

## LAB 6: SYSTEM IDENTIFICATION OF A SECOND ORDER SYSTEM

## Overview Of ID Techniques

10/06/2008

Four techniques are used to identify the system parameters  $\omega_n$  and  $\zeta$  for the 2<sup>nd</sup> order system studied in this experiment. These techniques fall into two categories: (1) Time domain techniques and (2) Frequency domain techniques.

## Time Domain Techniques

Both the time domain techniques require a step input to be given to the system. This is done in this lab by using a square wave from the function generator as input. The square wave can be thought of as a series of step inputs. The output is recorded using SYSTEMID.VI.

## 1. Percentage Overshoot Method

**Input:** Step Input

**Output:** As shown in Figure 1:

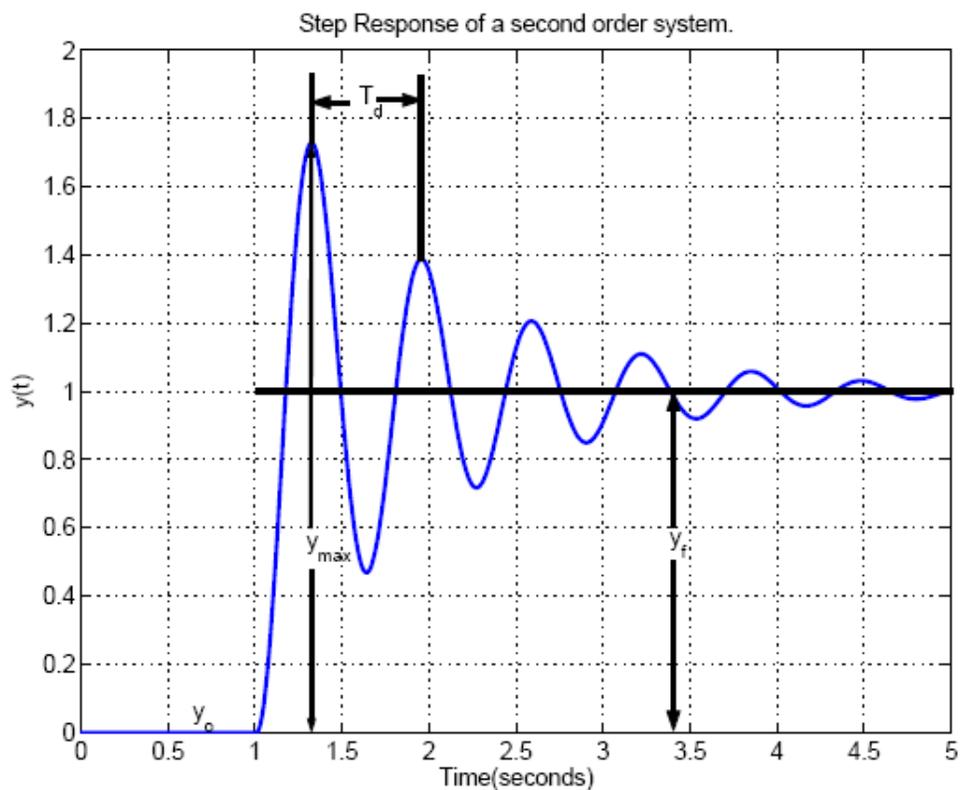


Figure 1: Step response of a second order system for percentage overshoot calculations  
Numbers are for illustrative purposes only.  $y_0$  is not equal to 0 in this experiment.

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**Procedure:**

- The analytical expression for the above step response ( $y(t)$ ) is given by

$$y(t) = y_0 + (y_f - y_0) \left[ 1 - \frac{\exp(-\zeta \omega_n t)}{\sqrt{1 - \zeta^2}} \cdot \sin(\omega_d t + \phi) \right]$$

where  $\phi = \tan^{-1} \left( \frac{\sqrt{1 - \zeta^2}}{\zeta} \right)$ . A version of this equation with  $y_0=0$  is available in equation

16 in page 6-12 of the course packet.

- This can be algebraically manipulated to give:

$$\frac{y_{\max} - y_f}{y_f - y_0} = \exp \left( - \frac{\zeta \pi}{\sqrt{1 - \zeta^2}} \right)$$

where  $y_{\max}$  is the amplitude of the first peak in the oscillations,  $y_f$  is the steady state value of the output and  $y_0$  is the initial value of the output. All these are marked in Figure 1.

The above equation can be rearranged to solve for  $\zeta$ .

- If the time period of the oscillations is  $T_d$ , then the *damped natural frequency* is given by  $\omega_d = \frac{2\pi}{T_d}$ . From the damped natural frequency and damping ratio, the undamped

natural frequency can be calculated using  $\omega_n = \frac{\omega_d}{\sqrt{1 - \zeta^2}}$ .

**Limitations:** For this method to work, the output must show oscillations. This only happens when the system is underdamped i.e. only if  $\zeta < 1$ .

