

# Analytical Modeling of Low-Loss Disk Flexure Resonators

Jonathan M. Puder  
OxideMEMS Lab  
Cornell University  
Ithaca, USA

Jeffrey S. Pulskamp, Ryan Q. Rudy, and  
Ronald G. Polcawich  
Sensors and Electron Devices Directorate  
US Army Research Laboratory  
Adelphi, USA

Sunil A. Bhawe  
OxideMEMS Lab  
Purdue University  
West Lafayette, IN

**Abstract**—This paper presents for the first time a closed form analytical solution for the motional resistance,  $R_m$ , of the transverse (1,1) mode of two port disk flexure resonators (DFRs), enabling rapid performance prediction and device optimization. This work is motivated by a recently reported -1 dB peak  $S_{21}$  and 9  $\Omega$  motional resistance DFR that has demonstrated the viability of non-traditional flexure based modes as high-performance, low frequency MEMS resonators. Modeled  $R_m$  values are validated using on-wafer extracted  $e_{31}$  piezoelectric stress constants, measured layer thicknesses, independently measured elastic moduli, and analytical mode shapes

**Keywords**— *Motional resistance, piezoelectricity, resonator*

## I. INTRODUCTION

Military communications systems, such as SINCGARS, continue to rely on sub-100 MHz filters, and require IF filters with narrow bandwidth, exceptional stop-band rejection, and frequency trimming capabilities. While early literature focused on flexure-based devices to meet these needs, in recent years research efforts have moved towards other modes. However, recently published resonator performance of -1 dB peak  $S_{21}$  and 9  $\Omega$  motional resistance ( $R_m$ ) for six parallel (1,1) PZT-on-silicon [1,2,3,4] disk flexure resonators terminated directly to 50  $\Omega$  highlights the potential of flexure-based devices as high-performance resonators at low frequencies [5]. To provide insight into the performance of these modes, this work will present an analytical expression for  $R_m$ , an important modeling parameter. This analytical expression is validated by fitting only the mechanical quality factor ( $Q_m$ ), and using independently measured or extracted parameters, including  $e_{31}$ .

## II. THEORY

### A. Motional Resistance

Modal analysis was used to model the (1,1) mode (Fig. 1) of the continuous disk as a driven, lumped mass-spring system by comparing the elastic energies of the two systems as well as the work done by the modal force and piezoelectric stress. Analytical mode shapes of the (1,1) mode of disk flexure consisting of modified and ordinary Bessel functions of the first kind [6], were used to obtain modal stresses and strains. The modal force and equivalent stiffness were used to derive a quasi-static displacement. Then, using extracted mechanical quality

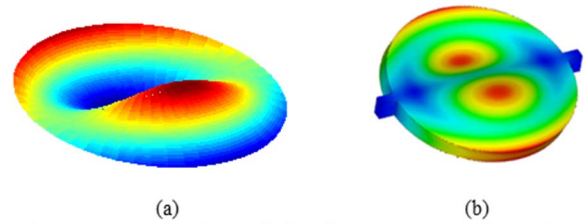


Fig. 1. The mode shape of disk flexure resonator from (a) FEA Eigen frequency analysis (b) analytical mode shapes expressions.

factor ( $Q_m$ ), the frequency response of the predicted real displacement was calculated. Displacement was converted to a frequency- and time-dependent charge on the output port via the direct piezoelectric effect. The application of a time derivative and rearrangement returns the closed-form expression for  $R_m$  in Equation (1),

$$R_m = \frac{4\pi\gamma\lambda^2 t_{tot}^2 \sqrt{Y_c \rho_c}}{Q_m \sqrt{3(1-\nu^2)} \left( k e_{31} h \int_{A_{el}} \beta r dr d\theta \right)^2} \quad (1)$$

$\beta$  in (1) is given by,

$$\beta = \cos(\theta) (J_1(kr) - I_1(kr)) \quad (2)$$

where  $\gamma$  is the ratio of modal mass to total mass,  $\lambda$  is a frequency constant,  $k$  is the wavenumber,  $t_{tot}$  is the total disk thickness,  $Y_c$  and  $\rho_c$  are the composite elastic moduli and density,  $\nu$  is Poisson's ratio,  $h$  is the distance from the neutral axis to the mid-plane of the piezoelectric layer, and  $J_1$  and  $I_1$  are the ordinary and modified Bessel functions of the first kind [6]. The model predicts high coupling due to energy transduction through both the  $e_{31}$  and  $e_{32}$  ( $e_{31}=e_{32}$  for PZT and AlN) constants, and motional resistance to be independent of radius.

