Klein parameters,  $\alpha$  and  $\beta$ , obtained using Eqs. (32) and (33) are not periodic in general. The real and the imaginary parts of  $\alpha$  and  $\beta$  are plotted in Fig. 2 for two periods ( $2T/\omega_z$ ). We note that the plots satisfy the normalization constraint that  $|\alpha|^2 + |\beta|^2 = 1$ .

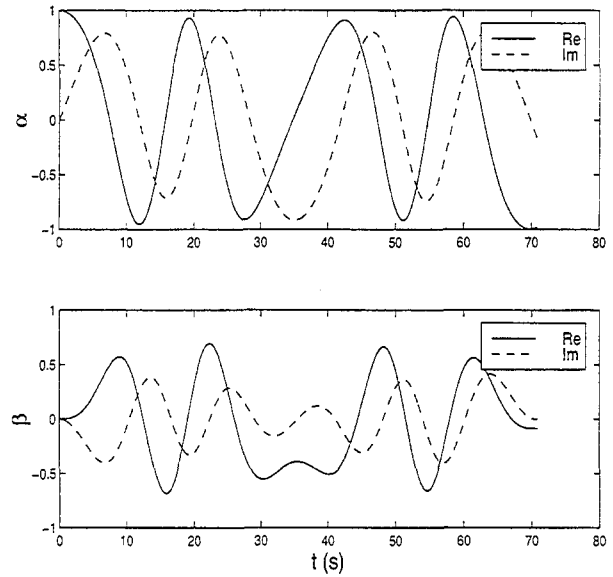
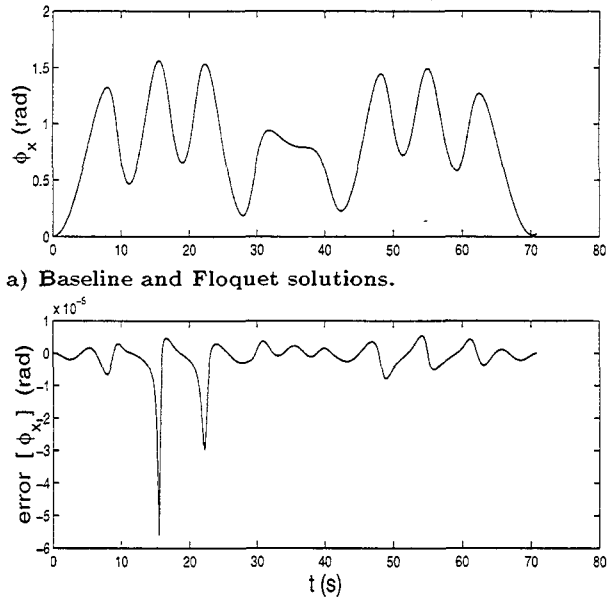


Fig. 2: Cayley-Klein functions  $\alpha$  and  $\beta$  for test case.



a) Baseline and Floquet solutions.

b) Baseline minus Floquet solution.

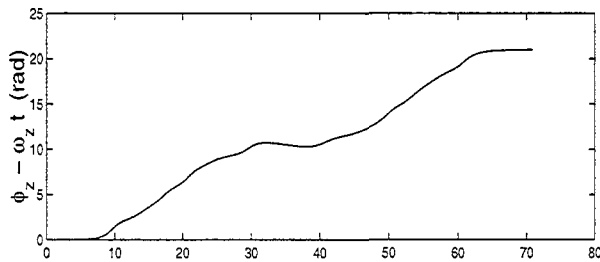
Fig. 3: Euler angle  $\phi_x$  for test case.

Fig. 3a shows two solutions for the Euler angle,  $\phi_x$ : the Floquet solution (obtained using the solutions for  $\alpha$  and  $\beta$  and Eq. (23)) and the baseline numerical integration. Because the results are indistinguishable at this scale, we show the difference between the results in Fig. 3b. The maximum error is about  $5.5 \times 10^{-5}$  rad out of 1.6 rad (at  $t = 16$  s) or about 0.003 %.

