

# Gas-Liquid Two-Phase Flow Studies using Three-Field Two-Fluid Model and Two- Group Interfacial Area Transport Equation

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# Motivation and Objective

- **Motivation:** Computational Fluid Dynamics (CFD) simulation with a good two-phase model can give detailed three dimensional aspects for better design and performance of Two-phase systems (Nuclear Reactor, steam generator, heat exchanger, bubble column chemical reactor, electronic cooling system)
- **Objective:**
  - ✓ To develop a CFD framework within two-fluid model Formulation (Three-Field Two-fluid model with two group IATE) which can be used to predict the bubbly to churn-turbulent flow.
  - ✓ Select the right set of closure relations based on physics of adiabatic two-phase flow
    - IATE models (coalescence and breakup models) – Sun et al. (2001) selected –Needs to be tested in 3D form against Non-Uniform Conditions in CFD code
    - Existing Hydrodynamics model
    - Develop appropriate model based on correct physics observed in experiments

# Presentation Outline

- **Introduction-Theoretical background**
- **Experimental setup and Non-Uniform data**
- **CFD Benchmark preparation**
- **Bubbly flow simulation Results**
- **Two-group Simulations (Cap-bubbly, Cap-Turbulent)**
- **Summary and Future work**

# Introduction

## Theoretical Background

- ◆ 3-D two-fluid model (Ishii, 1975; Ishii & Hibiki, 2010)

$$\frac{\partial \alpha_k \rho_k}{\partial t} + \nabla \cdot (\alpha_k \rho_k \bar{v}_k) = \Gamma_k$$

$$\frac{\partial \alpha_k \rho_k \bar{v}_k}{\partial t} + \nabla \cdot (\alpha_k \rho_k \bar{v}_k \bar{v}_k) = -\alpha_k \nabla p_k + \nabla \cdot \alpha_k (\bar{\tau} + \tau_k^t) + \alpha_k \rho_k \bar{g} + \bar{v}_{ki} \Gamma_k + \bar{M}_{ik} - \nabla \alpha_k \cdot \bar{\tau}_{ki}$$

$$\frac{\partial \alpha_k \rho_k \bar{i}_k}{\partial t} + \nabla \cdot (\alpha_k \rho_k \bar{i}_k \bar{v}_k) = -\nabla \cdot \alpha_k (\bar{q}_k + q_k^t) + \alpha_k \frac{D_k p_k}{Dt} + \bar{i}_{ki} \Gamma_k + a_i q_{ki}'' + \phi_k$$

- ◆ Interfacial transfer terms due to time average  
(Interfacial transfer term) =  $a_i \times$  (Driving flux)

### ◆ Conventional approach

- Flow regime dependent correlations- **Static** approach, Causes Numerical Bifurcation during transition

### ◆ Advanced Approach

- **Interfacial Area Transport Equation (IATE)**- **Dynamic** approach, Multidimensional (1-D, 3-D)

$$\frac{\partial a_i}{\partial t} + \nabla \cdot (a_i \bar{v}_i) = \frac{2}{3} \left( \frac{a_i}{\alpha_g} \right) \left[ \frac{\partial \alpha_g}{\partial t} + \nabla \cdot (\alpha_g \bar{v}_g) - \eta_{ph} \right] + \frac{1}{3\Psi} \left( \frac{a_i}{\alpha_g} \right)^2 \sum_j R_j + \pi D_{bc}^2 R_{ph}$$

# One Group IATE

- Particle transport equation

$$\frac{\partial f}{\partial t} + \nabla \cdot (f \mathbf{v}) + \frac{\partial}{\partial V} \left( f \frac{dV}{dt} \right) = \sum_j S_j + S_{ph}$$

- One Group number density transport Equation:  $n(\mathbf{x}, t) = \int_{V_{min}}^{V_{max}} f(V, \mathbf{x}, t) dV$

$$\frac{\partial n}{\partial t} + \nabla \cdot (n \mathbf{v}_{pm}) = \sum_j R_j + R_{ph}$$

- One Group Void transport:  $\alpha_g(\mathbf{x}, t) = \int_{V_{min}}^{V_{max}} f(V, \mathbf{x}, t) V dV$

$$\frac{\partial \alpha_g}{\partial t} + \nabla \cdot (\alpha_g \mathbf{v}_g) + \int_{V_{min}}^{V_{max}} \left\{ V \frac{\partial}{\partial V} \left( f \frac{dV}{dt} \right) \right\} dV = \int_{V_{min}}^{V_{max}} \left( \sum_j S_j + S_{ph} \right) V dV$$

$$\int_{V_{min}}^{V_{max}} \left\{ V \frac{\partial}{\partial V} \left( f \frac{dV}{dt} \right) \right\} dV = \left( \frac{\dot{V}}{V} \right) \{ -\alpha_g \}$$

$$\begin{aligned} \frac{dm_b}{dt} &= (\Gamma_g - \eta_{ph} \rho_g) \cdot \frac{V_b}{\alpha_g} \\ \frac{dm_b}{dt} &= \frac{d(\rho_g V_b)}{dt} = V_b \frac{d\rho_g}{dt} + \rho_g \frac{dV_b}{dt} \\ \frac{1}{V} \frac{dV}{dt} &= \frac{1}{\rho_g} \left( \frac{\Gamma_g - \eta_{ph} \rho_g}{\alpha_g} - \frac{d\rho_g}{dt} \right) \end{aligned}$$

$$\eta_{ph} \equiv \int_{V_{min}}^{V_{max}} S_{ph} V dV$$

$$\frac{\partial \alpha_g \rho_g}{\partial t} + \nabla \cdot \alpha_g \rho_g \mathbf{v}_g = \Gamma_g$$

- One Group IATE transport:

$$a_i(\mathbf{x}, t) = \int_{V_{\min}}^{V_{\max}} f(V, \mathbf{x}, t) A_i(V) dV$$

$$\frac{\partial a_i}{\partial t} + \nabla \cdot (a_i \mathbf{v}_i) + \int_{V_{\min}}^{V_{\max}} \left\{ A_i \frac{\partial}{\partial V} \left( f \frac{dV}{dt} \right) \right\} dV = \int_{V_{\min}}^{V_{\max}} \left( \sum_j S_j + S_{ph} \right) A_i dV$$

$$\downarrow$$

$$\int_{V_{\min}}^{V_{\max}} \left\{ A_i \frac{\partial}{\partial V} \left( f \frac{dV}{dt} \right) \right\} dV = \left( \frac{\dot{V}}{V} \right) \left( -\frac{2}{3} a_i \right)$$

$$\frac{\partial a_i}{\partial t} + \nabla \cdot (a_i \mathbf{v}_i) - \frac{2}{3} \frac{a_i}{\alpha_g} \left\{ \frac{\partial \alpha_g}{\partial t} + \nabla \cdot (\alpha_g \mathbf{v}_g) - \eta_{ph} \right\} = \int_{V_{\min}}^{V_{\max}} \left( \sum_j S_j + S_{ph} \right) A_i dV$$

$$\frac{\partial a_i}{\partial t} + \nabla \cdot (a_i \bar{\mathbf{v}}_i) = \left( \frac{2}{3} \right) \left( \frac{a_{i1}}{\alpha_{g1}} \right) \left( \frac{\partial \alpha_g}{\partial t} + \nabla \cdot (\alpha_g \bar{\mathbf{v}}_g) - \eta_{ph} \right) + \sum_j \phi_j$$

$R_j \equiv \int_V S_j dV$  : particle number density source/sink

$\phi_j \equiv \int_V S_j A_j dV$  : interfacial area concentration source/sink

$\eta_j \equiv \int_V S_j V dV$  : void fraction source/sink

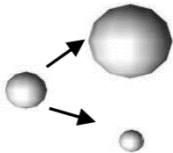
$$\frac{\partial a_i}{\partial t} + \nabla \cdot (a_i \bar{\mathbf{v}}_i) = \frac{2}{3} \left( \frac{a_i}{\alpha_g} \right) \left[ \frac{\partial \alpha_g}{\partial t} + \nabla \cdot (\alpha_g \bar{\mathbf{v}}_g) - \eta_{ph} \right] + \frac{1}{3\Psi} \left( \frac{a_i}{\alpha_g} \right)^2 \sum_j R_j + \pi D_{bc}^2 R_{ph}$$

# Interfacial Area Transport Equation

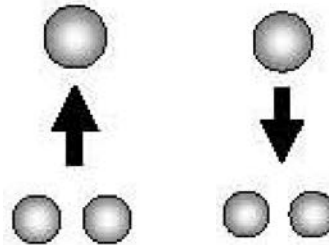
## ◆ Interfacial Area Transport Equation (Kojasoy and Ishii, 1995)

$$\frac{\partial a_i}{\partial t} + \nabla \cdot (a_i \vec{v}_i) = \frac{2}{3} \left( \frac{a_i}{\alpha_g} \right) \left[ \frac{\partial \alpha_g}{\partial t} + \nabla \cdot (\alpha_g \vec{v}_g) - \eta_{ph} \right] + \frac{1}{3\Psi} \left( \frac{a_i}{\alpha_g} \right)^2 \sum_j R_j + \pi D_{bc}^2 R_{ph}$$

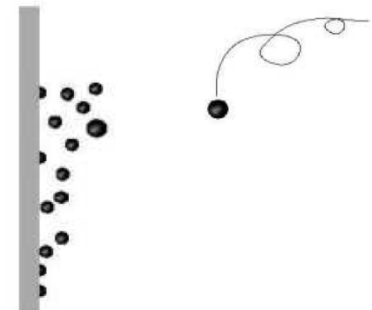
Contribution due to  
particle **volume**  
**change**



Contribution due  
to fluid particle  
interactions:  
**coalescence &**  
**breakup**



Contribution due  
to phase change:  
**nucleation &**  
**condensation**



# Key Bubble Length Scale and Bubble Group Boundary

## ◆ Two-Group Approach

- Group-1 bubbles: spherical and distorted bubbles
- Group-2 bubbles: cap, slug and churn bubbles

Description	Length scale
Spherical bubble limit	$D_{ds} = 4 \sqrt{\frac{2\sigma}{g\Delta\rho}} N_{\mu_f}^{1/3}$
Maximum distorted bubble limit	$D_{d,\max} = 4 \sqrt{\frac{\sigma}{g\Delta\rho}}$
Maximum cap bubble limit	$D_{c,\max} = 40 \sqrt{\frac{\sigma}{g\Delta\rho}}$
Critical bubble size at the group boundary in narrow channel	$D_c = 1.7 G^{1/3} \left( \frac{\sigma}{g\Delta\rho} \right)^{1/3}$