**2. Log Decrement Method**

**Input:** Step Input

**Output:** As shown in Figure 2

**Procedure:**

- Denote the first peak in the output as  $y_i$  (which is the same as  $y_{\max}$  for the percentage overshoot method). The peaks after the first peak are labeled in order as  $y_{i+1}$ ,  $y_{i+2}$ , ...,  $y_{i+n}$ , ...

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- Define  $\Delta = \frac{1}{n} \ln \left[ \frac{y_i - y_f}{y_{i+n} - y_f} \right]$  where  $n$  refers to the  $n^{\text{th}}$  peak after the first peak. Hence if  $n = 2$ , use  $y_{i+n} = y_{i+2}$  in the denominator.

With this definition, it can be shown that:

$$\zeta = \frac{\Delta \sqrt{1 - \zeta^2}}{2\pi}$$

The above equation can be arranged to solve for  $\zeta$ .

- Then, following the same procedure as in the percentage overshoot method, the undamped natural frequency  $\omega_n$  can be evaluated.

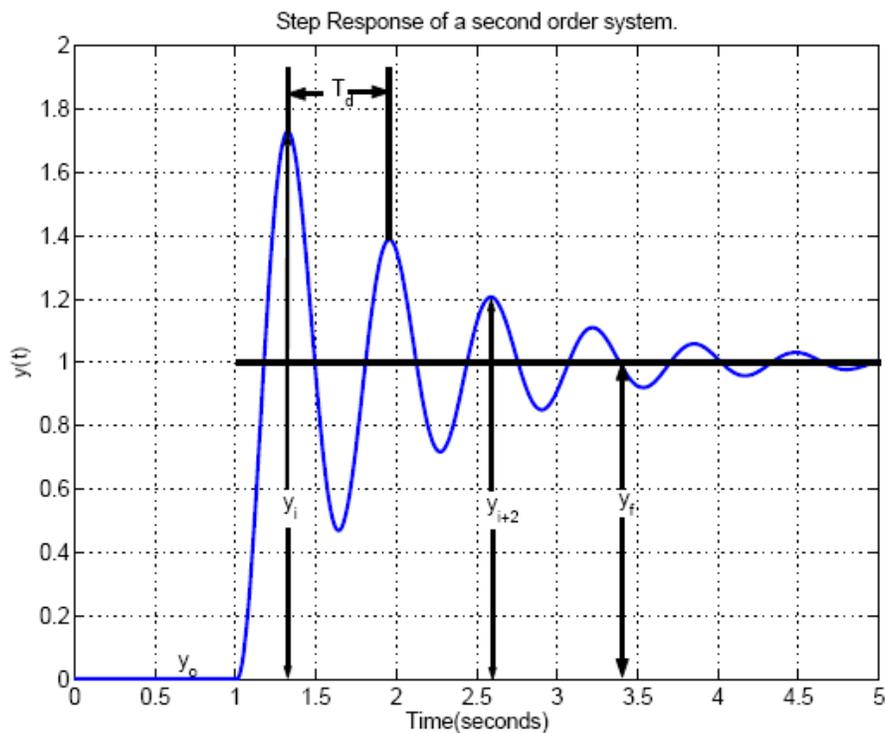


Figure 2: Step response of a second order system for log decrement calculations. Numbers are for illustrative purposes only.  $y_0$  is not equal to 0 in this experiment.

**Limitations:** Same as for the percentage overshoot method.

### Frequency Domain Techniques

Both the frequency domain techniques require the generation of the Bode plot for the system. In this lab, this is generated and recorded using BODE.VI.

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## 1. Half-Power Method

## Procedure:

The Bode plot must be converted from log – log scale to a linear scale for both the x and y axis. The magnitude plot in the Bode plot, plotted on a linear scale is shown in Figure 3.

