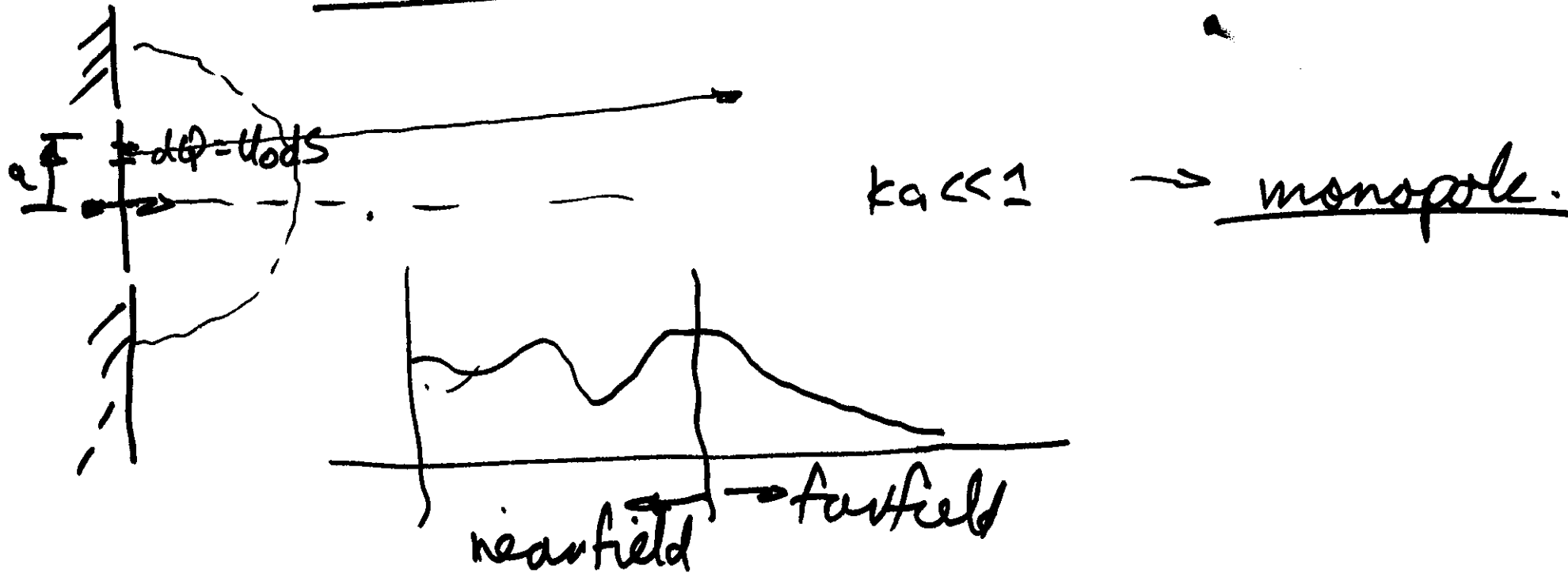
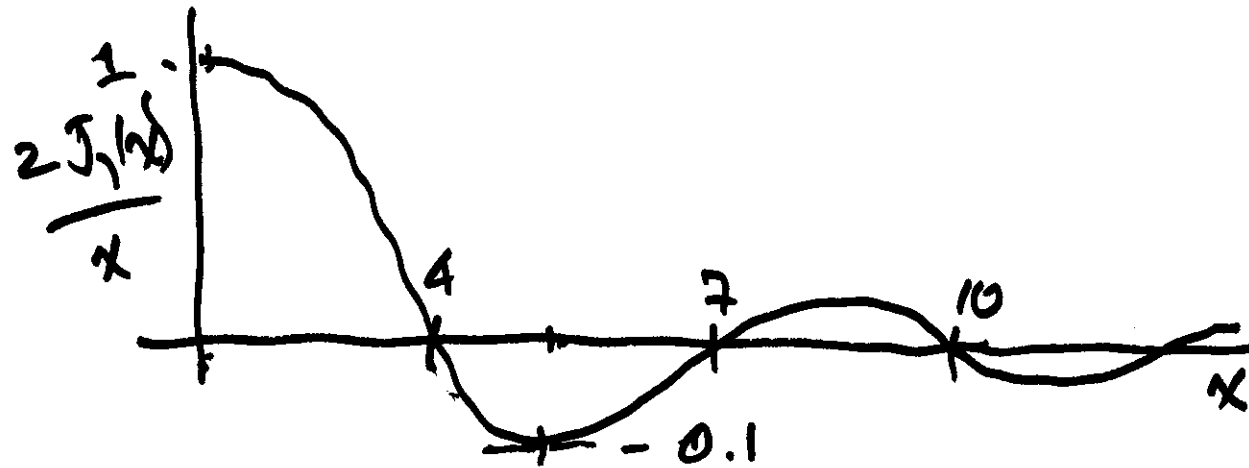
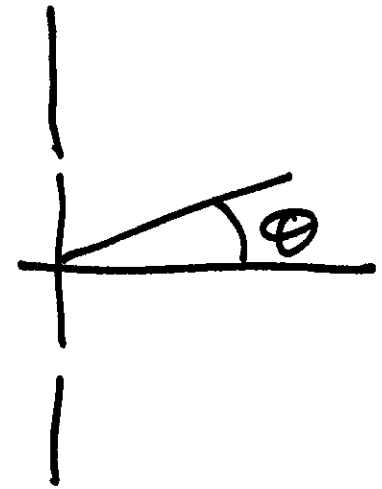