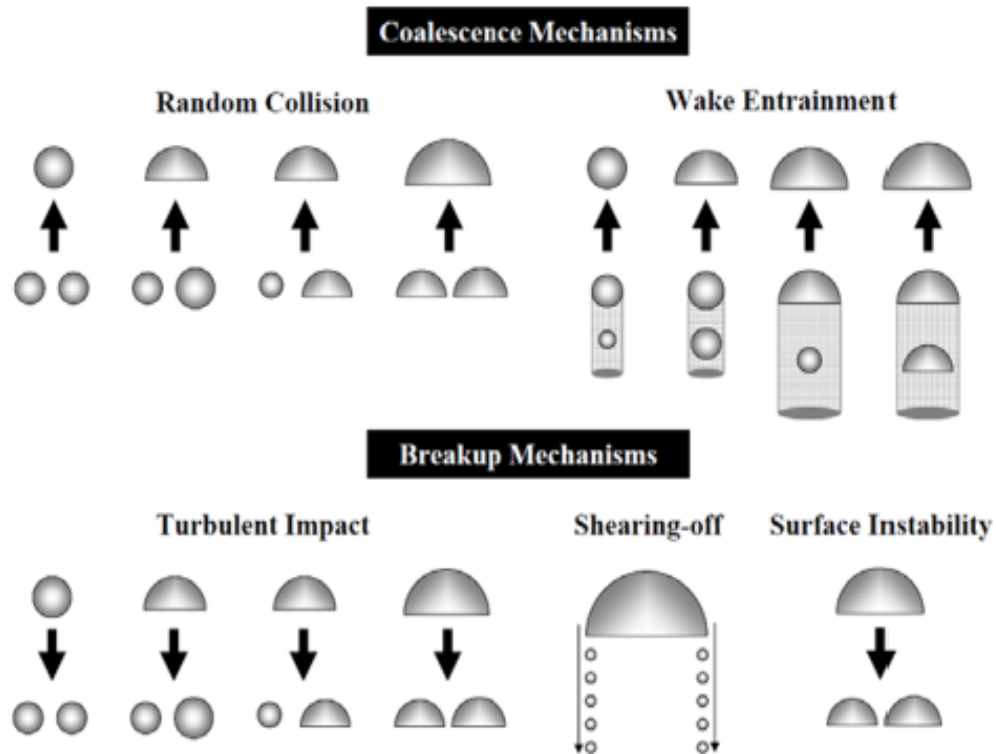
## Two-Group IATE transport:

$$\frac{\partial a_{i1}}{\partial t} + \nabla \cdot (a_{i1} \vec{v}_{i1}) = \left( \frac{2}{3} - \chi D_{c1}^{*2} \right) \left( \frac{a_{i1}}{\alpha_{g1}} \right) \left( \frac{\partial \alpha_{g1}}{\partial t} + \nabla \cdot \alpha_{g1} \vec{v}_{g1} \right) + \sum_j \phi_{j,1}$$

$$\frac{\partial a_{i2}}{\partial t} + \nabla \cdot (a_{i2} \vec{v}_{i2}) = \frac{2}{3} \left( \frac{a_{i2}}{\alpha_{g2}} \right) \left( \frac{\partial \alpha_{g2}}{\partial t} + \nabla \cdot \alpha_{g2} \vec{v}_{g2} \right) + \chi D_{c1}^{*2} \left( \frac{a_{i1}}{\alpha_{g1}} \right) \left( \frac{\partial \alpha_{g1}}{\partial t} + \nabla \cdot \alpha_{g1} \vec{v}_{g1} \right) + \sum_j \phi_{j,2}$$



# Schematics of Two-Group Bubble Interaction (Ishii et al., 2002)



# Two-Group IATE with Three-Field Two-Fluid Model: Two Gas Momentum Equations

## ◆ Two-group gas continuity equation

$$\frac{\partial(\alpha_{g1}\rho_g)}{\partial t} + \nabla \cdot (\alpha_{g1}\rho_g \bar{v}_{g1}) = -\Delta m_{12}$$

$$\frac{\partial(\alpha_{g2}\rho_g)}{\partial t} + \nabla \cdot (\alpha_{g2}\rho_g \bar{v}_{g2}) = \Delta m_{12}$$

## ◆ Two-group gas momentum equation

$$\frac{\partial \alpha_{g1}\rho_g \bar{v}_{g1}}{\partial t} + \nabla \cdot \alpha_{g1}\rho_g \bar{v}_{g1} \bar{v}_{g1} = -\alpha_{g1}\nabla p_{g1} + \nabla \cdot \alpha_{g1} \left( \bar{\tau}_{g1} + \tau_{g1}^t \right) + \alpha_{g1}\rho_g \bar{g} - \Delta m_{12} \bar{v}_{g1i} + \bar{M}_{ig1} - \nabla \alpha_{g1} \cdot \bar{\tau}_{g1i}$$

$$\frac{\partial \alpha_{g2}\rho_g \bar{v}_{g2}}{\partial t} + \nabla \cdot \alpha_{g2}\rho_g \bar{v}_{g2} \bar{v}_{g2} = -\alpha_{g2}\nabla p_{g2} + \nabla \cdot \alpha_{g2} \left( \bar{\tau}_{g2} + \tau_{g2}^t \right) + \alpha_{g2}\rho_g \bar{g} + \Delta m_{12} \bar{v}_{g2i} + \bar{M}_{ig2} - \nabla \alpha_{g2} \cdot \bar{\tau}_{g2i}$$

## ◆ Two-group IATE

$$\frac{\partial a_{i1}}{\partial t} + \nabla \cdot (a_{i1} \bar{v}_{i1}) = \left( \frac{2}{3} - \chi D_{c1}^{*2} \right) \left( \frac{a_{i1}}{\alpha_{g1}} \right) \left( \frac{\partial \alpha_{g1}}{\partial t} + \nabla \cdot \alpha_{g1} \bar{v}_{g1} \right) + \sum_j \phi_{j,1}$$

$$\frac{\partial a_{i2}}{\partial t} + \nabla \cdot (a_{i2} \bar{v}_{i2}) = \frac{2}{3} \left( \frac{a_{i2}}{\alpha_{g2}} \right) \left( \frac{\partial \alpha_{g2}}{\partial t} + \nabla \cdot \alpha_{g2} \bar{v}_{g2} \right) + \chi D_{c1}^{*2} \left( \frac{a_{i1}}{\alpha_{g1}} \right) \left( \frac{\partial \alpha_{g1}}{\partial t} + \nabla \cdot \alpha_{g1} \bar{v}_{g1} \right) + \sum_j \phi_{j,2}$$

$a_i \times$  (Driving flux)

Coalescence and  
Breakup mechanism

# Momentum closures

$$M_{id} = \frac{\alpha_d}{B_d} \left( F_d^D + F_d^L + F_d^W + F_d^{TD} + F_d^{VM} + F_d^B \right)$$

$$= M_d^D + M_d^L + M_d^W + M_d^{TD} + M_d^{VM} + M_d^B$$

Interfacial Forces	Phases	Nature	Coefficient
Drag Force	Gas and Liquid	Interfacial force	Ishii and Zuber (1979)
Wall Lubrication Force	Gas and Liquid	Interfacial force	Antal et al. (1991) $C_{w1} = -0.01, C_{w2} = 0.05$
Lift force	Gas and Liquid	Interfacial force	Hibiki and Ishii (2007) $C_{L,G1} = 0.01, C_{L,G2} = -0.5$
Turbulent dispersion force	Gas and Liquid	Interfacial force	Bertodano (1992) $C_{TD,1} = 0.25$
Momentum Transfer by Bubble interaction mechanism (Random Collision)	Gas and Liquid	Interfacial force	Sharma et al. (2017, 2019)
Bubble-induced turbulence	Liquid	Turbulence Induced by relative motions of Bubbles in liquid	Sato et al. (1981) $C_{Sato,G1} = 0.6$ $C_{Sato,G2} = 2.1$ Lee (2010); Lopez de Bertodano et al.(2006); Prabhudharwadkar et al. (2012)

Liquid phase  
Turbulence:  
k-ε model

$$\mu_{t,f} = \mu_{SI} + \mu_{BI}$$

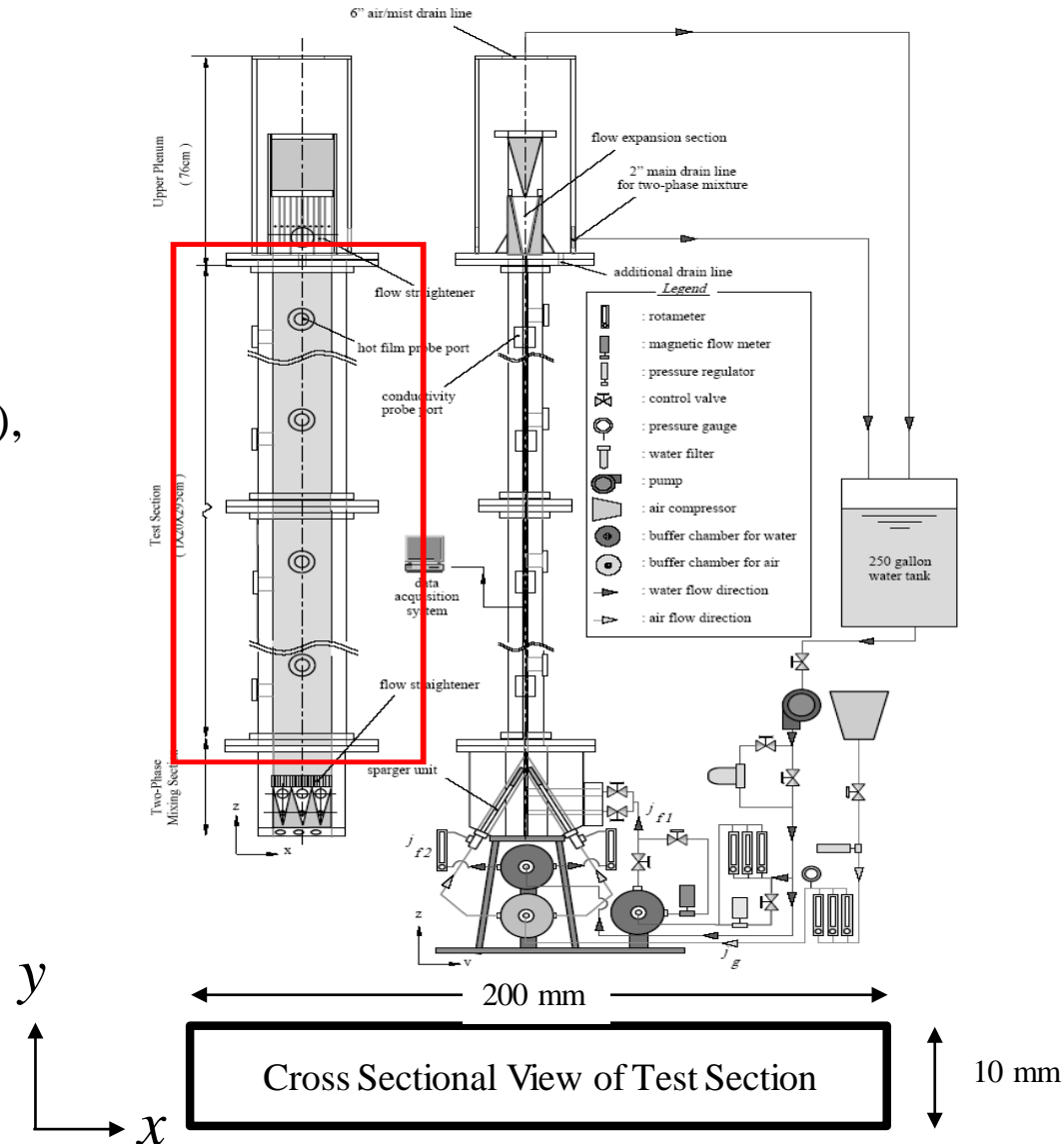
$$\mu_{SI} = C_{\mu} \rho_f \frac{k_f^2}{\varepsilon_f}; \mu_{BI} = C_{BI} \rho_f \alpha D_b |\nu_g - \nu_f|$$

Gas phase turbulence: zero order

$$\nu_{t,g} = \frac{\nu_{t,f}}{\sigma} \Rightarrow \mu_{t,g} = \frac{\rho_g}{\rho_f} \frac{\mu_{t,f}}{\sigma}; \sigma = 1$$