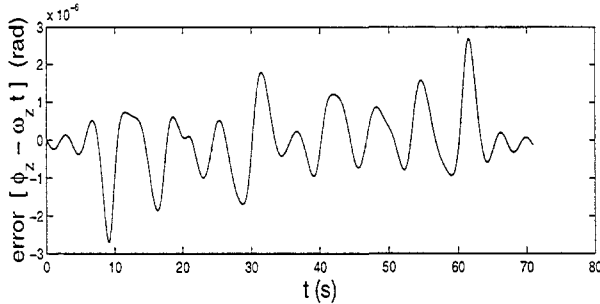
A similar comparison for the spin angle,  $\phi_z$ , is shown in Fig. 4. We note that  $\phi_z$  is very large for large values of time. For convenience we introduce a smaller angle  $\Phi_z$ , obtained by subtracting off the linearized solution for  $\phi_z$ :

$$\Phi_z = \phi_z - \omega_z t \quad (56)$$

In Fig. 4a we show the baseline solution for the angle  $\Phi_z$  and the Floquet solution obtained using  $\alpha$  and  $\beta$  and Eqs. (25) and (56). To check the accuracy, we again show the difference between the two solutions. Here we see the error is about  $3 \times 10^{-6}$  rad out of 1 rad (at  $t = 9$  s) or 0.0003%. Since the errors in the baseline solution are  $\mathcal{O}(10^{-12})$ , the difference represents the true error in the Floquet solution.



a) Baseline and Floquet solutions.



b) Baseline minus Floquet solution.

Fig. 4: Angle  $\Phi_z = \phi_z - \omega_z t$  for test case.

### Accuracy Control

In order to study the effects of  $M$  on the accuracy of the Floquet solution, we conduct the following parametric study. We first fix a value for  $M$ . We then compute the baseline numerical integration and Floquet solutions for  $\phi_x$  using transverse torque  $M_x$  in the range of 0 to 225 Nm. For each of these trajectories, we compute the difference between the two solutions. In Fig. 5 we plot percent error versus the maximum absolute value,  $|\phi_x|_{max}$ . As expected, the error increases as the angle increases; larger values of  $M$  result in smaller errors. We note that the

plateau at the bottom of the figure occurs due to the errors in the baseline numerical integration and not the Floquet solution. Also as  $\phi_x$  approaches  $90^\circ$ , the error increases rapidly due to the well-known Euler angle singularity.

A similar study is conducted for the angle  $\Phi_z$ . In Fig. 6, percent error versus the maximum absolute value,  $|\Phi_z|_{max}$  is plotted. Again, the error increases as the angle increases and larger values of  $M$  result in smaller errors. A similar numerical integration plateau occurs.

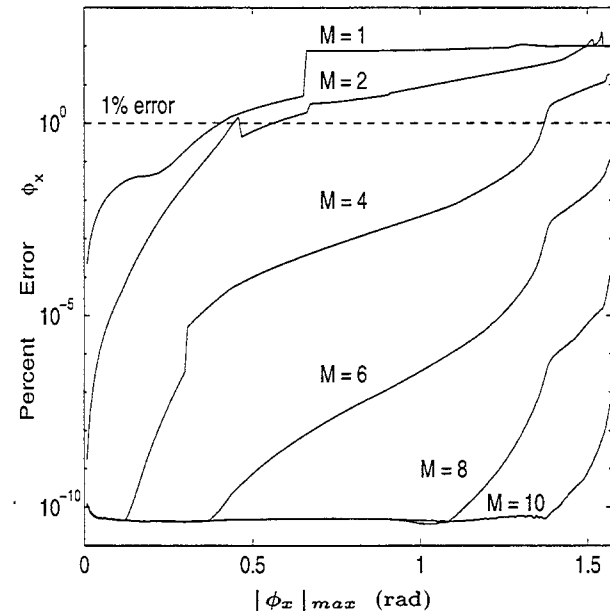


Fig. 5: Percent error in maximum absolute value of  $\phi_x$  for various transverse torques (symmetric case).