Extended sources.



far field

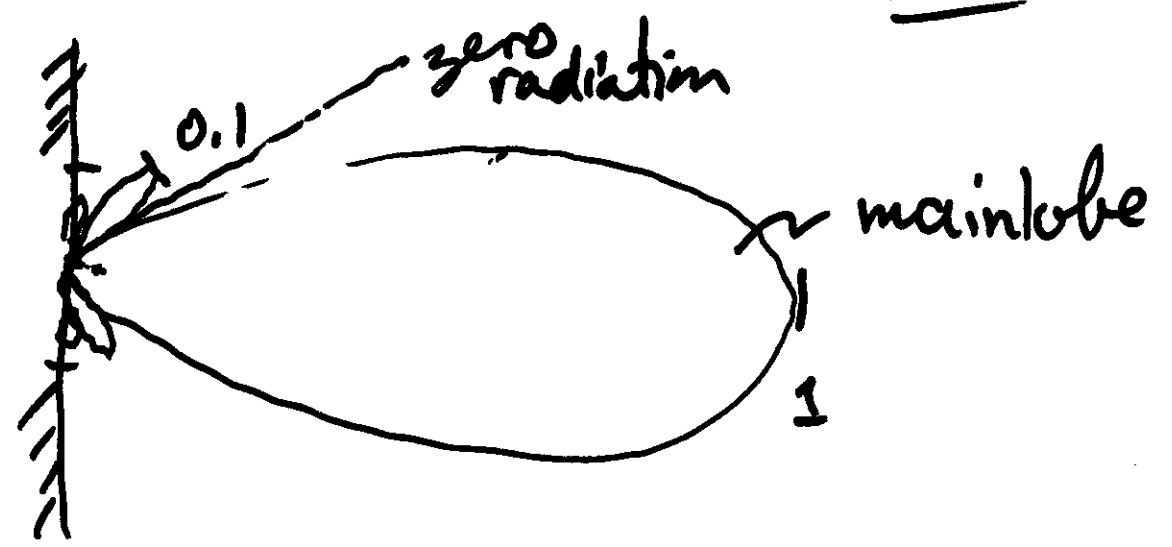
$$\vec{p}(r, \theta) = \underbrace{i \frac{f_0 e}{2} U_0 \left(\frac{a}{r} \right) e^{-ikr}}_{df} \left[2 \frac{J_1(ka \sin \theta)}{ka \sin \theta} \right]$$



$$x = \underline{ka \sin \theta}$$

$ka \ll 1$ $\left[\right] \approx 1$ for all angles
monopole-like radiation

$ka > 1$ we have the possibility of sidelobes
& ~~the~~ nulls.