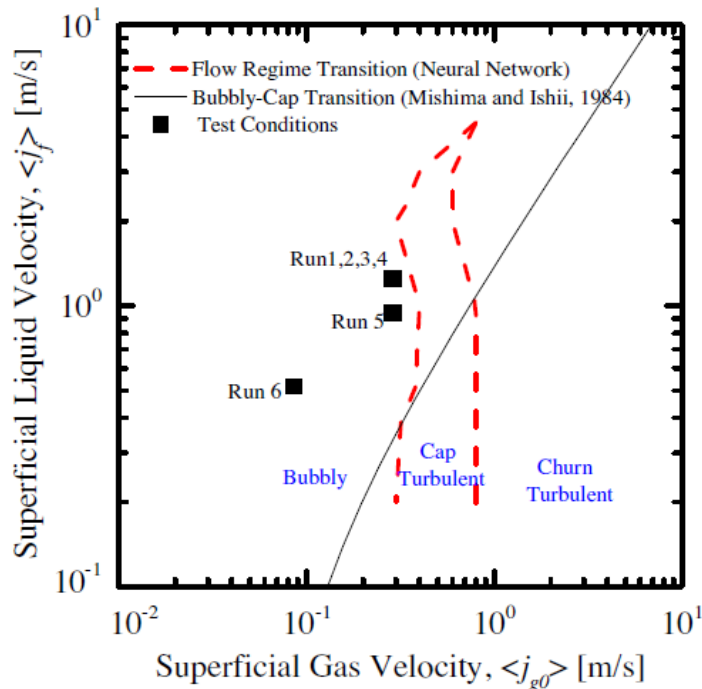
# Experimental Facility

- ◆ Rectangular channel
  - 200 mm x 10 mm x 2950 mm
- ◆ Total test section length
  - $z/D_h \sim 150$
- ◆ Local instrumentation ports
  - $z/D_h = 34.8$  (Port 2), 88.2 (Port 4), 142 (Port 6)
- ◆ Flow regime of interest
  - Bubbly
  - Cap-turbulent
  - Churn-turbulent
- ◆ Measured flow parameters
  - $\alpha_g, a_i, v_g, D_{Sm}$
- ◆ Experimental database
  - **Uniform** inlet injection
  - **Non-uniform** inlet injection



# Experimental Database Used for Benchmarking with Non-Uniform Inlet Boundary Conditions

## Non-Uniform inlet injection



### Uncertainty and Possible Error:

Kim et al (2000):

Void Fraction: 5%

Interfacial Area conc.: 10%

Le Corre et al (2003):

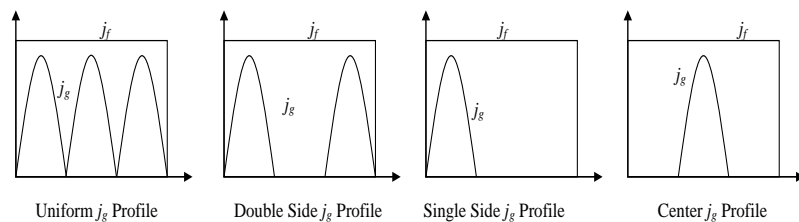
Void Fraction: 7%

Gas velocity: 12%

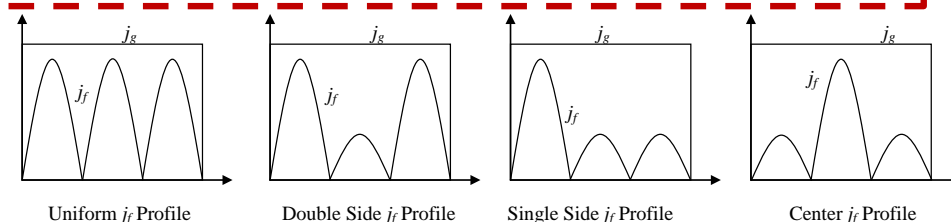
Interfacial Area conc.: 25%

### Factors affecting:

- finite size probe effect (missing bubbles)
- data acquisition (sampling frequency, total sampling time)
- signal processing (filtering, threshold)



## Uniform Liquid Injection



## Uniform Gas Injection

# CFD simulation strategy

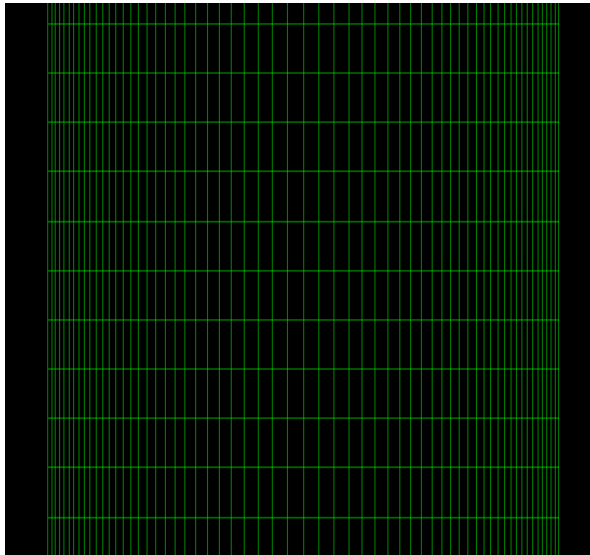
## ◆ B.C

### ● Inlet :

- ✓ **Local measured void fraction, gas velocity , IAC at Port 2**
- ✓ **Liquid velocity Profile** variation in x- direction obtained from gas velocity by subtracting slip velocity, in y- direction  $1/7^{\text{th}}$  power law variation used

### ● Outlet atmospheric pressure

## Mesh



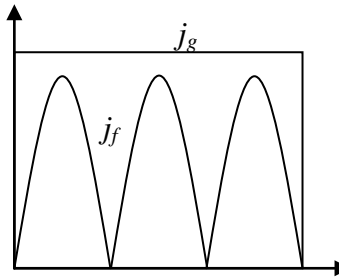
z Mesh Configuration: 60x10x162

→ **Wall B.C.** No slip for liquid phase  
and Free slip for Gas phase

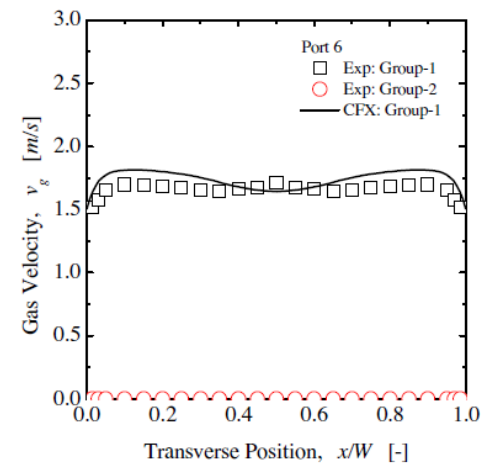
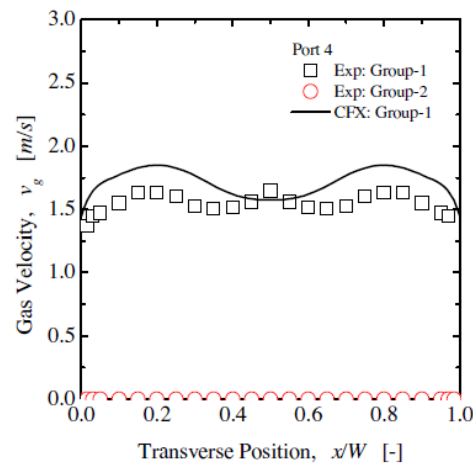
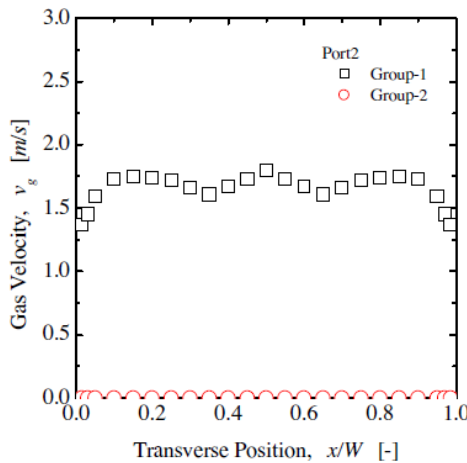
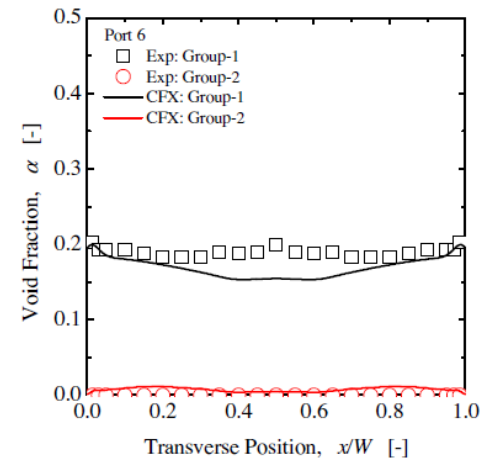
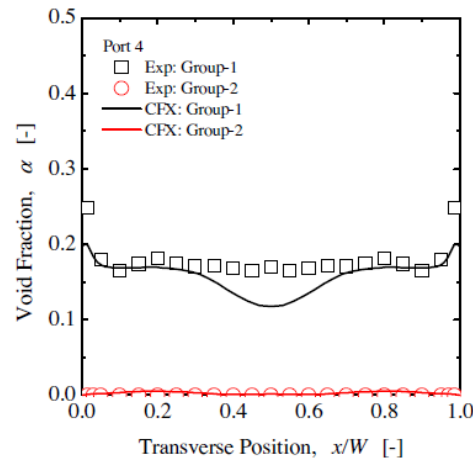
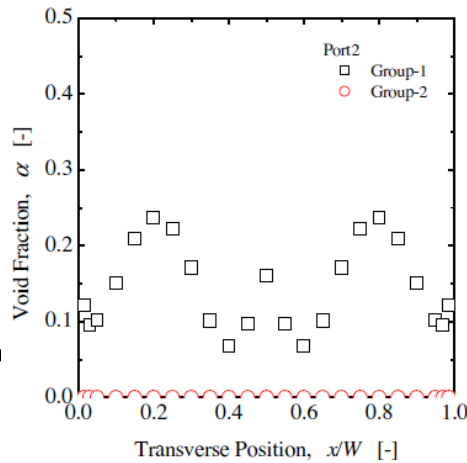
# Bubbly Flow - Run1