Figures 5 and 6 can be used to choose  $M$  to achieve a given accuracy in the Floquet solution, but only for the test case. For the general case it would be useful to have a method which selects  $M$  for a desired accuracy.

### Eigenvector Selection

At any given truncation level,  $M$ , there are  $2M + 1$  distinct values of  $s^2$  which arise from solving Eq. (39). Some of these values will be better than others, so we must sort the wheat from the chaff. The essential idea is that since we centered the truncation about  $j = 0$ , then those eigenvectors which are most nearly centered about  $j = 0$  should be most accurate. We illustrate this in Figs. 7 and 8, which show two of the 30 eigenvector spectra for

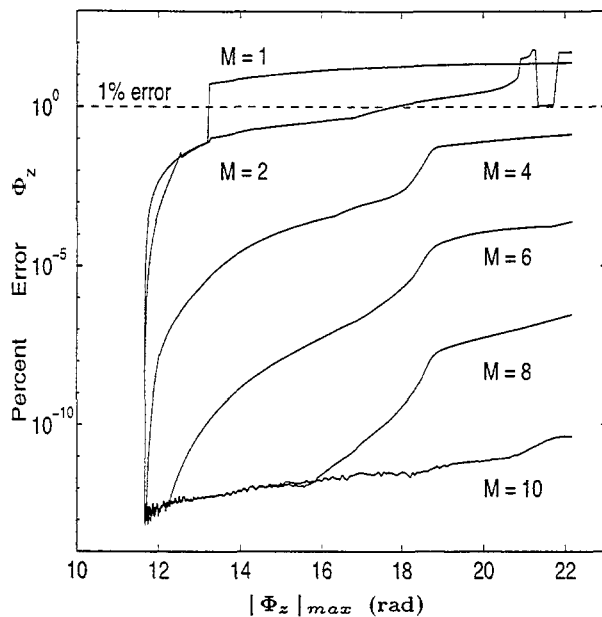


Fig. 6: Percent error in maximum absolute value of  $\Phi_z$  for for various transverse torques (symmetric case).

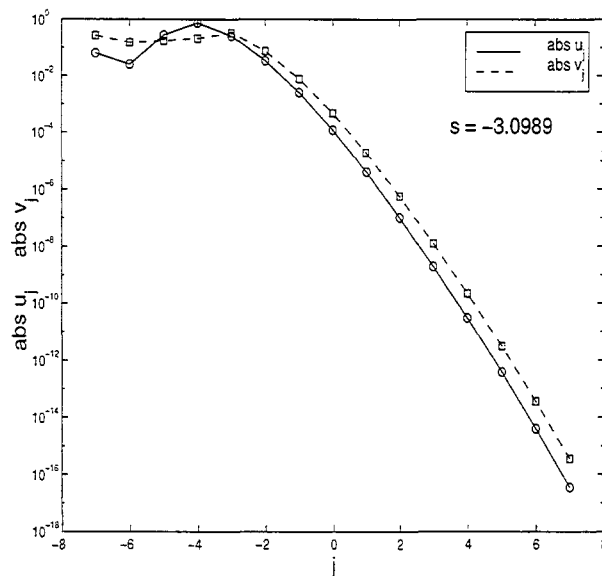


Fig. 7: A suboptimal eigenvalue/eigenvector selection of  $M_x = 225 \text{ Nm}$ ,  $M = 7$ .

an  $M = 7$  truncation of the test case. In Fig. 7, the peak occurs near the left edge of the window, so the neglected terms in  $j < -7$  are not small and the solution is poor. In Fig. 8, the peak is near the middle of the window, and the neglected terms on both the left and right ( $|j| > 7$ ), are clearly less than  $10^{-5}$  in magnitude. This is the best we can do with  $M = 7$ .