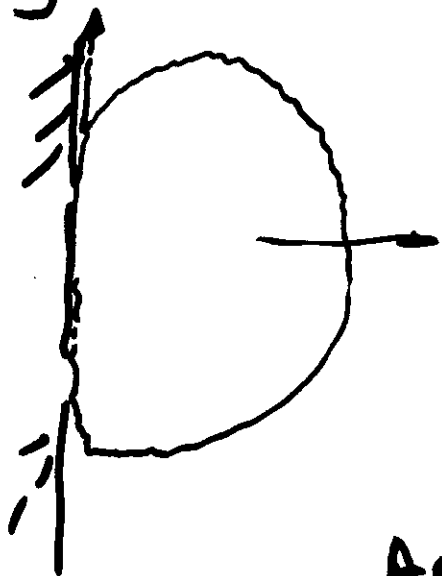
main lobe becomes narrower
as the frequency increases

$ka \gg 1$



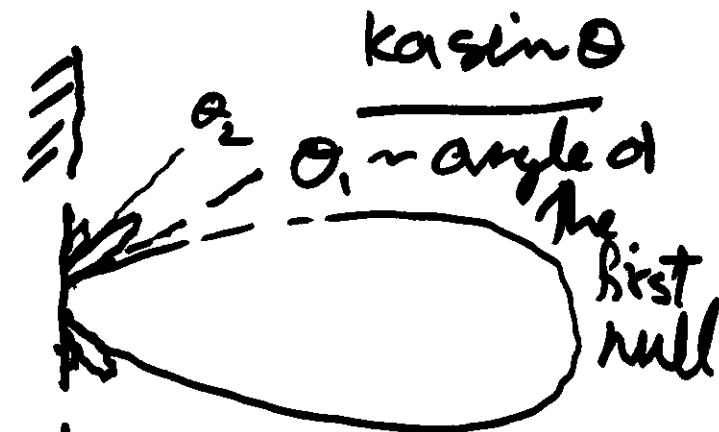
$ka \ll 1$

highly directional radiation



$ka = 4$

Appendix A5



$ka \gg 1$

First minimum happens when $ka \sin \theta_1 = j_{11}$

First zero of The Bessel func

$$J_1(j_{11}) = 0$$

$$J_1(j_{12}) = 0$$

$$J_1(j_{13}) = 0$$

5

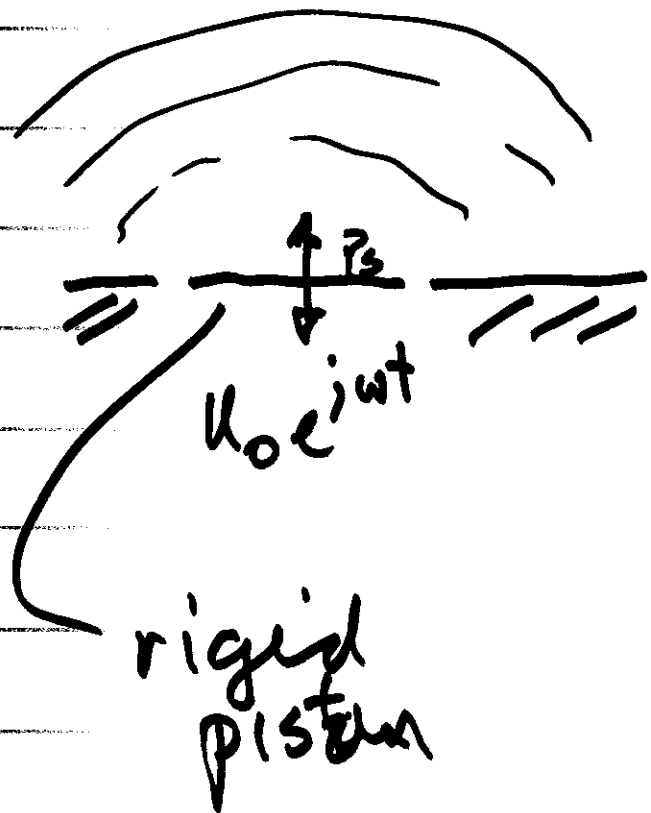
at high frequencies ($ka \gg 1$) j_{11} is
reached at progressively smaller angles

"beam width" becomes narrower

Public Address Systems

- many h.f. drivers to achieve uniform dispersion
- smaller number of low frequency drivers.