$$\langle j_{g,0} \rangle = 0.289 \text{ m/s}$$

$$\langle j_f \rangle = 1.25 \text{ m/s}$$



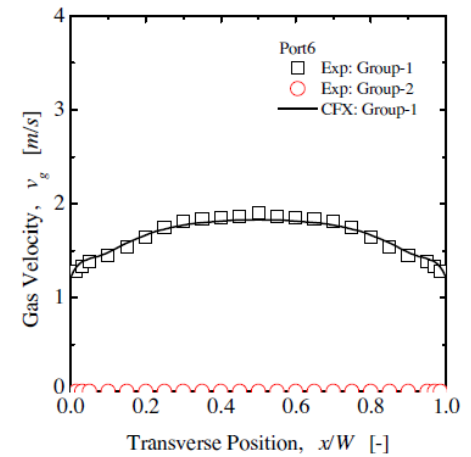
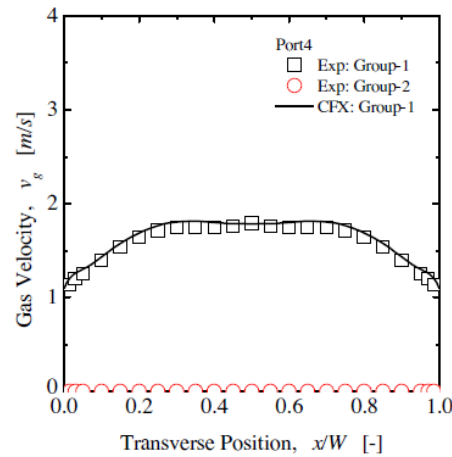
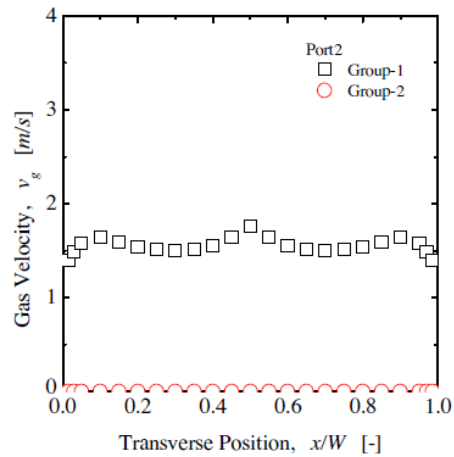
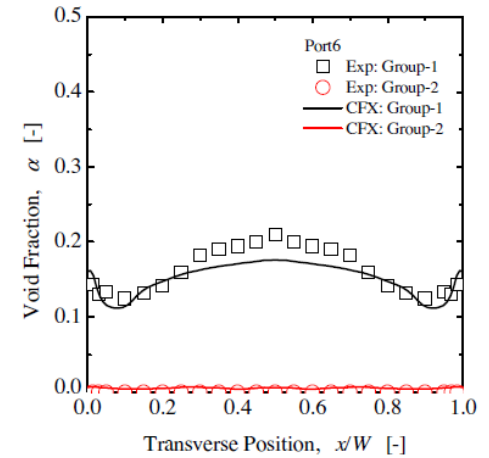
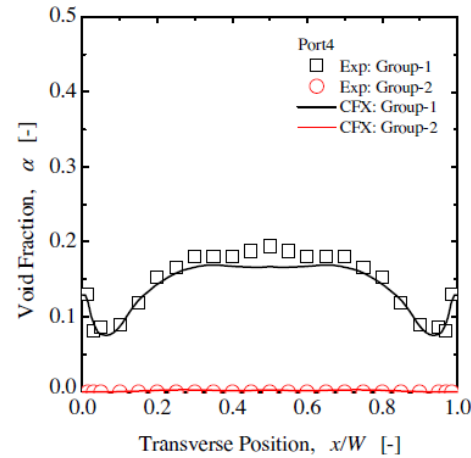
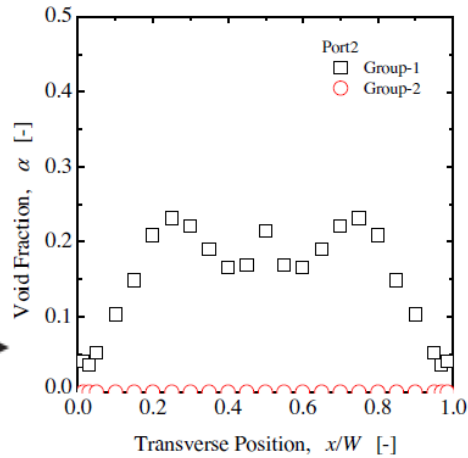
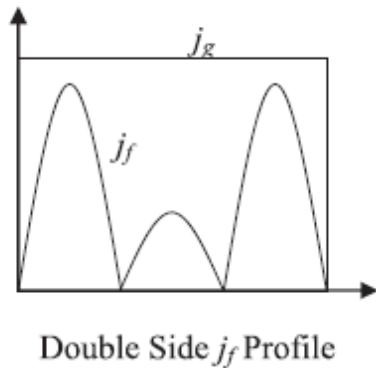
Uniform  $j_f$  Profile



# Bubbly Flow - Run 2

$$\langle j_{g,0} \rangle = 0.289 \text{ m/s}$$

$$\langle j_f \rangle = 1.25 \text{ m/s}$$

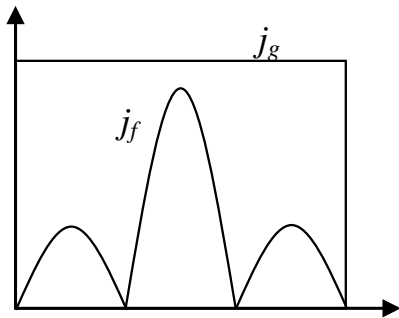




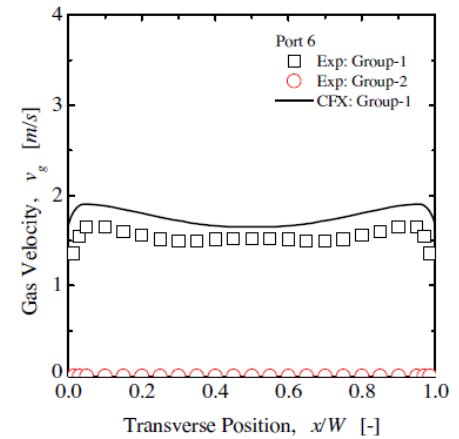
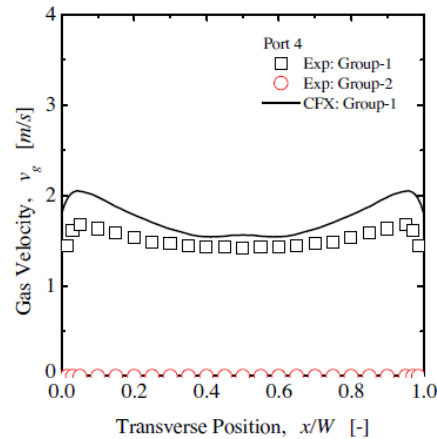
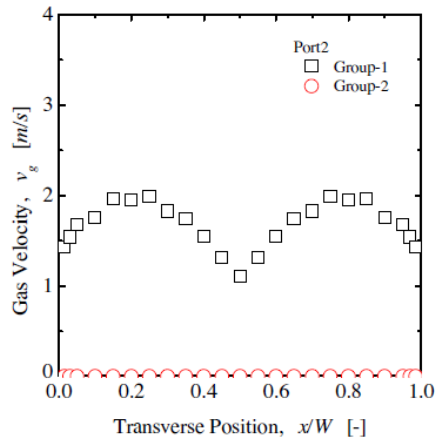
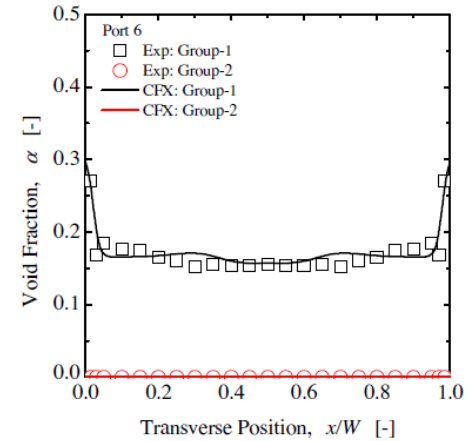
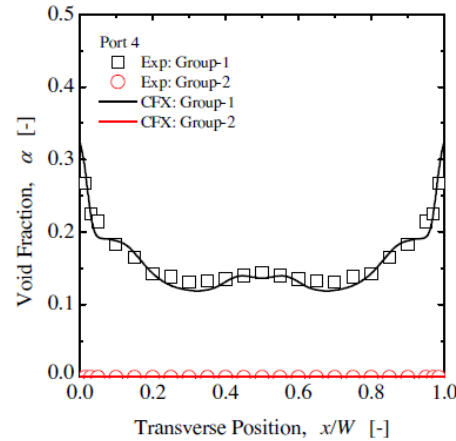
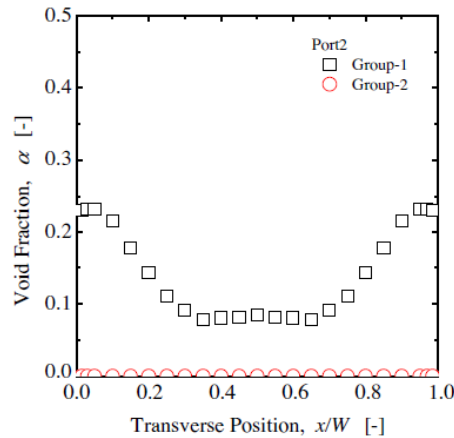
# Bubbly Flow - Run 3

$$\langle j_{g,0} \rangle = 0.289 \text{ m/s}$$

$$\langle j_f \rangle = 1.25 \text{ m/s}$$



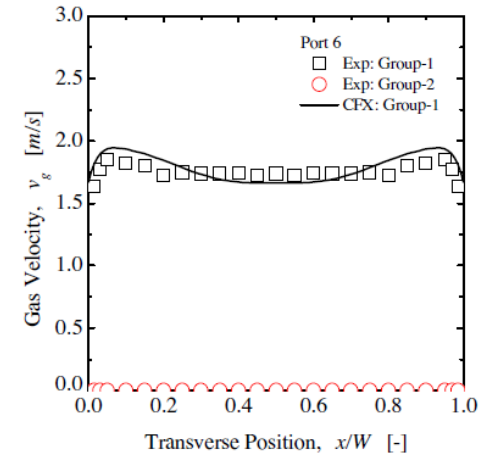
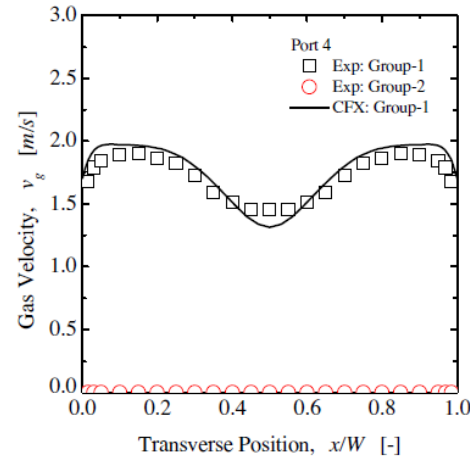
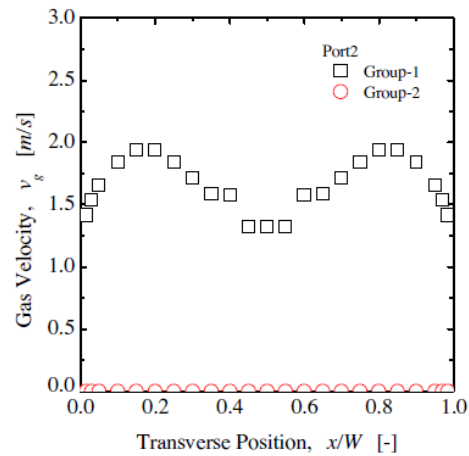
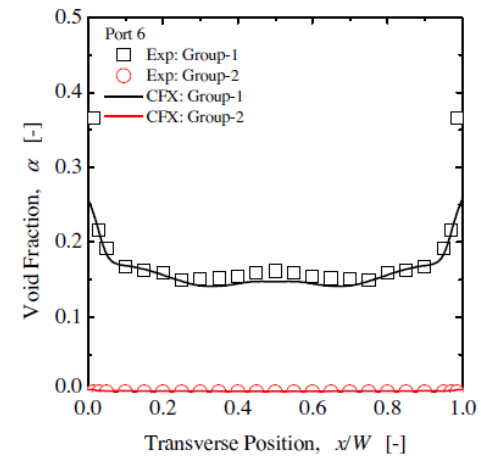
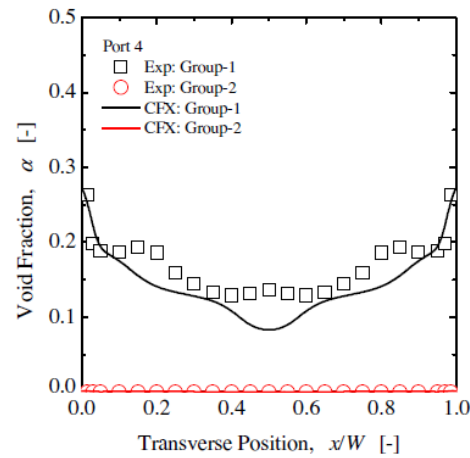
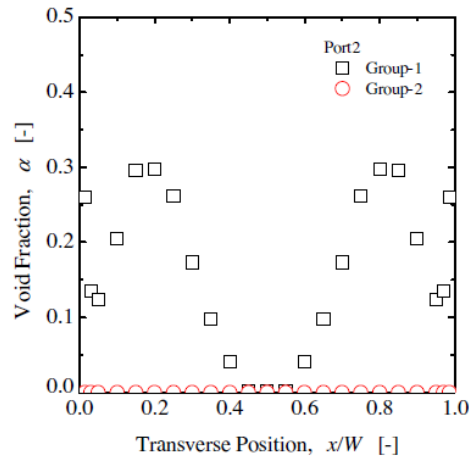
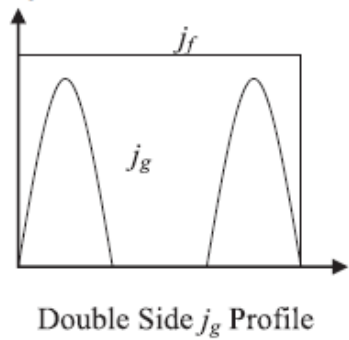
Center  $j_f$  Profile