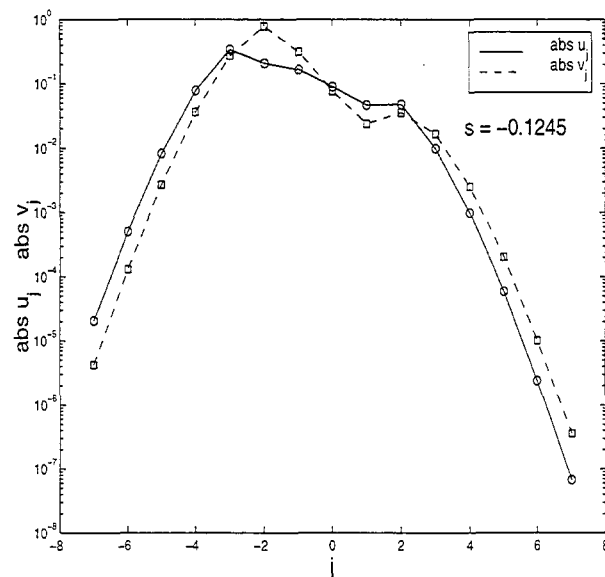


Fig. 8: Optimal eigenvalue/eigenvector selection of  $M_x = 225 \text{ Nm}$ ,  $M = 7$ .

The above selection process can be automated by measuring the error in a spectrum,  $\epsilon$ , from the size of its end elements:

$$\epsilon = |u_{-M}| + |v_{-M}| + |u_M| + |v_M| \quad (57)$$

There will be  $4M + 2$  values of  $\epsilon$ ; the best solution is the one for which  $\epsilon$  is minimum. For the test case with  $M = 7$ , this optimal solution is shown in Fig. 8. It is worth noting that the poor result in Fig. 7 corresponds to  $s + 6k = 0.1183$ , which is 5% off the target of  $s = 0.1245$ . The occurrence of errors of this order is to be expected from the size of the  $j = -7$  Fourier coefficients (about  $10^{-1}$ ) seen at the left in Fig. 7.

### Automatic Error Control

Having shown how to select the best eigensolution at a given truncation, we can easily see how to automatically select  $M$  to give any specified accuracy in the solution:

- 1) Pick a tolerance  $\epsilon_{max}$  and the smallest reasonable truncation :  $M = 1 + \nu / k$ .
- 2) Solve the truncated eigenproblem, selecting that vector with minimum error,  $\epsilon_{min}$ .
- 3) If  $\epsilon_{min} < \epsilon_{max}$ , stop.  
If  $\epsilon_{min} > \epsilon_{max}$ , increase  $M$ ; repeat 2) & 3).

The assumption made in this algorithm is that the errors in the solution (for  $u$  and  $v$ ) are smaller than the last retained Fourier coefficients.

It is, naturally, wasteful to compute all  $4M + 2$  eigensolutions when most of them are thrown away. For this reason, a practical approach is to use an iterative eigensolver which computes only a few of the eigenvalues closest to the previous optimum. On the first step, the center eigenvalue is set at  $s = 1/2$ , based on the small-torque solution, Eq. (49).

