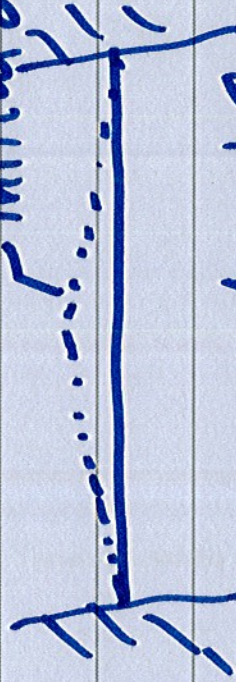


- responds only at the forcing freq
initial displacement



Free

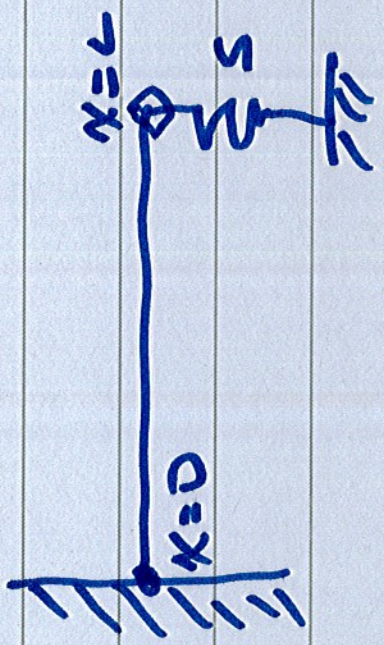
- responds at the
natural freqs

$$\sin kL = 0$$

$$k_n L = n\pi$$

$$y(x, t) = \sum_{n=2}^{\infty} \tilde{a}_n \sin k_n x e^{j\omega_n t}$$

2.5.4 Other b.c.'s



$$y(x,t) = - \sum_j A_j \sin k x e^{j \omega t}$$

$$-T \frac{dy}{dx} \Big|_{x=L} - s y \Big|_{x=L} = 0$$

$$\sum_j k T A_j \cos k L + \sum_j s A_j \sin k L = 0$$

$$- \frac{k s T}{s} = \tan(k L)$$

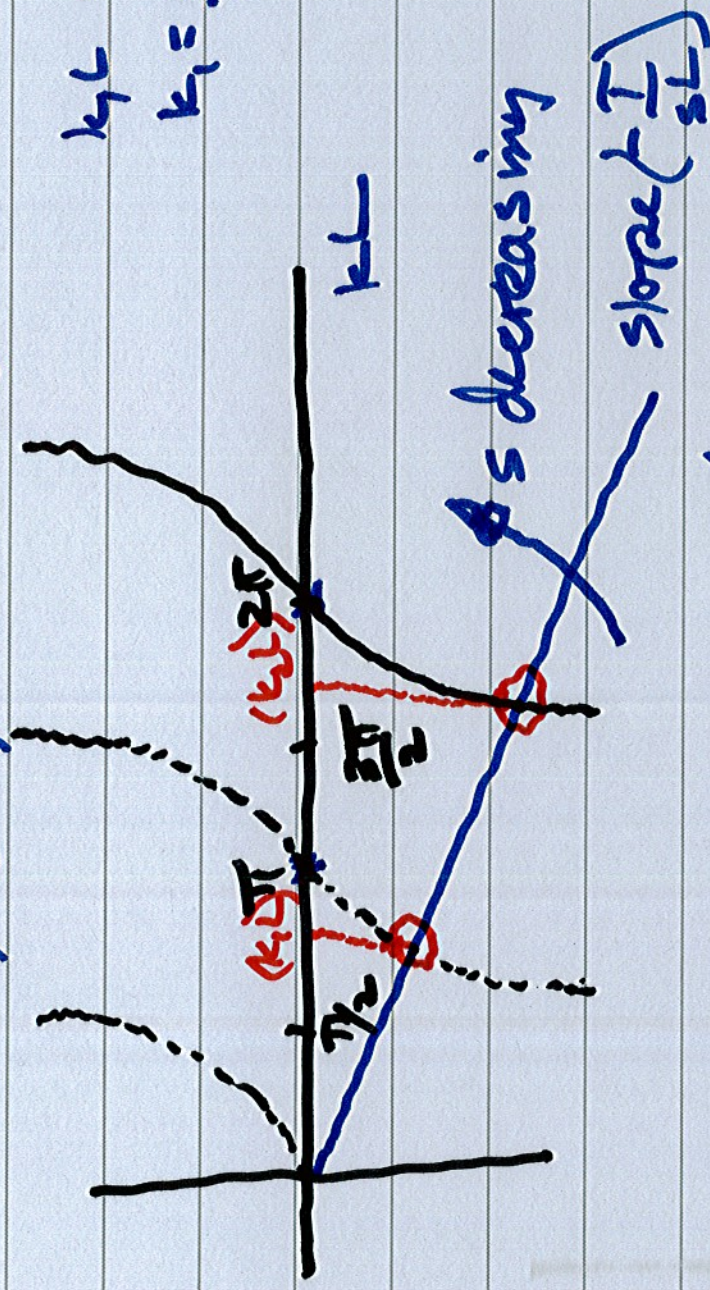
3

$$\left[-(k_{nl}) \left(\frac{T}{sL} \right) = \tan k_{nl} L \right]$$