# Bubbly Flow - Run 4

$$\langle j_{g,0} \rangle = 0.295 \text{ m/s}$$

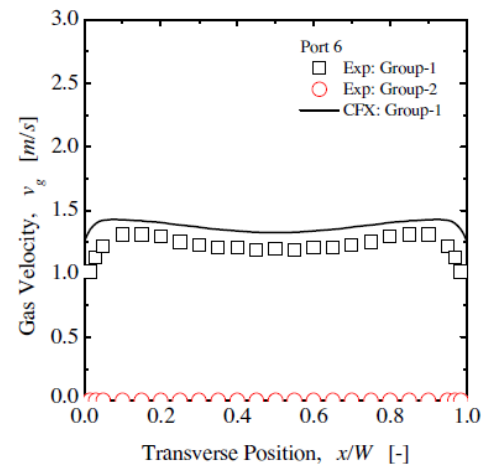
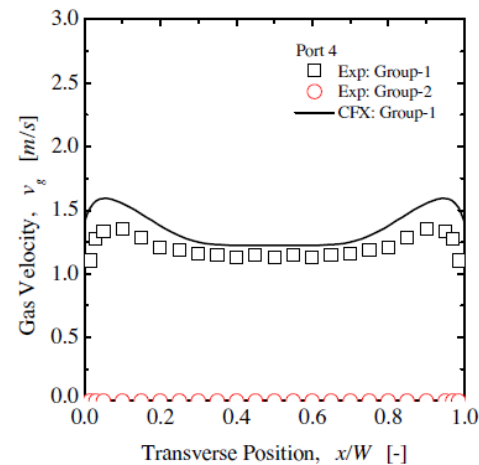
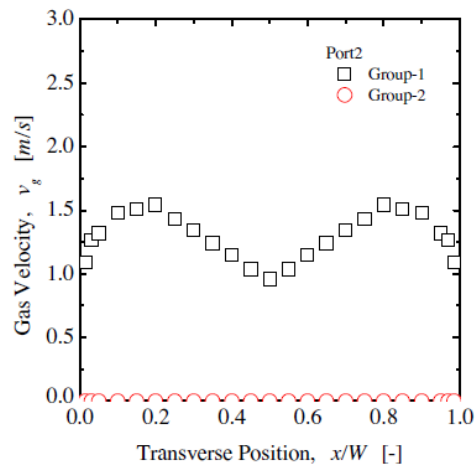
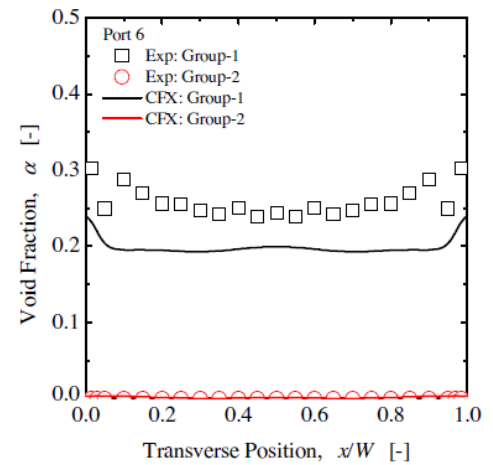
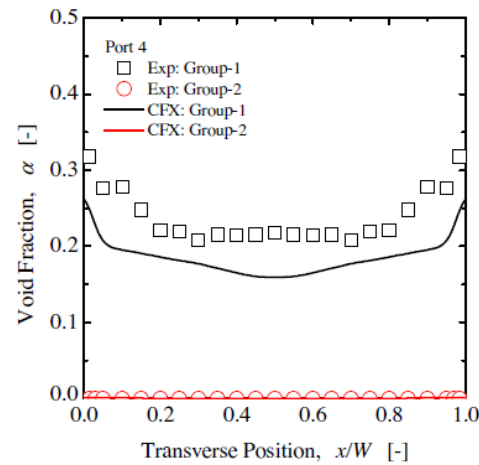
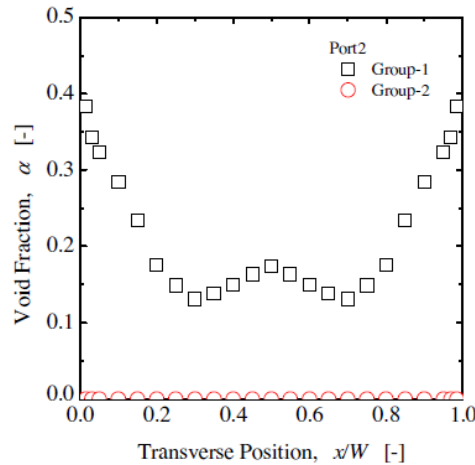
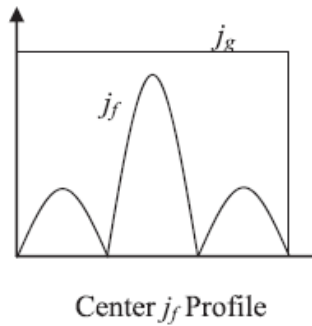
$$\langle j_f \rangle = 1.25 \text{ m/s}$$



# Bubbly Flow - Run 5

$$\langle j_{g,0} \rangle = 0.289 \text{ m/s}$$

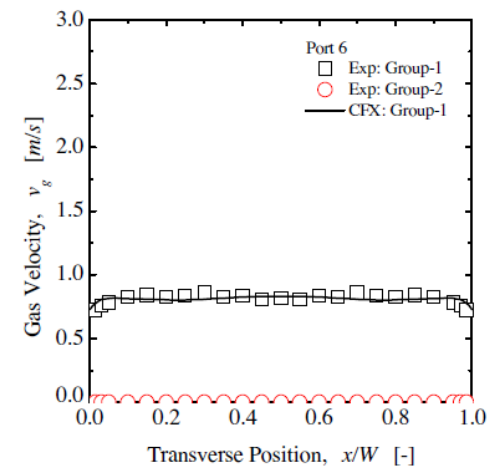
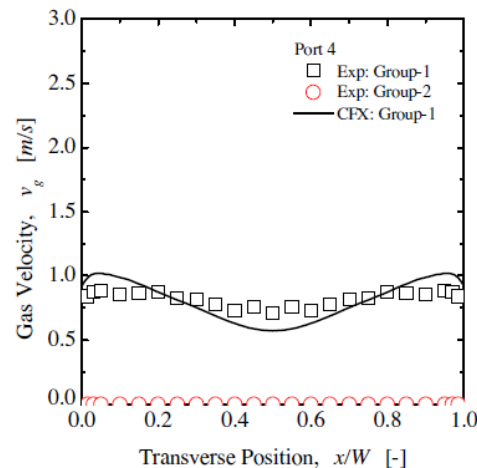
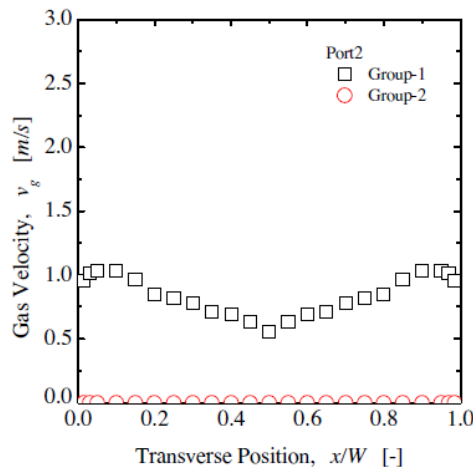
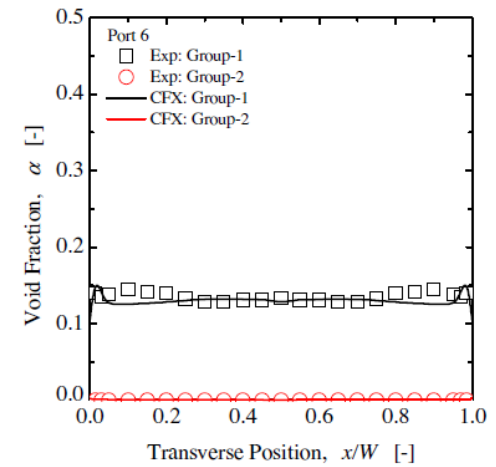
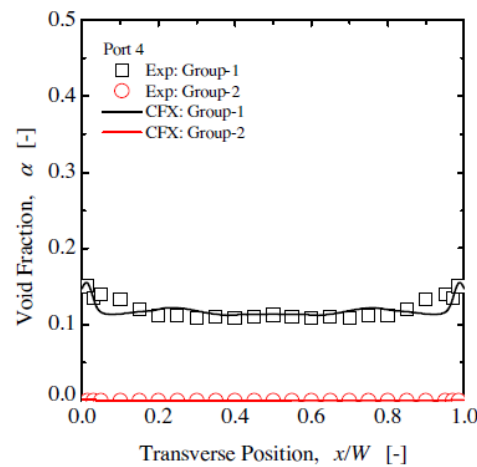
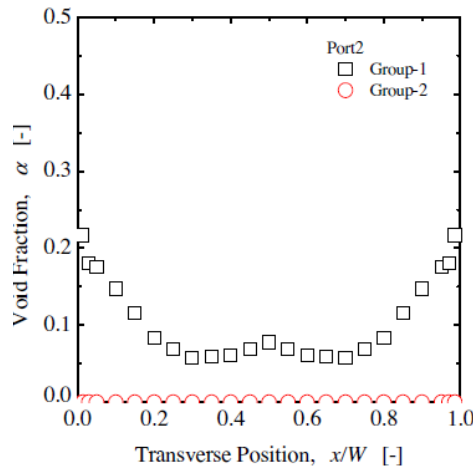
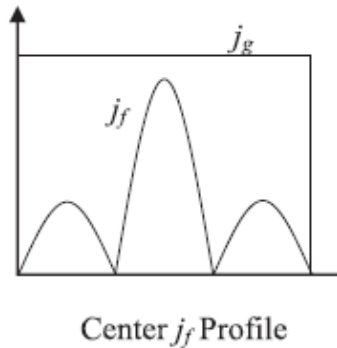
$$\langle j_f \rangle = 0.943 \text{ m/s}$$



