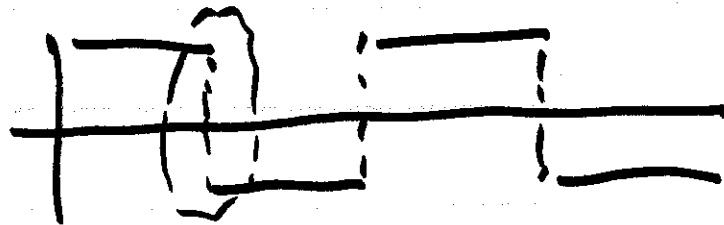
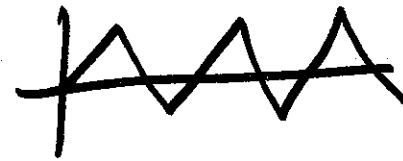
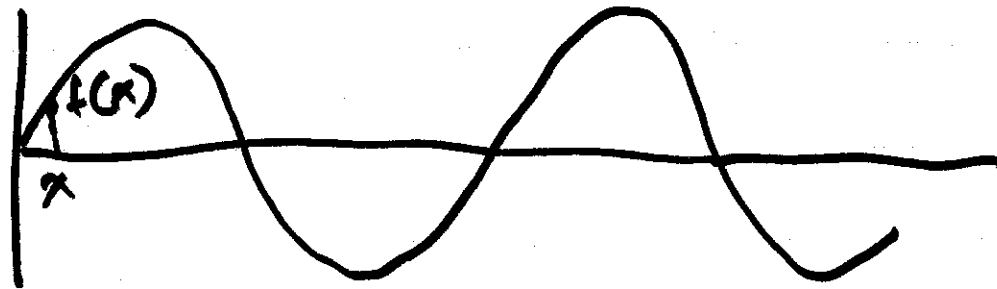


Webpage: engineering.purdue.edu/ME513

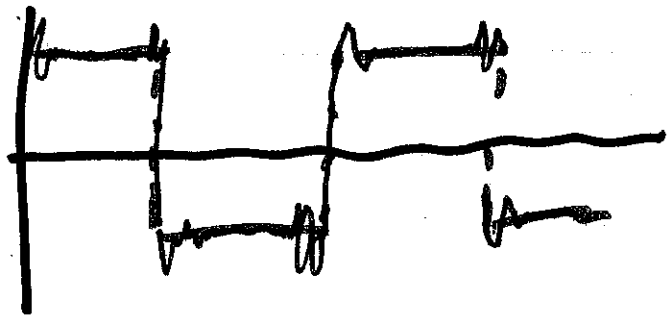
Single-valued function

One value of function for each value of independent variable



Gibb's Phenomenon

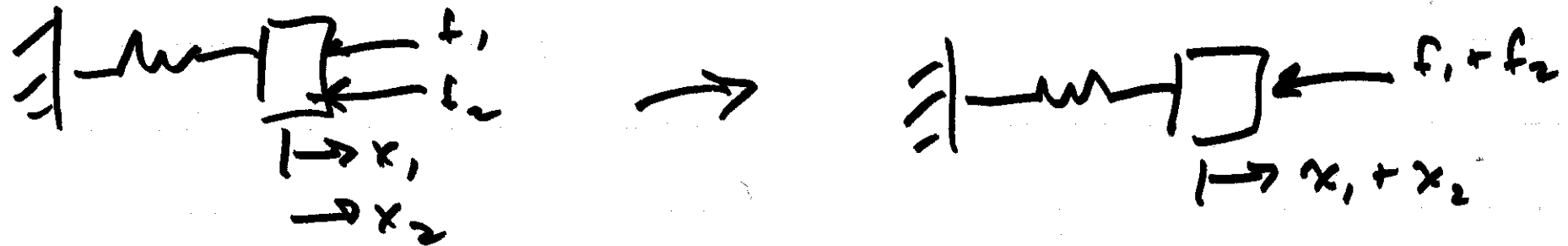
effect of discontinuity on Fourier series fit