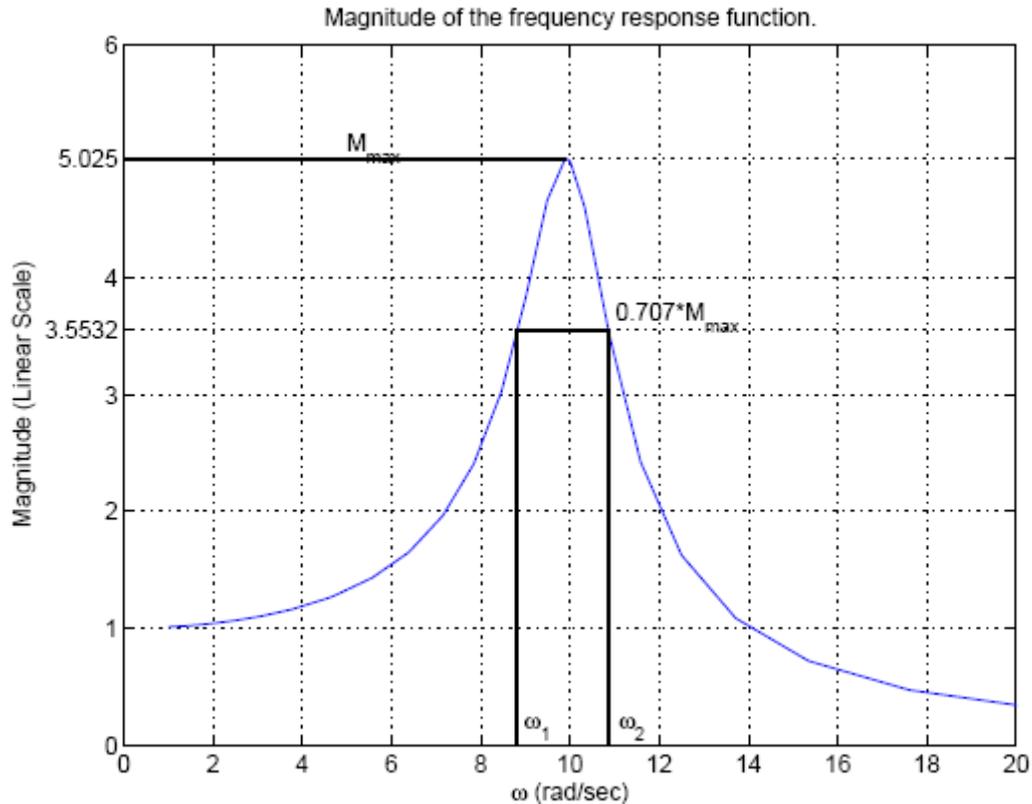


Figure 3: FRF magnitude plot in linear scale.  
Numbers are for illustrative purposes only.

- In order to evaluate  $\omega_n$ , refer to the phase plot of the Bode plot. The undamped natural frequency,  $\omega_n$ , is the frequency at which the phase  $\phi = -90^\circ$ . Alternately, a Lissajous figure can be used in the lab to evaluate  $\omega_n$ .

- Let the peak value of the frequency response function be denoted  $M_{max}$ . This peak occurs at a frequency called the *resonant natural frequency*, denoted by  $\omega_r$ .

- Calculate  $\frac{M_{max}}{\sqrt{2}} = 0.707M_{max}$ . Now find the two frequencies,  $\omega_1$  and  $\omega_2$ , at which the

FRF attains this magnitude. Then, the damping ratio can be calculated using  $\zeta = \frac{\omega_2 - \omega_1}{2\omega_n}$

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**Limitations:** For this method to work, the FRF magnitude plot must show a peak. This only happens when  $\zeta < \frac{1}{\sqrt{2}}$ .

## 2. Slope of Phase method

### Procedure:

- Convert the phase plot of the Bode plot from log-log scale to linear scale in both phase and frequency. This is shown in Figure 4.
- Express phase in radians and frequency in radians/second.
- Evaluate the undamped natural frequency,  $\omega_n$ , in the same manner as for the half-power method.

- The damping ratio is evaluated using  $\zeta = \frac{-1}{\omega_n} \left\{ \left[ \frac{d\phi}{d\omega} \right]_{\omega=\omega_n} \right\}^{-1}$  where  $\left[ \frac{d\phi}{d\omega} \right]_{\omega=\omega_n}$  is the slope of the phase plot at  $\omega = \omega_n$  or at  $\phi = -\pi/2$  radians.

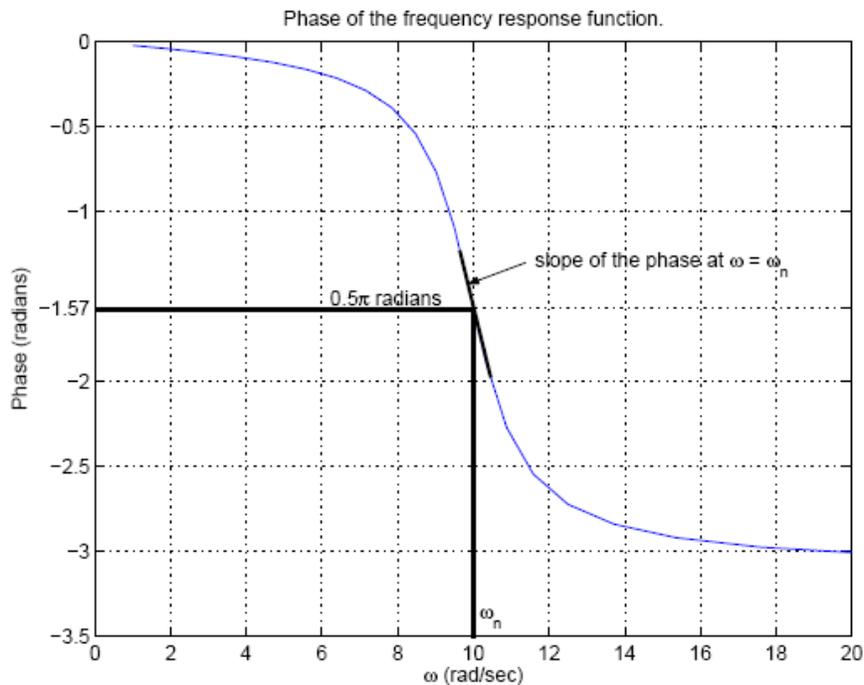


Figure 4: FRF phase vs. frequency in linear scale  
Numbers are for illustrative purposes only.

**Limitations:** None