# Bubbly Flow - Run 6

$$\langle j_{g,0} \rangle = 0.086 \text{ m/s}$$

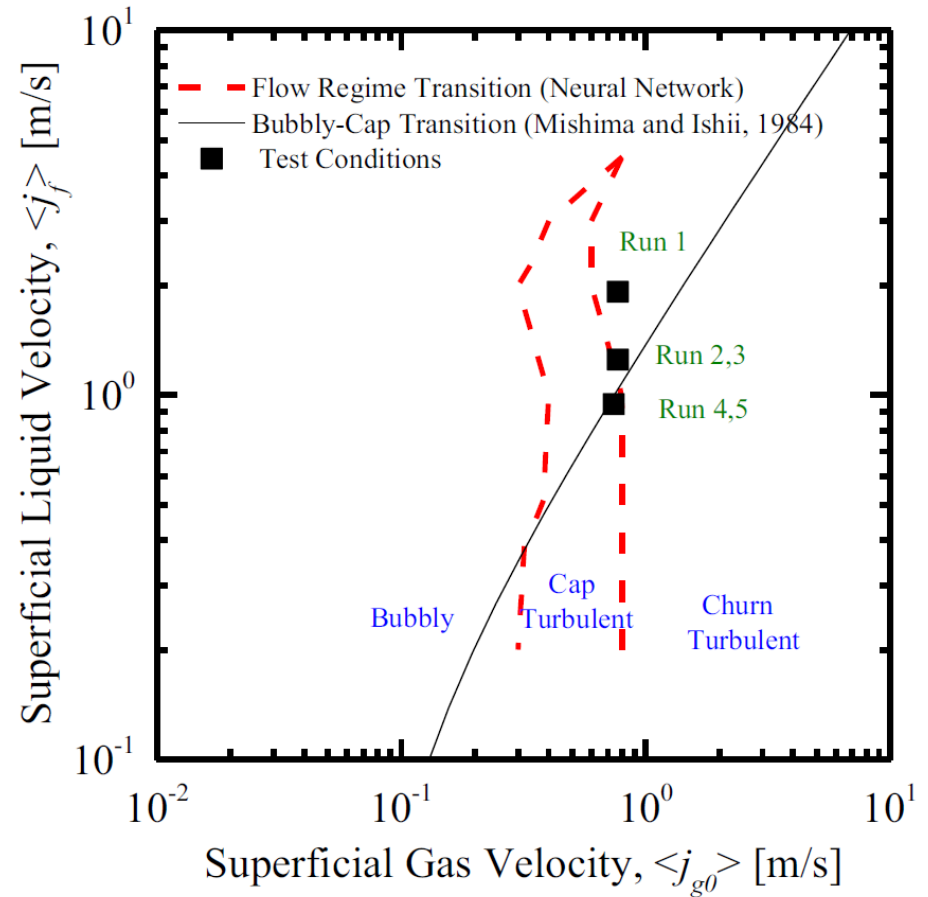
$$\langle j_f \rangle = 0.515 \text{ m/s}$$



- Hydraodynamics model predicting the trend in measured void fraction
- Good prediction of development of velocity profile by models

## Two-group case

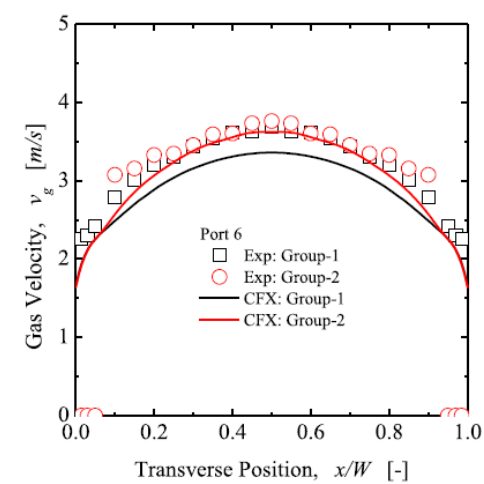
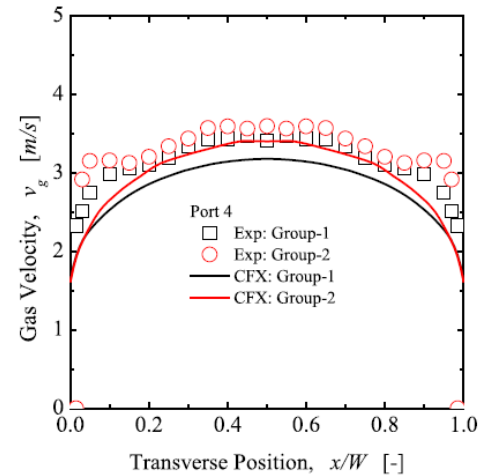
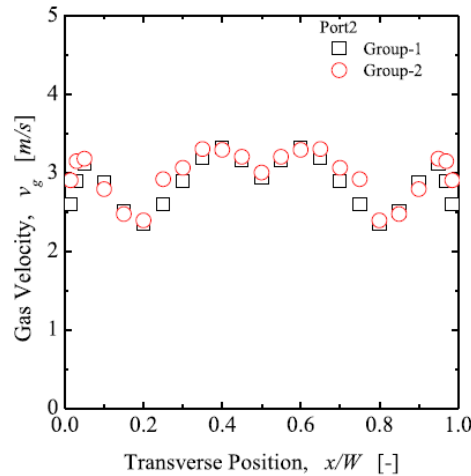
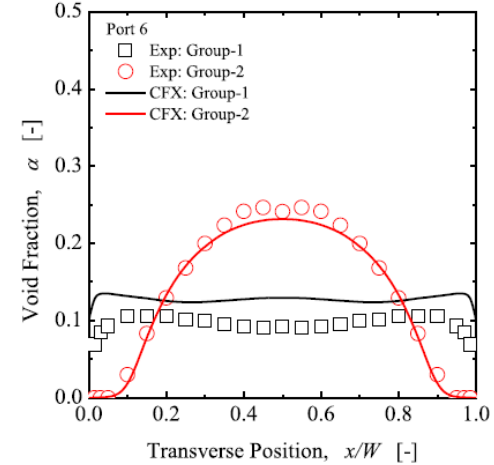
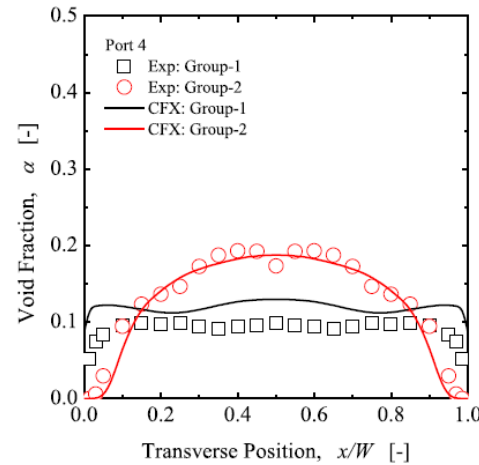
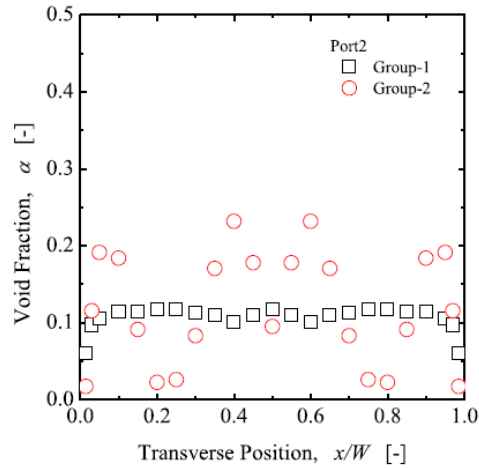
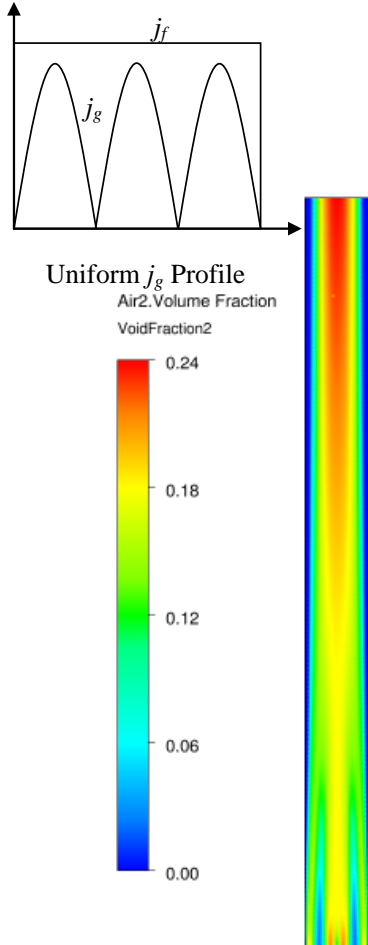
◆ Cap-bubbly, cap-turbulent, churn-Turbulent flows



# Two-Group cases: Run 1

$$\langle j_{g,0} \rangle = 0.77 \text{ m/s}$$

$$\langle j_f \rangle = 1.92 \text{ m/s}$$

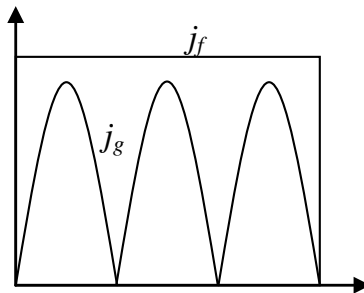


□ Good prediction for overall trend and magnitude of each bubble group in transverse direction