characteristic eqn

4

$$-(k_{nL}) \left(\frac{I}{sL} \right) = \tau_{nL} k_{nL}$$



$$k_{nL} \rightarrow k_1$$

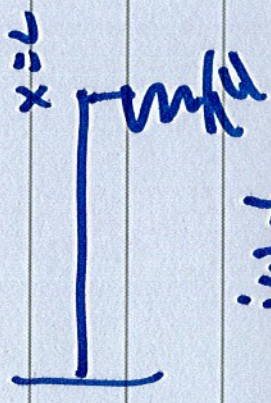
$$k_1 = \frac{u_1}{c} \rightarrow \omega_1$$

as $s \rightarrow \infty$ approach
 The fixed-fixed case

$$k_{nL} = n\pi$$

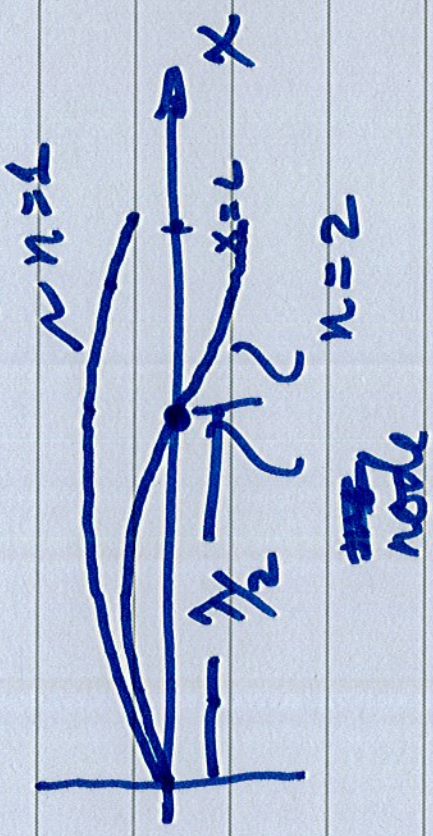
natural freqs are decreased
 when s is finite compared
 to fixed case

mode shape



$y_n(x,t) = \tilde{a}_n \sin(k_n x) e^{i\omega t}$ n th mode

k_1 is smaller than in the fixed-fixed case



$y(x,t) = \sum_{n=1}^{\infty} \tilde{a}_n \sin(k_n x) e^{i\omega t}$

Summary

- Derivation of a wave eqn

(modeling)

