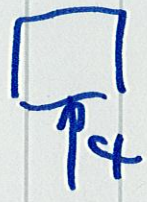
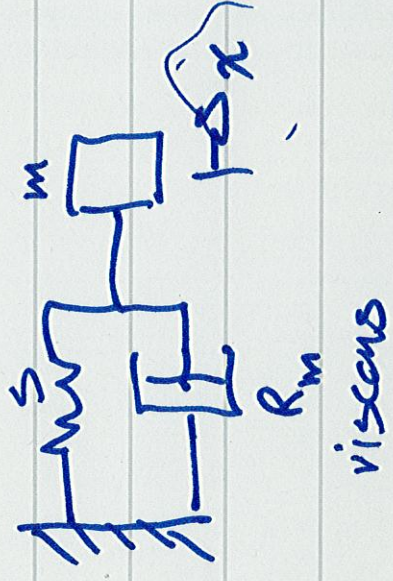


- governing eqns
- wave equation
- possible solutions
- boundary conditions



Restoring force

$$f = -sx - R_m \frac{dx}{dt}$$

BOM

$$f = m \frac{d^2x}{dt^2}$$

$$\ddot{x} = \ddot{A} e^{i\omega t}$$

$$\frac{d^2x}{dt^2} + \left(\frac{R_m}{m}\right) \frac{dx}{dt} + \omega_0^2 x = 0$$

$$\gamma = -\left(\frac{R_m}{m}\right) \pm \sqrt{\left(\frac{R_m}{m}\right)^2 - 4\omega_0^2}$$

2

$$\beta = \frac{R_m}{2m}$$

$$\gamma = -\beta \pm \sqrt{\beta^2 - \omega_0^2}$$

ω_0 = damped
natural freq

Usually - underdamped

$$\beta < \omega_0$$

$$\gamma = -\beta \pm j \sqrt{\omega_0^2 - \beta^2}$$

$$\omega_0^2 - \beta^2 = \omega_d^2 \quad \text{positive}$$

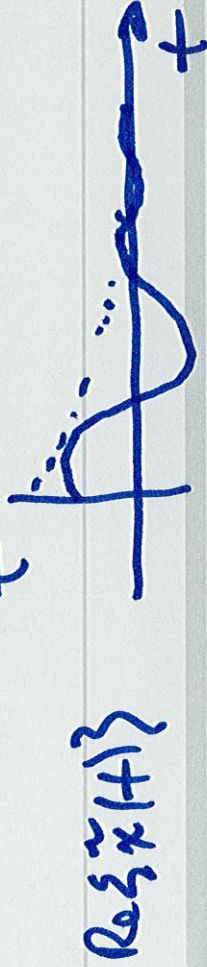
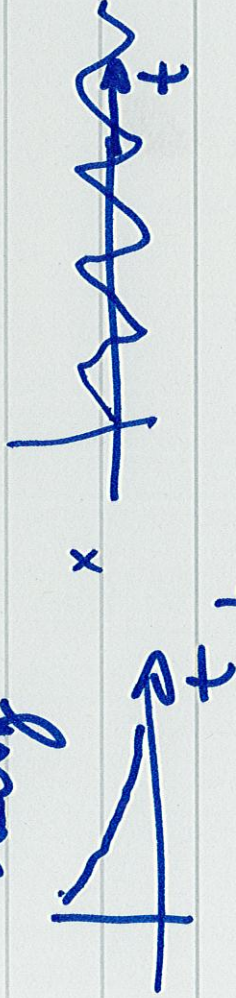
$$\delta = -\beta \pm j\omega_d$$

complete solution

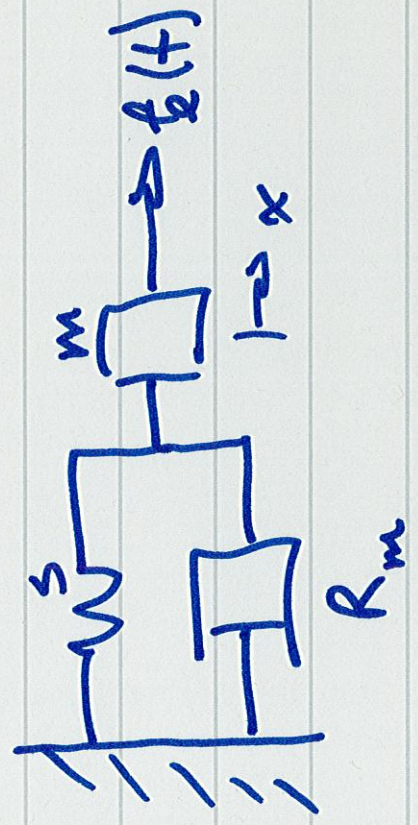
$$\begin{aligned} \tilde{x} &= \tilde{A}_1 e^{-\beta t} e^{+j\omega_d t} + \tilde{A}_2 e^{-\beta t} e^{-j\omega_d t} \\ &= e^{-\beta t} \left[\tilde{A}_1 e^{+j\omega_d t} + \tilde{A}_2 e^{-j\omega_d t} \right] \quad \text{at } t=0 \\ &\quad \tilde{x} = \tilde{A}_1 + \tilde{A}_2 \end{aligned}$$

