# **Numerical Analysis of Mist-Cooled High Power Components in Cabinets**

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## **Objective**

The present work quantifies the potential for thermal management of a sealed cabinet using an evaporating mist introduced upstream of the high-power electronic components. The effect of droplet size and the mist loading fraction on the heat sink temperature reduction is computed and parametrically analyzed.

# **Approach**

$$
\frac{\partial}{\partial x_i}(\rho u_i)=S_m
$$

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$$
\frac{\partial}{\partial x_i} \left( \rho u_i u_j \right) = -\frac{\partial p}{\partial x_i} + \frac{\partial \tau_{ij}}{\partial x_j} + F_i
$$
\n
$$
\frac{du_P}{dt} = \frac{18\mu}{\rho_p D_p^2} \frac{C_D \operatorname{Re}_D}{24} + F_{other}
$$

$$
\frac{\partial}{\partial x_i} \left( \rho c_p u_i T \right) = \frac{\partial}{\partial x_i} \left( k_i \frac{\partial T}{\partial x_i} \right) + \dot{q}^m \qquad m_P c_{p,P} \frac{dT_P}{dt} = h A_P (T - T_P) + h_{fg} \frac{dm_P}{dt}
$$

$$
\frac{\partial}{\partial x_i}(\rho u_i) = S_m \qquad \frac{\partial}{\partial t} = -h_m(\rho_{v,s} - \rho_v)A_P
$$
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$$
\frac{\partial}{\partial x_i}(\rho u_i u_j) = -\frac{\partial p}{\partial x_i} + \frac{\partial \tau_{ij}}{\partial x_j} + F_i \qquad \frac{du_P}{dt} = \frac{18\mu}{\rho_p D_p^2} \frac{C_D \text{ Re}_D}{24} + F_{other} \qquad \text{m:\n
$$
\frac{\partial}{\partial x_i}(\rho c_\rho u_i T) = \frac{\partial}{\partial x_i} \left(k_i \frac{\partial T}{\partial x_i}\right) + \dot{q}^m \qquad m_P c_{p,P} \frac{dT_P}{dt} = hA_P (T - T_P) + h_{fg} \frac{dm_P}{dt} \text{ eq}
$$
\n
$$
\frac{\partial}{\partial x_i}(\rho u_i m_v) = \frac{\partial}{\partial x_i} \left(\rho D_{AB} \frac{\partial m_v}{\partial x_i}\right) + S_m \qquad Sh = \frac{h_m D_p}{D_{AB}} = 2.0 + 0.6 \text{ Re}_D^{0.5} \text{ Sc}^{1/3}
$$
\n
$$
\text{Re}_D = \frac{\rho D_p |u_p - u|}{\mu} \qquad Nu = \frac{hD_p}{k} = 2.0 + 0.6 \text{ Re}_D^{0.5} \text{ Pr}^{1/3}
$$
$$

### **Impact**

- Elimination of hot/cold aisles
- Reduced operating expenditure by avoiding wasteful cooling of room air and pumping air over long distances
- Ergonomic benefits (reduced acoustic noise)
- Increased thermal densities through shelf-level cooling and increased fan speeds
- Prevention of pollutants and dust from entering

## Disperse phase<sup>racks</sup>

 $\frac{dm_P}{dt} = -h_m(\rho_{v,s} - \rho_v)A_P$ 

**Continuous phase <b>Disperse phase** Disperse phase Discrete Phase Model (DPM) in FluentAir is treated like continuum phase, droplets as the dispersed phase. The interactions between the two phase is accounted by mass, momentum and heat exchange (source/sink terms in Navier-Stokes  $lm_P$  equations for air flow)

### **Interactions between two phases**

$$
F_i = \sum \left( \frac{18\mu C_D \text{ Re}_D}{24\rho_p D_p^2} \left( u_{P,i} - u_i \right) + F_{other,i} \right) \dot{m}_P \Delta t
$$
  

$$
S_m = \frac{\Delta m_P}{m_{P0}} \frac{\dot{m}_{P0}}{dV} \qquad \dot{q}'' = \left[ \frac{\overline{m}_P}{m_{P0}} c_{p,P} \Delta T_P + \frac{\Delta m_P}{m_{P0}} \left( -h_{fg} + \int_{T_{ref}}^{T_p} c_{p,v} dT \right) \right] \frac{\dot{m}_{P0}}{dV}
$$



**Large droplets of mist from** 

**atomizer directed into circuit packs**



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