**Results for a Near-Symmetric Case**

It is only natural to be curious as to what happens when the aforementioned Floquet solution is applied to a more realistic case. We choose the following near-symmetric mass properties:

$$\begin{aligned} I_x &= 3012 \text{ kg}\cdot\text{m}^2, & I_y &= 2761 \text{ kg}\cdot\text{m}^2 \\ I_z &= 4627 \text{ kg}\cdot\text{m}^2 \end{aligned} \tag{58}$$

with a moderately large transverse torque

$$M_x = 100 \text{ Nm}, \quad M_y = 0, \quad M_z = 0 \tag{59}$$

and use the same initial conditions as the previous test case. Here we note that the inertia parameter,  $k$ , is slightly larger with a value of

$$k = 0.6020 \tag{60}$$

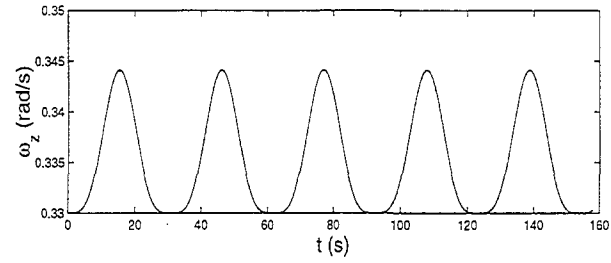
and Eqs. (15)-(17) yield the following smaller  $\omega_j$  values

$$\omega_{-1} = 0.0036 i, \quad \omega_0 = 0.2461 i, \quad \omega_1 = -0.2496 i \tag{61}$$

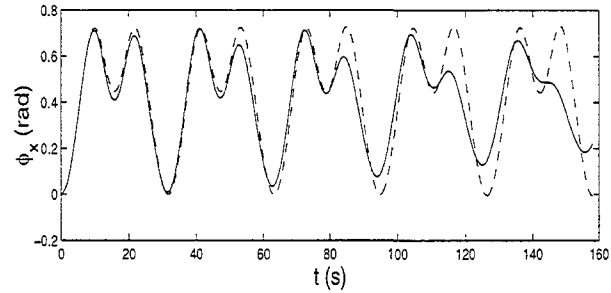
We see in Fig. 9 that  $\omega_z$  is periodic with a small amplitude fluctuation of 5%, not constant as assumed from Eq.(4). Figures 10 and 11 show the results for  $\phi_x$  and  $\Phi_z$  respectively. We notice that the Floquet solution seems to track reasonably well for a while, then diverges from the baseline solution. However, the accuracy is significantly poorer (than the symmetric test case), even in the first oscillation. We know that this error is not due to truncation, since varying  $M$  from 2 to 10 makes no difference. The reason for the inaccuracy lies in the fact that  $\omega_z$  is not constant. It is possible to improve the solution by including perturbations to  $\omega_z$  due to the neglected term in Eq. (3). We expect these additional terms to be periodic, however the analysis is beyond the scope of the current paper.

**Conclusions**

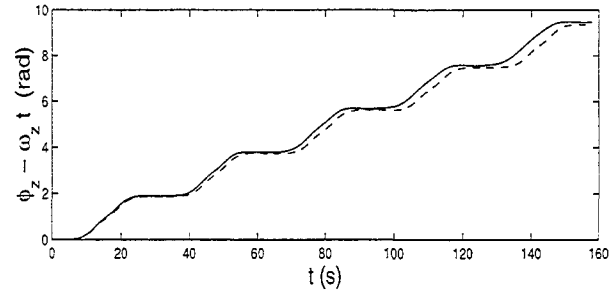
In this paper we considered the problem of a spinning symmetric spacecraft subject to large constant transverse torques. The Floquet solution presented



**Fig. 9: Angular velocity  $\omega_z$  for near-symmetric case.**



**Fig. 10: Euler angle  $\phi_x$  for near-symmetric case.**



**Fig. 11: Angle  $\Phi_z = \phi_z - \omega_z t$  for near-symmetric case.**

here, based on a Cayley-Klein formulation of the kinematic equations, is much more accurate and efficient than any previously found linear solutions, even when the angular excursion of the spin axis is large. The major assumption is that the spin rate is constant. This method is highly accurate for the symmetric case; however, when the theory is applied to the near-symmetric case, the error will be driven by the variation in  $\omega_z$ . This solution may find applications in onboard computations of spacecraft maneuvers and in maneuver analysis and optimization.

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