### B. Comparison to Other Low Frequency Modes

To understand the advantages of disk resonators, they are compared and contrasted with two other common modes found at low frequencies. Beam resonators are often designed to excite length extension (LE) modes, and have spurious modes of beam

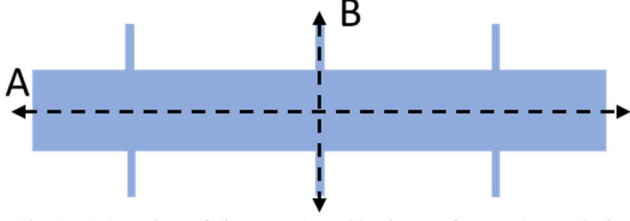


Fig. 2. A top view of the resonator with planes of symmetry marked. Rectangular electrodes are either symmetric about A or symmetric/antisymmetric about B.

flexure (BF). Equations for the motional resistance of these modes are given by [7]

$$R_{m_{Beam\ Flexure}} = \frac{\sqrt{Y_c I \rho_c w_{tot} t_{tot}}}{4LQ_m \left[ e_{31} h \int_L \frac{d^2 u_n}{dx^2} w_{el}(x) dx \right]^2} \quad (3)$$

$$R_{m_{Beam\ Extension}} = \frac{n\pi w_{tot} t_{tot} \sqrt{Y_c \rho_c}}{2Q_m \left[ e_{31} \int_L \frac{du_n}{dx} w_{el}(x) dx \right]^2} \quad (4)$$

Where the  $w_{tot}$  is the total width of the resonator,  $I$  is the area moment of inertia,  $u_n$  is the unity normalized mode shape of the  $n^{th}$  mode,  $L$  is the length of the resonator, and  $w_{el}$  is the width of the rectangular electrodes. For these expressions, the electrodes are assumed to symmetric or anti symmetric about the planes shown in the schematic of Fig. 2.

Comparing first to beam flexure, both modes are inversely proportional to  $h^2$  and  $e_{31}^2$ . However, the  $R_m$  of BF modes scale linearly with  $L$  and inversely with  $w_{tot}$ , while the disk is not expected to scale with  $R$ . However, if  $L$  and  $w_{tot}$  are changed at the same rate, then the BF  $R_m$  is not affected. This approximate relationship would occur as a device is scaled to higher frequencies. Therefore, both modes will have consistent  $R_m$  across  $R$  and  $L/w_{tot}$  ratios. Since  $R$  and  $L$  are the primary frequency determining dimension in both modes, consistent  $R_m$  across a wide range of frequencies should be expected, if  $t_{tot}$  is held constant and the  $L/w_{tot}$  ratio is maintained. This principle is confirmed in [8], where parameterized simulations of disk resonators with varied widths and silicon thicknesses are run. For both modes, a thicker device with the piezoelectric material far from the neutral axis will result in lower  $R_m$  due the scaling with  $h$ , assuming that  $Q_m$  is not affected by layer thicknesses. However, very thick devices will violate the assumptions used to derive the analytical mode shapes and invalidate the derived  $R_m$  expressions. Finally, for both modes, it is possible to have infinite  $R_m$  if the range of axis and midplane of the piezoelectric layer are coincident, since  $h$  would be zero. The  $R_m$  of BF and DFRs show very similar behavior, which is expected due to both modes being characterized by out of plane flexure.

Comparing DFR  $R_m$  and LE  $R_m$ , first it is apparent that the LE mode cannot scale with  $h$  since this parameter is only defined for flexure resonators. Additionally, the  $R_m$  for LE modes can never be infinite in a non-trivial case. The LE mode  $R_m$  is expected to scale directly with  $t_{tot}$ , while that of the DFR will not have such a clear relation, since  $h$  and  $t_{tot}$  share a complex relationship. The LE mode  $R_m$  is not expected to scale with the

primary frequency determining dimension,  $L$ , nor is the DFR  $R_m$ . However, the LE  $R_m$  does scale inversely with the other in plane dimension,  $w_{el}$ . Therefore, by maintaining a constant width and changing the length, devices with the same  $R_m$  may be fabricated across a wide range of frequencies for LE resonators.