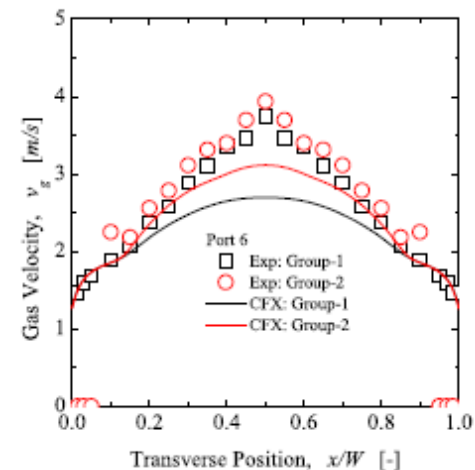
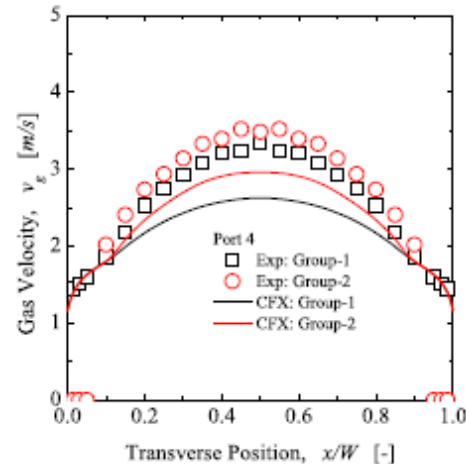
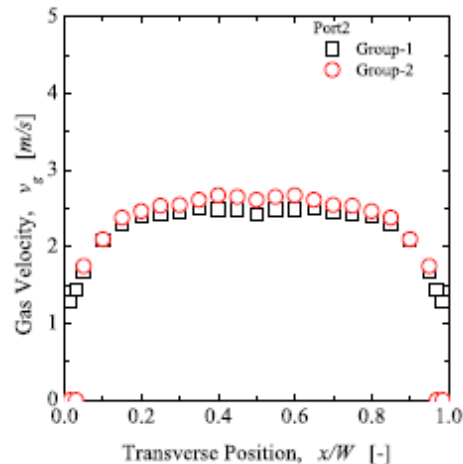
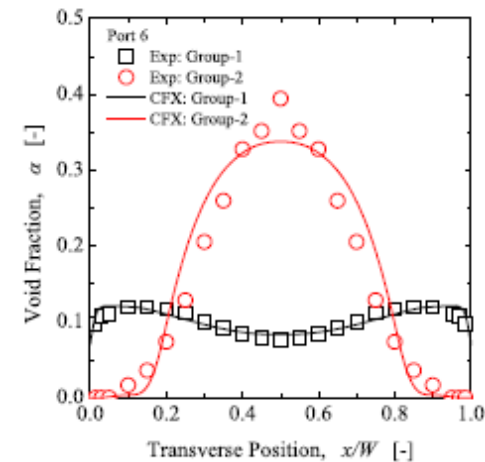
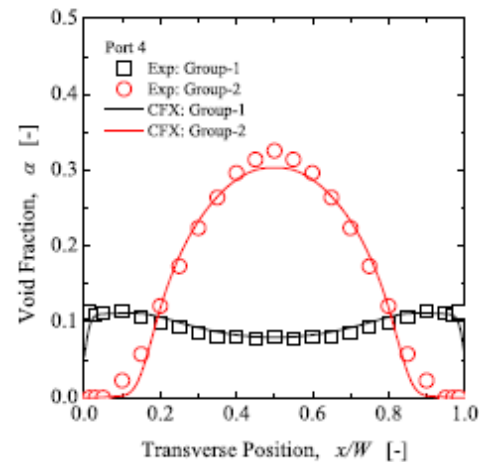
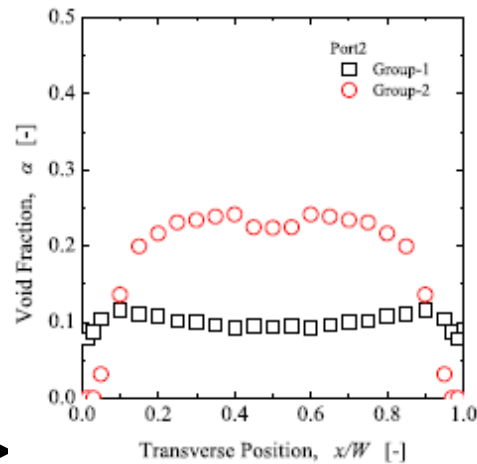
## Two-Group cases: Run 2

$$\langle j_{g,0} \rangle = 0.74 \text{ m/s}$$

$$\langle j_f \rangle = 1.25 \text{ m/s}$$



Uniform  $j_g$  Profile

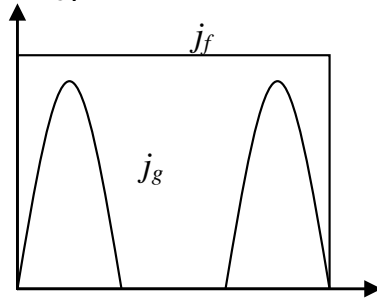


- Good prediction for overall trend of phase distribution in transverse direction.

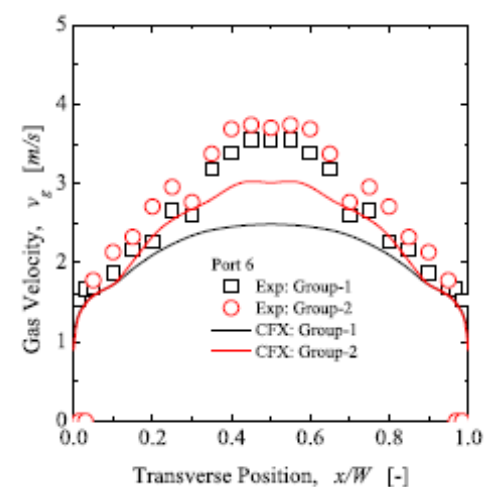
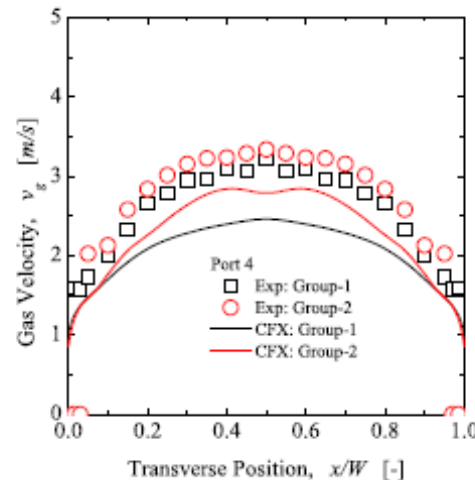
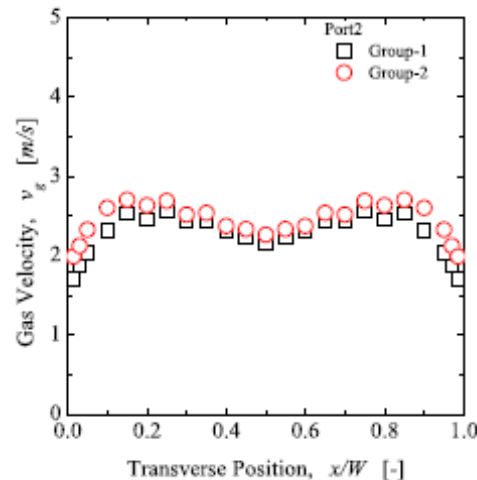
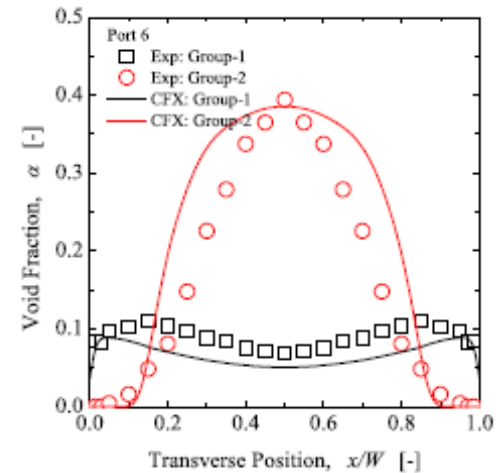
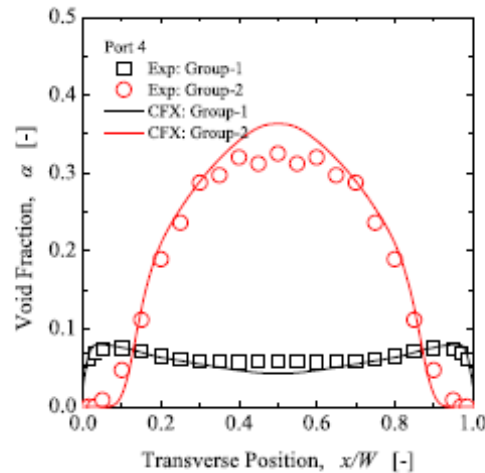
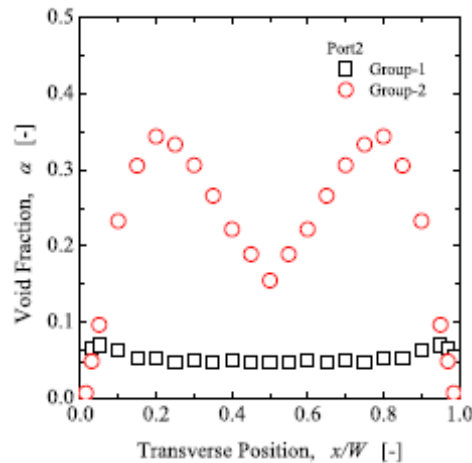
## Two-Group cases: Run 3

$$\langle j_{g,0} \rangle = 0.77 \text{ m/s}$$

$$\langle j_f \rangle = 1.25 \text{ m/s}$$



Double Side  $j_g$  Profile



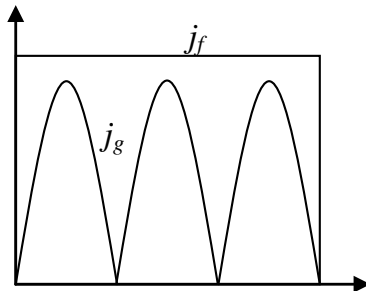
- Very Good prediction for overall trend of phase distribution in transverse direction.
- Underprediction of volume transfer from G-2 to G-1 bubbles



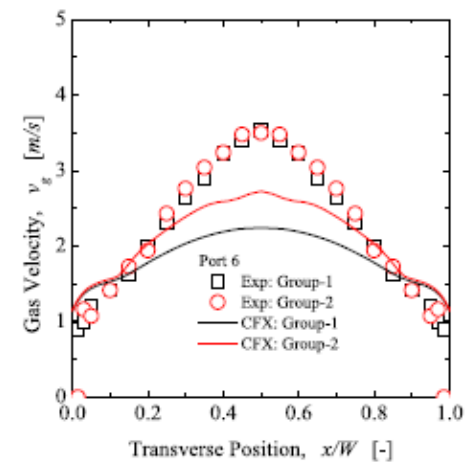
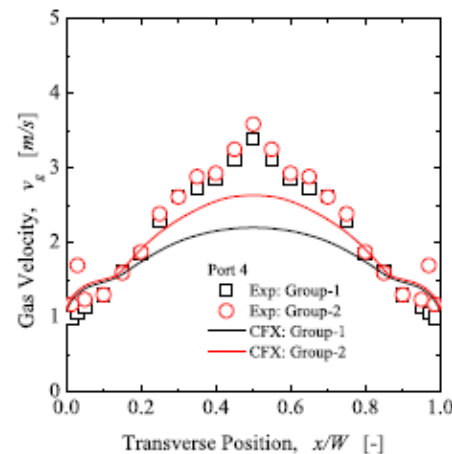
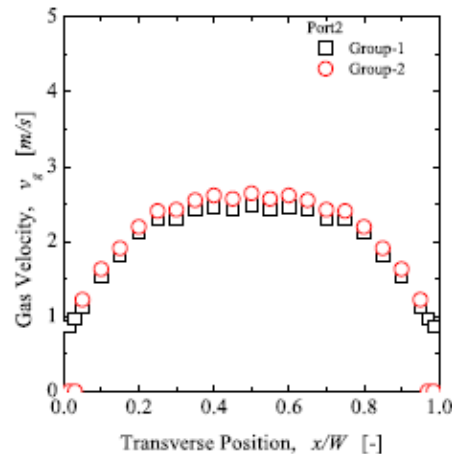
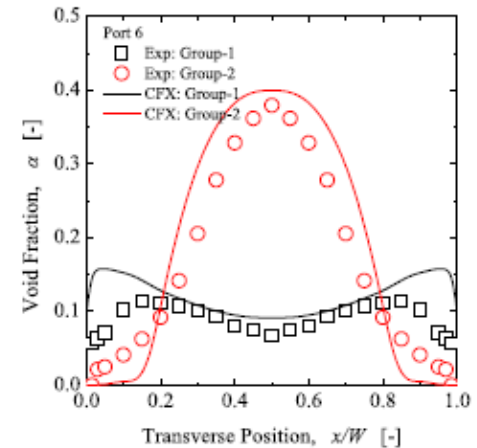
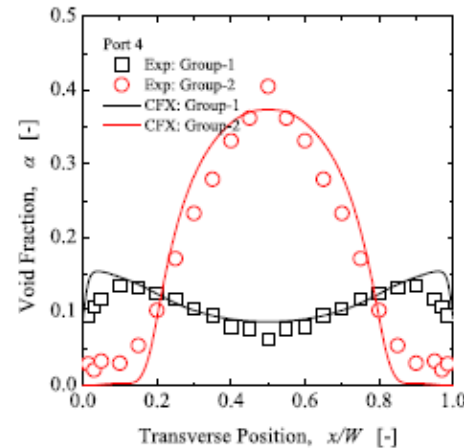
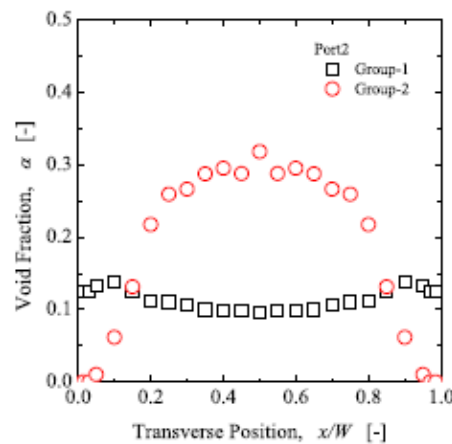
# Two-Group cases: Run 4