5.4.1.2

Radiation Impedance

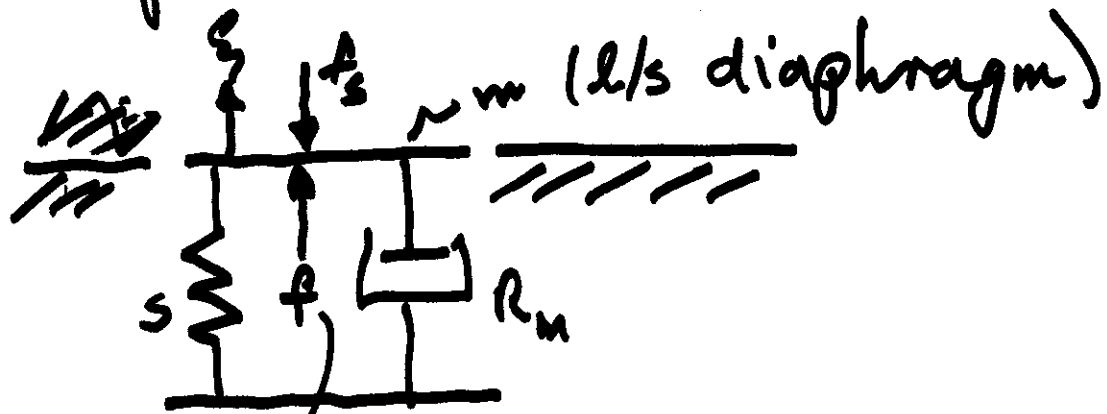
$$z_r = \frac{\text{force exerted on the radiator by the sound field}}{\text{source velocity}}$$

mechanical impedance

$$Z_r = \frac{\int_S P_{surface} dS}{\rho_0 c^2} \sim \text{total force acting on piston}$$

$\rho_0 c^2$ uniform rigid piston

Loudspeaker - finite internal mechanical impedance



force generated by the voice coil

Equation of Motion

$$f - f_s = m \frac{d^2 \xi}{dt^2} + R_m \frac{d\xi}{dt} + s\xi$$

assume harmonic motion $e^{j\omega t}$

$$u = j\omega \xi \quad \text{diaphragm velocity}$$

$$f - f_s = j\omega m u + R_m u + \frac{s}{j\omega} u$$

$$= \left[R_m + j \left(\omega m - \frac{s}{\omega} \right) \right] u$$

in vacuo mechanical
impedance of the l/s

$$f = f_s + z_m u$$

$$= (z_r + z_m) u$$

$$z_r = \frac{f_s}{u}$$

radiation impedance mechanical impedance

$$u = \frac{f}{z_r + z_m}$$

loudspeaker response is determined by both mechanical & radiation impedance

$$Z_r = R_r + jX_r$$

\uparrow \uparrow
 resistance reactance

Power Delivered by the piston to the sound field

$$\overline{P} = \frac{1}{T} \int_0^T \operatorname{Re}\{f_s\} \operatorname{Re}\{u\} dt$$

harmonic case $e^{j\omega t}$

$$\begin{aligned} \overline{P} &= \frac{1}{2} \operatorname{Re}\{f_s u^*\} \\ &= \frac{1}{2} \operatorname{Re}\{Z_r u u^*\} \end{aligned}$$

$$\begin{aligned} Z_r &= \frac{f_s}{u} \\ f_s &= Z_r u \end{aligned}$$

$$\bar{\Pi} = \frac{|u|^2}{2} \underbrace{\operatorname{Re} \{z_r\}}_{R_r}$$

$$= \underbrace{\frac{|u|^2}{2}}_{\substack{\text{m.s. velocity} \\ \text{of the piston}}} R_r$$