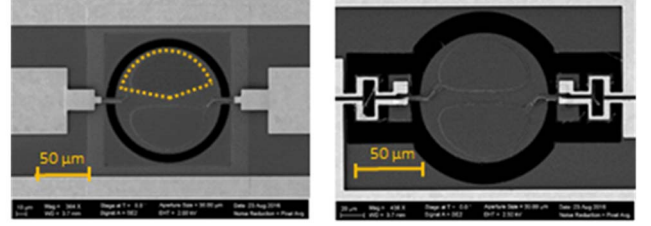


Fig. 3. Micrographs of fabricated devices with both types of anchors. The radius of the disks was 56  $\mu\text{m}$ . The approximated electrode shape used in modeling is traced over Fig. 3a.

### III. EXPERIMENTAL VALIDATION

#### A. Device Details

To validate the derived  $R_m$  of the DFRs, devices were fabricated in a PZT on silicon stack [1]. The devices consisted of a 1  $\mu\text{m}$  buried  $\text{SiO}_2$  layer, a 10  $\mu\text{m}$  silicon device layer, 300 nm of  $\text{SiO}_2$ , 125 nm of platinum for the bottom electrode, 0.5  $\mu\text{m}$  of PZT, and 50 nm of platinum for the top electrode. The disks were designed to have a 56  $\mu\text{m}$  radius. Additionally, the disks were fabricated with two types of anchors, which may be seen in Fig 3.

#### B. $e_{31}$ Piezoelectric Coefficients

Cantilever test structures were used to independently extract the electric-field dependent piezoelectric  $e_{31}$  constants under quasi-static conditions [9,10,11]. This field dependence arises directly from the fact that PZT is a ferroelectric material. At 0.5  $\mu\text{m}$  PZT film thickness, the coercive voltage of PZT is less than 2 V. This allows for more rigorous model validation at various operating points due to the fact that the piezoelectric stress constants of PZT can vary between 3-10 C/m<sup>2</sup>. The  $e_{31}$  were found to vary widely across the wafer, and three examples of extracted coefficients may be seen in Fig. 4.

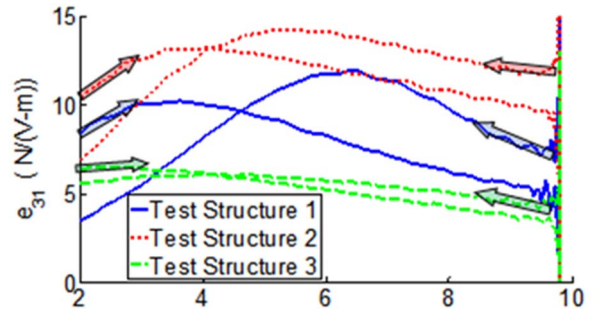


Fig. 4. The  $e_{31}$  constants extracted from cantilever test structures. The widely varying cross-wafer values partially explain the variation seen in  $R_m$ . The discontinuity at 10V is due to data processing artifacts. Arrows indicate the direction of the unipolar voltage sweep start from 0V up to 10 V and returning to 0V.

The coefficients were extracted using a 90 Hz sine wave with minimum and maximum values of 0 and 10 V, respectively.

Actual devices are operated in a small-AC-on-large-DC signal condition at much higher frequencies. This discrepancy in testing conditions likely contributed to agreement discrepancies in subsequent  $R_m$  validation.

### C. Measurement and Model Comparison

The scattering parameters of the PZT-on-silicon resonators were measured on a Rhode & Schwarz ZVB8 network analyzer terminated to  $50 \Omega$  and calibrated using short, open, load, and through standards (GGB CS-5) with varied superimposed DC biases on both ports. The experimental  $R_m$  was extracted by fitting to the well-known modified Butterworth van-Dyke (mBVD) model, and  $Q_m$  was extracted via fitting. Tether resistances were obtained via resistivity test structures and combined designed and measured geometries. Using extracted properties, independently measured mechanical material properties [10], and only one fitted parameter ( $Q_m$ ), values of  $R_m$  were calculated (Fig. 5). Frequencies varied from 22.2 to 22.6 MHz at 10V. The extracted  $R_m$  varied from  $48\Omega$  to  $335\Omega$ , which may be attributed to the widely varying  $e_{31}$  (Fig. 4) and  $Q_m$ . Both  $e_{31}$  and  $Q_m$  vary cross-wafer and tune with voltage. Although there is a large spread in  $R_m$ , the model still matches well without fitting  $e_{31}$ . The average error across all voltages was 23%.

### D. Predicting High Performance

Fig. 6 plots the predicted  $S_{21}$  of device 2 of Fig 5. when placed in parallel six times along with the measured performance of the device from [5] at 8 V. Device 2 was located on the same die as the device from [5]. Since the frequency and shunt capacitance are related to the radius, it is necessary to array the resonators to obtain the desired shunt impedance. This

agreement illustrates the ability of this model to accurately predict the behavior of low loss and low  $R_m$  devices.

