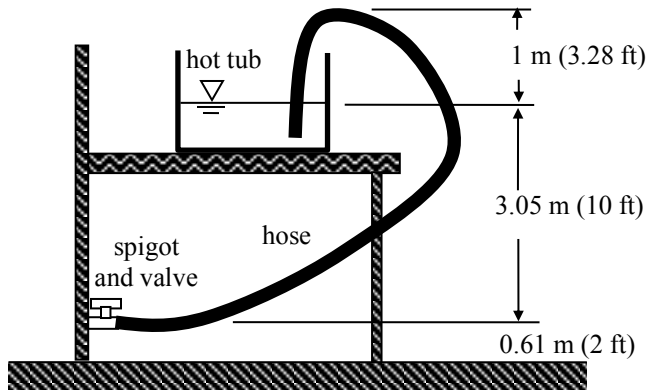


A hot tub sits on a deck as shown in the figure below. A homeowner plans to fill the hot tub with water from a 1.91 cm (0.75 in.) diameter, 7.62 m (25 ft) length of old garden hose attached to an outdoor spigot (aka faucet) located underneath the deck. The hose has an internal roughness of 0.5 mm ( $1.97 \cdot 10^{-2}$  in.) and the gage water pressure just upstream of the spigot valve is 379 kPa (55 psig).

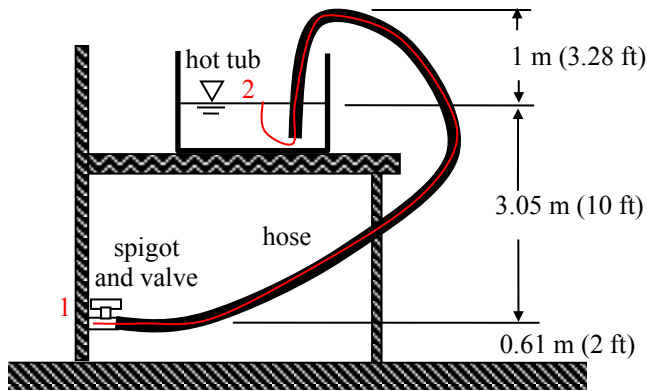


water density =  $1000 \text{ kg/m}^3$  ( $62.4 \text{ lb}_m/\text{ft}^3$ )  
 water kinematic viscosity =  $1.00 \cdot 10^{-6} \text{ m}^2/\text{s}$   
 ( $1.05 \cdot 10^{-5} \text{ ft}^2/\text{s}$ )

The minor losses due to the bends in the hose are much smaller than the minor loss due to the valve, which has a loss coefficient of 2.

Determine the volumetric flow rate of water into the hot tub. Clearly state and justify all significant assumptions.

SOLUTION:



Apply the Extended Bernoulli Equation from 1 to 2.

$$\left( \frac{p}{\rho g} + \alpha \frac{\bar{v}^2}{2g} + z \right)_2 = \left( \frac{p}{\rho g} + \alpha \frac{\bar{v}^2}{2g} + z \right)_1 - H_L + H_S \quad (1)$$

where

$$p_{1,g} = 3.79 \cdot 10^3 \text{ Pa} \text{ and } p_{2,g} = 0 \quad (2)$$

$$\bar{V}_1 = \bar{V}_h \text{ and } \bar{V}_2 \approx 0 \quad (3)$$

$$z_2 - z_1 = \Delta z = 3.05 \text{ m} \quad (4)$$

$$H_L = \left[ f \left( \frac{L}{D} \right) + K_{\text{valve}} + K_{\text{exit}} + K_{\text{bends}} \right] \frac{\bar{V}_h^2}{2g} \quad (5)$$

$$H_S = 0 \quad (6)$$

Assume the flow in the hose is fully turbulent so that,

$$\alpha_1 \approx 1, \quad (7)$$

and the friction factor is solely a function of the relative roughness,

$$\frac{e}{D} = \frac{0.5 \cdot 10^{-3} \text{ m}}{1.91 \cdot 10^{-2} \text{ m}} = 2.62 \cdot 10^{-2} \quad (8)$$

Using the Moody diagram, the friction factor is,

$$f \approx 0.054 \quad (9)$$

Substitute into Eq. (1) and solve for the average water speed in the hose.

$$\Delta z = \frac{p_{1,g}}{\rho g} + \frac{\bar{V}_h^2}{2g} - \left[ f \left( \frac{L}{D} \right) + K_{\text{valve}} + K_{\text{exit}} + K_{\text{bends}} \right] \frac{\bar{V}_h^2}{2g} \quad (10)$$

Note that the major loss coefficient of,

$$K_{\text{major}} = f \left( \frac{L}{D} \right) = 21.65, \quad (11)$$

is larger than the valve and exit losses (recall that from the problem statement,  $K_{\text{bends}} \ll K_{\text{valve}}$ , and thus the bend losses may be neglected), with  $K_{\text{valve}}/K_{\text{major}} = 0.09$  and  $K_{\text{exit}}/K_{\text{major}} = 0.05$ . We should retain these minor losses since they are not small enough to be neglected.

Solving Eq. (10) for the hose average velocity gives,

$$\bar{V}_h = \sqrt{\frac{2g \left( \Delta z - \frac{p_{1,g}}{\rho g} \right)}{1 - f \left( \frac{L}{D} \right) - K_{\text{valve}} - K_{\text{exit}}} } \Rightarrow \bar{V}_h = 5.4 \text{ m/s} \quad (12)$$

The volumetric flow rate is,

$$Q = \bar{V}_h \frac{\pi}{4} D^2 \Rightarrow \boxed{Q = 1.6 * 10^{-3} \text{ m}^3/\text{s}} \quad (13)$$

Verify that the flow is indeed in the fully turbulent zone.

$$\text{Re}_D = \frac{\bar{V}_h D}{\nu} = \frac{(5.4 \text{ m/s})(1.91 * 10^{-2} \text{ m})}{1.0 * 10^{-6} \text{ m}^2/\text{s}} = 1.0 * 10^5 \quad (14)$$

This value of the Reynolds number is in the fully turbulent zone, so the assumption of fully turbulent flow was a good one.