Circular Piston

$$R_r = \pi a^2 \rho_0 c R_1(2ka)$$

$$X_r = \pi a^2 \rho_0 c X_1(2ka)$$

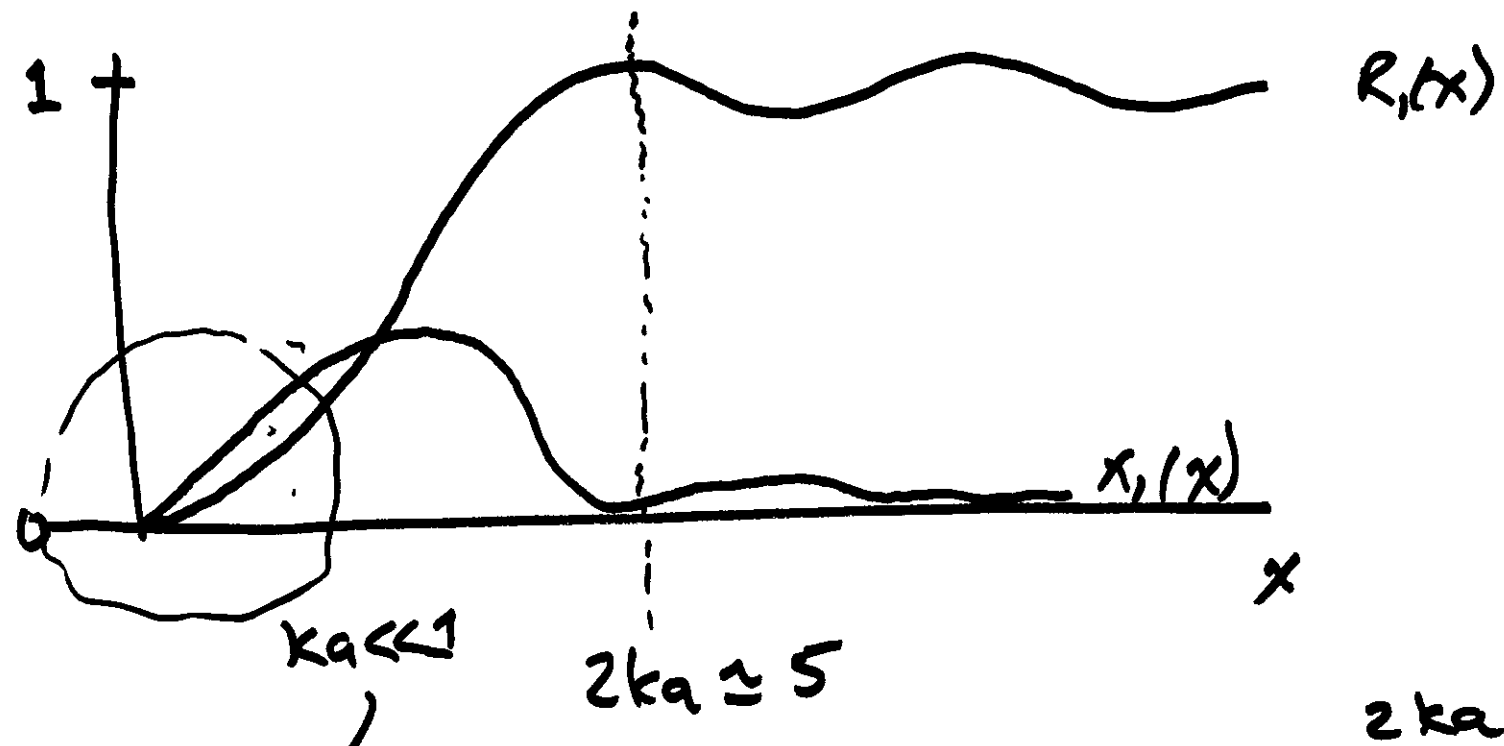
]

circular rigid
piston in an
infinite baffle.

piston radius of a

$$R_1(x) = 1 - \frac{2 J_1(x)}{x}$$

$$X_1(x) = \frac{4}{\pi} \left[\frac{x}{3} - \frac{x^3}{3^2 \cdot 5} + \frac{x^5}{3^2 \cdot 5^2 \cdot 7} - \dots \right]$$



$$\left(\begin{array}{l} R_r(x) \\ X_r(x) \end{array} \right)$$

small piston case

$$R_r \approx \frac{\pi a^2}{2} \rho_0 c (ka)^2$$

$$X_r \approx \frac{\pi a^3}{2} \rho_0 \frac{8}{3\pi} (ka) \rightarrow \text{mass-like impedance}$$