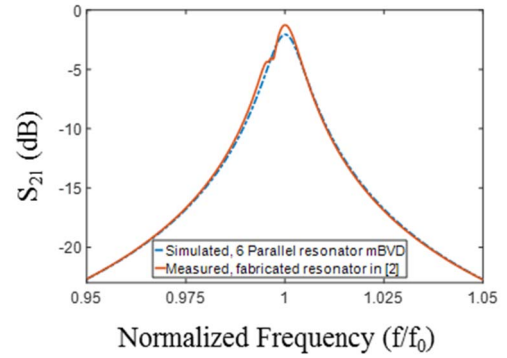


Fig. 6. Simulated  $S_{21}$  for Device 2 of Figure 5 in parallel six times, and measured resonator data from the device in [2] at 8V. Both devices were located on the same die. Discrepancies arise from unknown individual disk  $Q_m$ , varying  $e_{31}$  and center frequency misalignment.

## IV. CONCLUSION

The  $R_m$  model presented and validated has provided insight into the design of disk resonators, and has been able to predict the high performance demonstrated in [5]. They are expected to maintain consistent motional resistance across a wide range of frequencies, so long as the device thickness does not approach the radius. Thicker devices with the piezoelectric layer far from the neutral axis are predicted to have a lower  $R_m$  as a general trend if  $Q_m$  is not affected by relative layer thicknesses. These insights, along with the demonstrated low loss in fabricated devices, warrant a more thorough investigation into the scaling of this family of modes to higher frequencies.

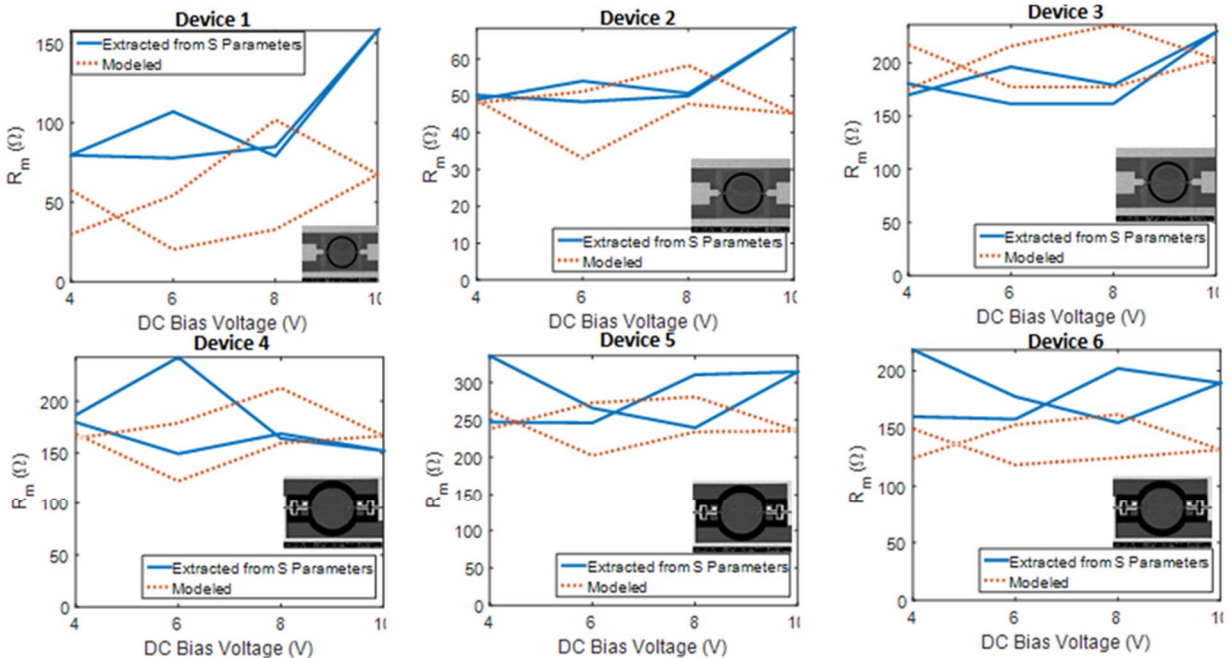


Fig 5. Extracted (solid) and modeled (dashed)  $R_m$ . Modeled  $R_m$  used extracted  $e_{31}$  and  $Q_m$  (Fig. 2). Discrepancies in curve shapes is attributed to differences in  $e_{31}$  extraction conditions (quasi-static) and RF testing conditions. Overall average error was 23%.

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