$\underbrace{\hspace{10em}}_{\text{exponential decay}} \times \underbrace{\hspace{10em}}_{\text{oscillatory term}}$

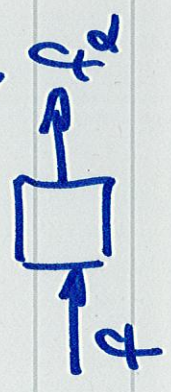
as $t \rightarrow \infty$
 $\tilde{x} \rightarrow 0$



1.3 Forced Oscillation



1.3.1 Governing Eqns



EOM $f + f_e = m \frac{d^2x}{dt^2}$ (1)

Respring force

$f = -sx - R_m \frac{dx}{dt}$ (2)

$$m \frac{d^2 x}{dt^2} + k_m \frac{dx}{dt} + s \cdot x = f_e(t) \quad (3)$$

inhomogeneous ODE

Example $f_e(t) = F \cos \omega t \quad t \geq 0$



Laplace Transform
→ algebraic solution in transform space.

$x \approx$ inhomogeneous eqn
 $x = x_{\text{transient}} + \underline{x_{\text{steady state}}}$

for real systems

since $R_n > 0$ always

transient soln decays to negligible
values as t increases

1.3.2 Steady State Solution

Assume complex form for the driving force

$$\tilde{f}_e = F e^{j\omega t} \quad \Re \{ \tilde{f}_e(t) \} = f \cos \omega t$$

driving frequency.
if $f = \text{real}$

$$\uparrow \omega \rightarrow \boxed{\phantom{\tilde{f}_e(t)}} \rightarrow \tilde{f}_e(t) = F e^{j\omega t}$$

Linear systems

- in a steady state
The response at the driving frequency.

ω - forcing freq
- you control

ω_0 - natural freq
- property of a system

Assumed solution

$$\underline{f_e = F e^{j\omega t} \quad \tilde{x} = \tilde{A} e^{j\omega t}}$$

sub into eqn (3)

$$- \omega^2 m \tilde{A} e^{j\omega t} + j\omega R_m \tilde{A} e^{j\omega t} + s \tilde{A} e^{j\omega t} = F e^{j\omega t}$$

$$\tilde{A} = \frac{F}{- \omega^2 m + j\omega R_m + s}$$

$$= \frac{F}{j\omega (R_m + j(\omega m - \frac{s}{\omega}))}$$

$$\tilde{x} = \tilde{A} e^{j\omega t}$$

$$= \frac{F e^{j\omega t}}{j\omega (R_m + j(\omega m - \frac{1}{\omega C}))}$$

steady-state response

Physical Displacement

$$x(t) = \operatorname{Re} \{ \tilde{x}(t) \}$$

$$\text{Velocity: } \tilde{u} = \frac{d\tilde{x}}{dt} = j\omega\tilde{x}$$

$$\text{Acceleration: } \tilde{a} = \frac{d^2\tilde{x}}{dt^2} = \frac{d\tilde{u}}{dt} = -\omega^2\tilde{x} = j\omega\tilde{u}$$

$$\tilde{u} = \frac{F_e j\omega t}{R_m + i(\omega m - (\frac{1}{\omega C}))}$$

progressive plane

$$|P| = \rho c$$

$$|u| = \frac{|P|}{\rho c} \quad \text{say } P = 1 \text{ Pa} \quad 94 \text{ dB}$$

$$= \underline{0.0025} \text{ m/s}$$

$$\omega |x| = |u|$$

$$|x| = \frac{|u|}{\omega} = \frac{2.5 \times 10^{-3}}{2\pi \times 10^3}$$

$$f = 1000 \text{ Hz} \quad \approx 0.5 \times 10^{-6} \text{ m}$$

1.3.3 Mechanical Impedance

Define $\tilde{Z}_m = \frac{\text{complex driving force}}{\text{complex velocity at the driven point}}$

$$= \frac{\tilde{F}_e}{\tilde{v}}$$

Damped SDOF

$$\tilde{Z}_m = \frac{\tilde{F}_e e^{j\omega t}}{\tilde{F}_e e^{j\omega t}} \frac{1}{R_m s + j(\omega m - \frac{s}{\omega})}$$

$$= R_m + j(\omega m - \frac{s}{\omega})$$

$$\tilde{Z}_m = R + jX$$

mechanical resistance mechanical reactance

$$R = \operatorname{Re} \{ \tilde{Z}_m \}$$

$$X = \operatorname{Im} \{ \tilde{Z}_m \}$$

$$\tilde{Z}_m = R_m + j(\omega_m - \frac{s}{\omega})$$