Superposition

"Linear Acoustics"
- small oscillations

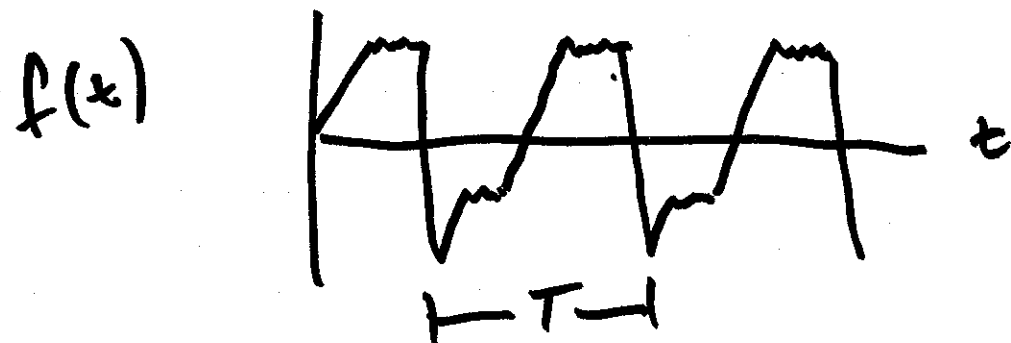
- Add functions and get results
 - sum of individual functions



- allows for frequency-dependent analysis

Fourier Analysis

Decompose periodic functions
into sinusoidal components



$$f(t) = \frac{1}{2} A_0 + A_1 \cos \omega_1 t + A_2 \cos \omega_2 t + \dots \\ + B_1 \sin \omega_1 t + B_2 \sin \omega_2 t + \dots$$

$$\omega_1 = \frac{2\pi}{T} \quad \omega_2 = 2\omega_1$$

$$A_n = \frac{2}{T} \int_0^T f(t) \cos \omega_n t \, dt \quad \omega_n = \frac{2\pi n}{T}$$