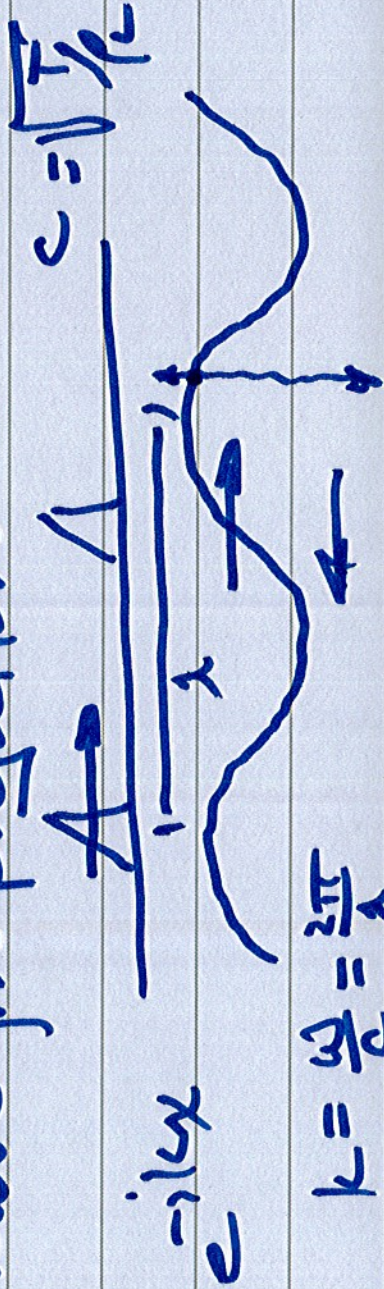
- restoring force

- com

- inertia

- stiffness

wave propagation

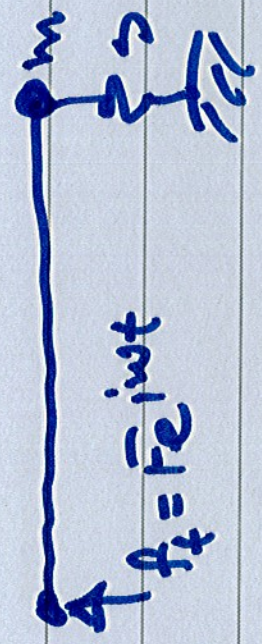


e^{-ikx}

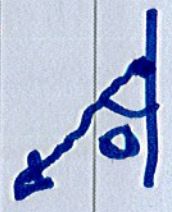
$$k = \frac{2\pi}{\lambda}$$

Boundary Conditions

- fixed
- stiffness
- mass
- force



$$T \sin \theta = ma$$



$$\sin \theta \approx -\frac{dy}{dx}$$

Forced Response $F e^{i\omega t}$

$$y(x,t) = A e^{i(\omega t - kx)} + B e^{i(\omega t + kx)}$$

linear system

Impedance $\frac{\text{Force}}{\text{velocity}}$

Z_{mo}

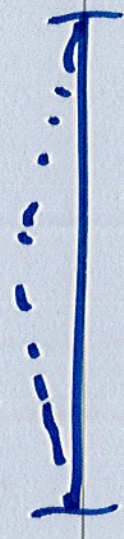
f.c characteristic
impedance

- standing wave

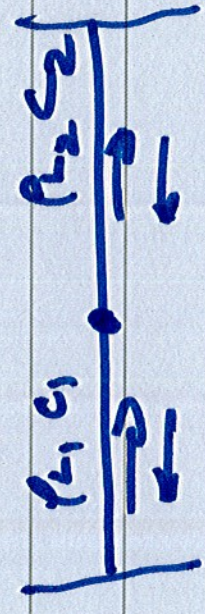
- propagating

↑ forced

↓ free



- natural freqs] characteristic
 - mode shapes] eqn



3. The Acoustic Wave Eqn + Simple soln's

5.1 → 5.13

3.1 Introduction

- sound - propagating, small amplitude
fluctuations in pressure () in an
elastic medium

- "ideal" acoustics

- assume that fluid is
inviscid, lossless +
adiabatic

- small amplitude fluctuations
"linear" acoustics

- wave propagation

- inertia 1.2 kg/m^3

- stiffness

Derive a wave eqn

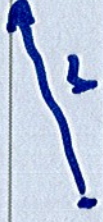
$P \neq u$ { - equation of state } - restoring force
- continuity

$P \neq u$ - momentum] - EOM

to eliminate

single equation in u

- 1-D solutions



- plane

- cylindrical waves]

- spherical waves]

r

- Acoustic Intensity

- specific Acoustic Impedance

- Decibel