$$\langle j_{g,0} \rangle = 0.74 \text{ m/s}$$

$$\langle j_f \rangle = 0.943 \text{ m/s}$$



Uniform  $j_g$  Profile

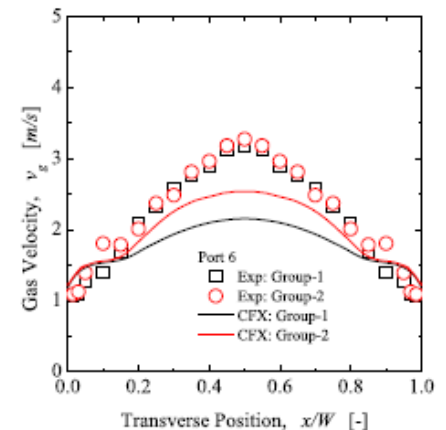
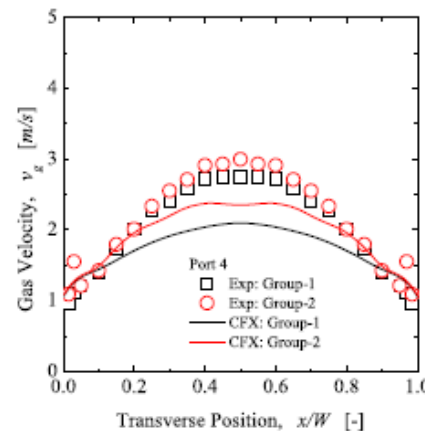
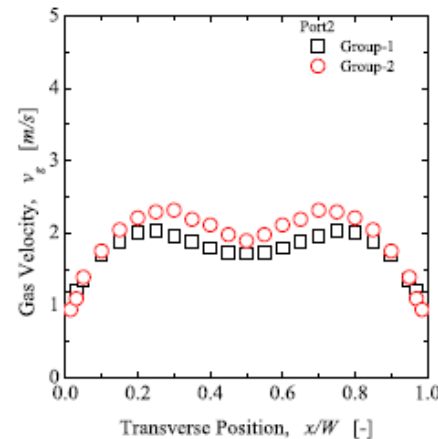
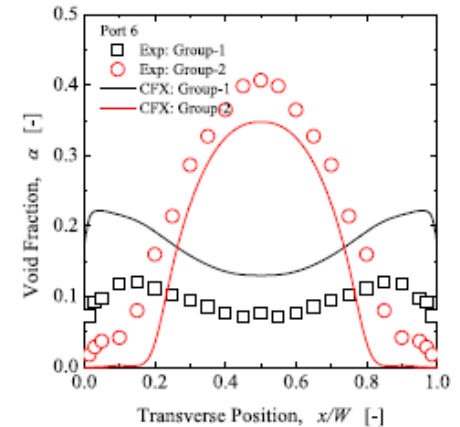
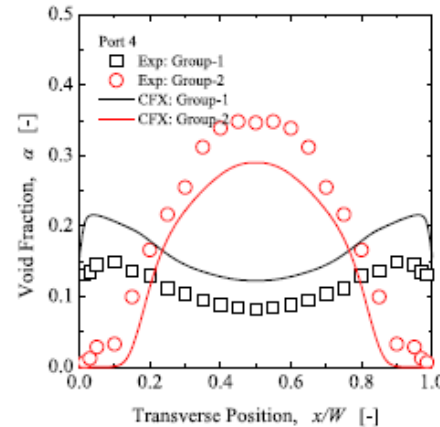
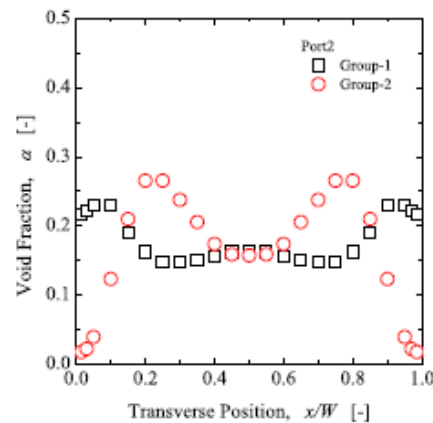
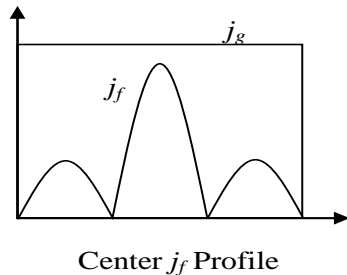


- Very Good prediction for overall trend of phase distribution in transverse direction.
- Underprediction of volume transfer from G-2 to G-1 bubbles

## Two-Group cases: Run 5

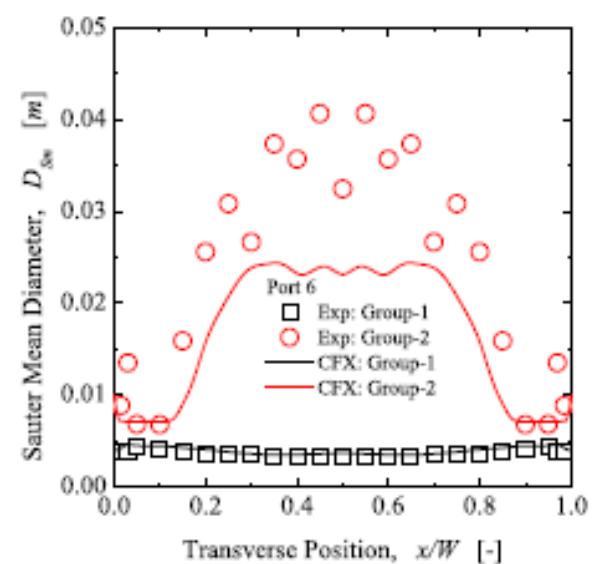
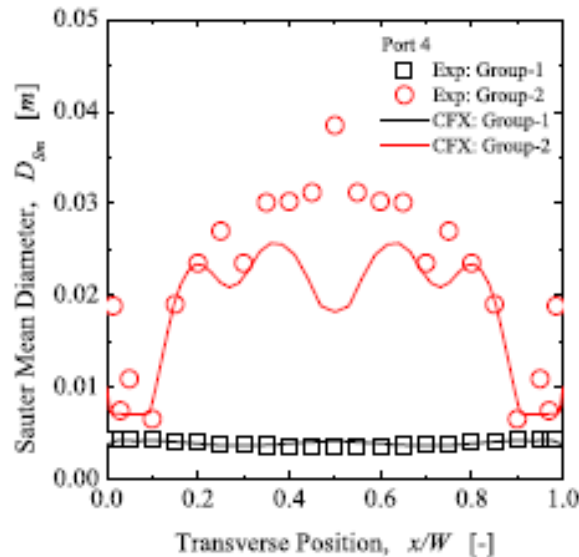
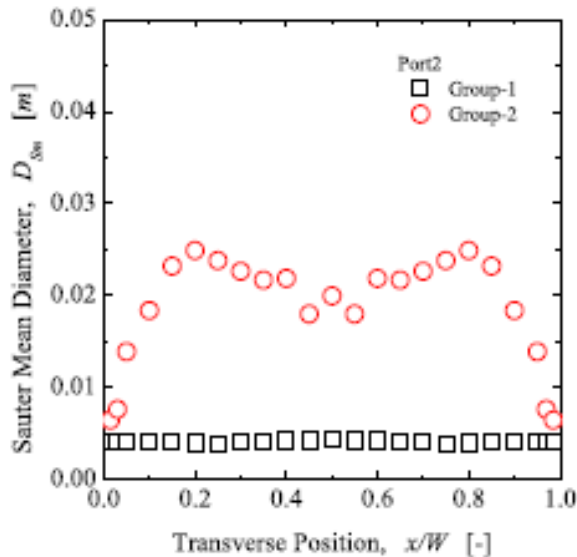
$$\langle j_{g,0} \rangle = 0.74 \text{ m/s}$$

$$\langle j_f \rangle = 0.942 \text{ m/s}$$



- Very Good prediction for overall trend of phase distribution in transverse direction.
- Underprediction of volume transfer from G-1 to G-2 bubbles by coalescence
- Underprediction of the increase in G-2 bubble sauter mean diameter.

## Two-Group cases: Run 5



### Summary :

- Hydrodynamics model gives satisfactory prediction in the trend of phase distribution
- Discrepancies were found in IATE model prediction in some cases , specially in intergroup transfer model
  - Need to establish a good set of flow transition experimental database for IATE benchmarking
  - Need to improve coalescence model

# Conclusion

- ◆ CFD Framework with three-field two-fluid model and two-group IATE was prepared
- ◆ Benchmarking simulations was carried out against uniform and non-uniform inlet test conditions.
- ◆ In general, Hydrodynamic models along with IATE showing satisfactory prediction of phase distribution
  - ◆ Bubble coalescence model will be further investigated

# Thank you!!

# Publications

- ◆ **Sharma, S. L.**, Ishii, M., Hibiki, T., Schlegel, J., Liu, Y., Buchanan, Jr., J.R., 2019, "Beyond Bubbly Two-Phase Flow Investigation using a CFD Three-Field Two-Fluid Model", International Journal of Multiphase Flow, 113, p.1-15.
- ◆ **Sharma, S. L.**, Hibiki, T., Ishii, M., Brooks, C. S., Schlegel, J., Liu, Y., Buchanan, Jr., J.R., 2017, "Turbulence-induced Bubble Collision Force Modeling and Validation in Adiabatic Two-phase Flow using CFD", Nuclear Engineering and Design Journal, 312, p.399-409.