$$B_n = \quad \quad \quad \int_0^T f(t) \sin \omega_n t \, dt$$

Summary

• physical model



represent in mathematical form

- To oscillate
 - mass
 - stiffness

Approach to problem solving

① Governing equations

- force

- motion $(f=ma)$

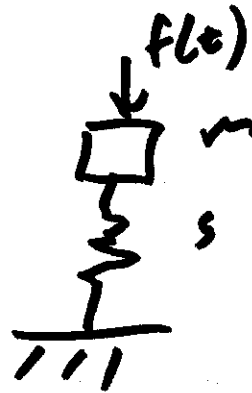
② Combine \rightarrow 2nd ODE

③ Identify possible solutions
in a convenient form

$$x = A e^{j\omega t}$$

④ Find the solutions that satisfy boundary conditions

Free response
- respond at natural frequency



$$\omega_0 = \sqrt{\frac{s}{m}}$$

Forced response
 $f(t) = Fe^{i\omega t}$ - responds at driving frequency

Resonance

- system is driven at a natural frequency
- mechanical impedance

$$\tilde{Z}_m = R_m + j(\omega m - \frac{s}{\omega})$$

- mass-like
- stiffness-like
- resistive

$$\text{Im}\{\tilde{Z}_m\} = 0$$

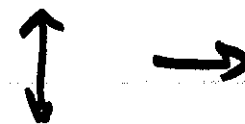
$$\tilde{Z}_m = \frac{f_c}{\omega}$$

2. Vibrating String

- vibration of extended systems
- derive a wave equation



- transverse wave propagation
 - normal to direction of propagation



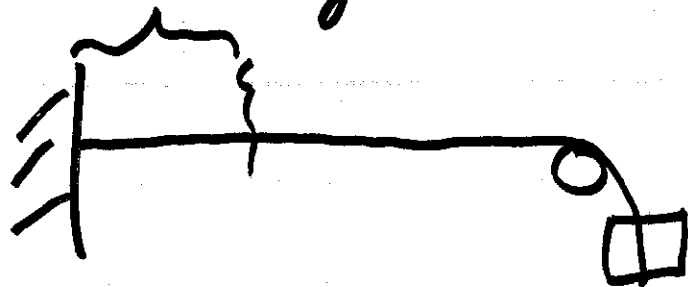
- phase speed
+ particle velocity

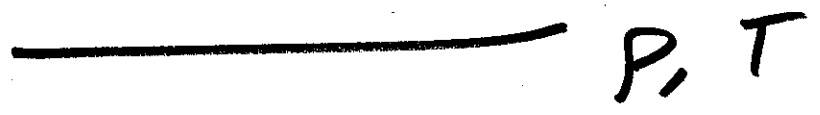
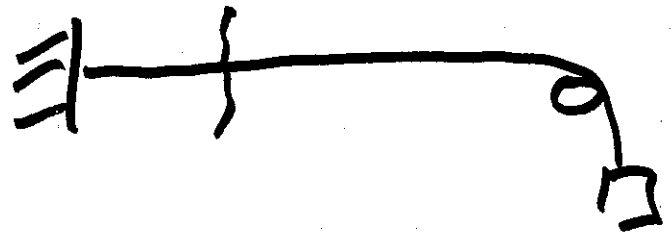
- wave number

- application of various BC's
- characteristic impedance
- standing waves & propagating waves

2.1 Derivation of a wave equation

governing
vibration of transverse
tensioned
string



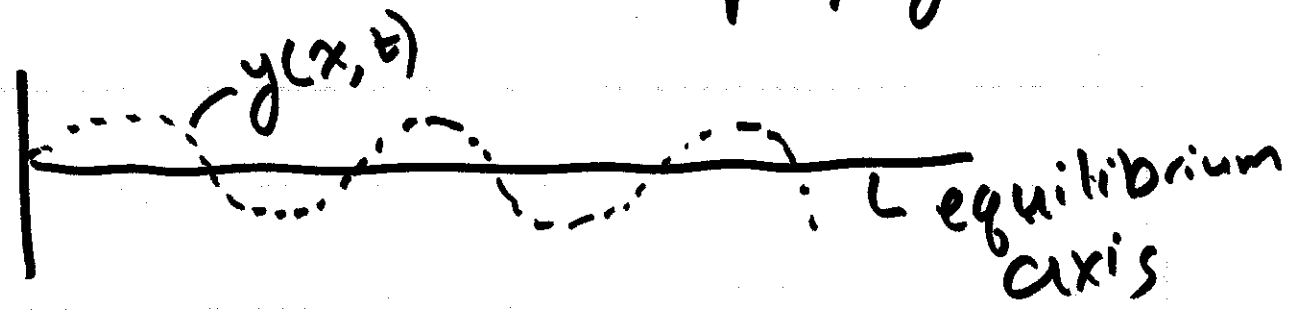


- restoring force
- dynamic equation



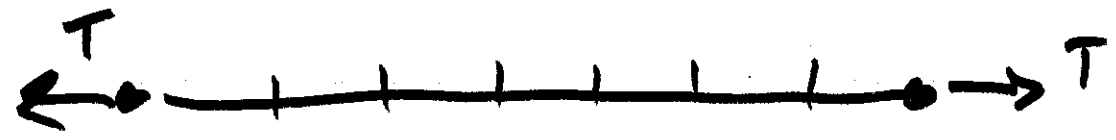
Transverse vibration

- displacement \perp to the equilibrium axis + the direction of propagation

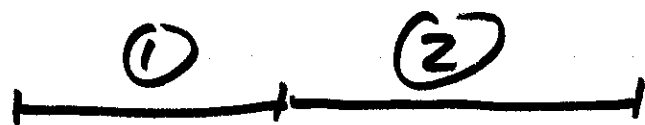


$y(x,t)$ - instantaneous transverse displacement

Assumptions:

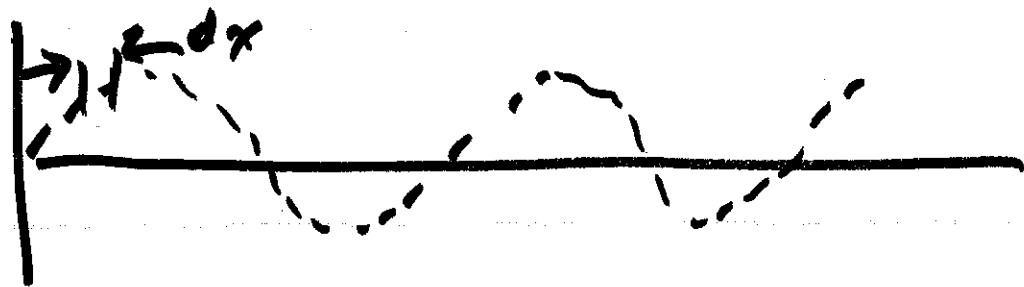
1. Uniform static tension
A horizontal line representing a string with several vertical tick marks. At each end of the line, there is a dot with an arrow pointing outwards, labeled with the letter 'T', representing tension.
2. Small amplitude response
- linear approximation
3. No losses in any segment

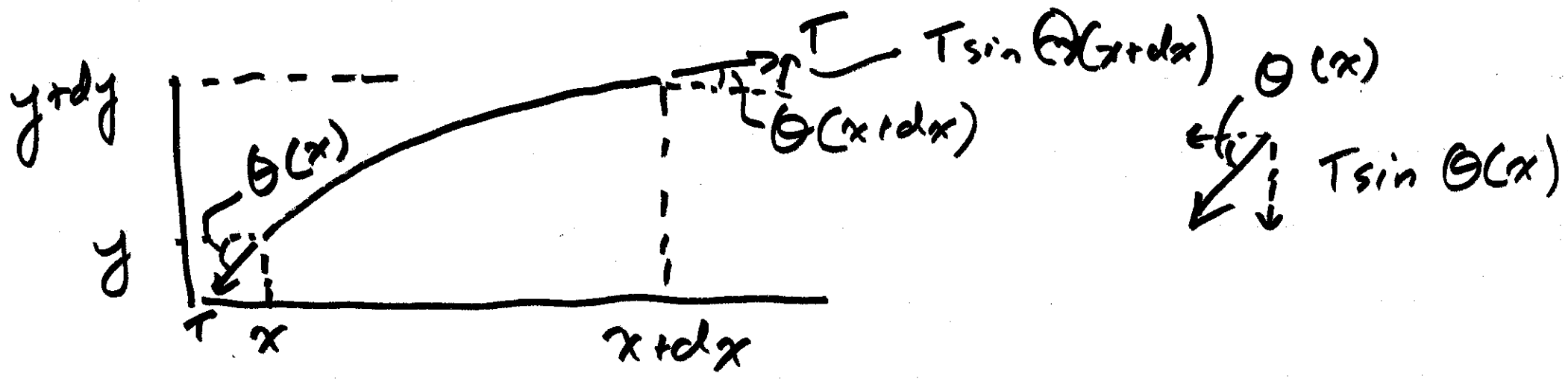
4. Uniform mass/length



5. Gravity force acting on the string is negligible

2.1.1 Restoring force equation





Net vertical force (in the y-direction)

$$df_y = T \sin \theta(x+dx) - T \sin \theta(x)$$

(15)

Linear $\sin(x)$ via Taylor's series expansion
 $f(x+dx) = f(x) + \left(\frac{df}{dx}\right) dx + \frac{1}{2} \left(\frac{d^2f}{dx^2}\right) (dx)^2 + \dots$

Small amplitude \rightarrow linear approximation
 so series can be truncated

$$\sin \theta(x+dx) \approx \sin \theta(x) + \frac{d[\sin \theta(x)]}{dx} dx$$

$$df_y = T \cancel{\sin \theta(x)} + T \frac{d[\sin \theta(x)]}{dx} dx$$

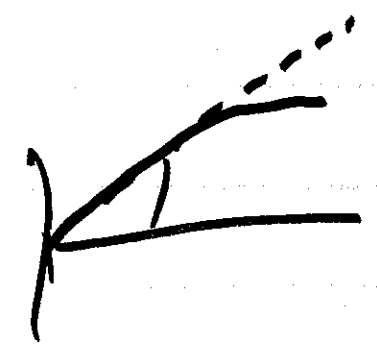
$$- T \cancel{\sin \theta(x)}$$

$$df_y = T \frac{d[\sin \theta(x)]}{dx} dx$$

Small response $\rightarrow \theta$ is always small

when θ is small

$$\sin \theta \approx \tan \theta \approx \frac{dy(x)}{dx} \quad \text{slope}$$



$$df_y \approx T \frac{d\left[\frac{dy}{dx}\right]}{dx} dx$$

$$\boxed{df_y = T \frac{\partial^2 y}{\partial x^2} dx} \quad (1)$$

curvature of the string

$y(x, t)$

Curvature

