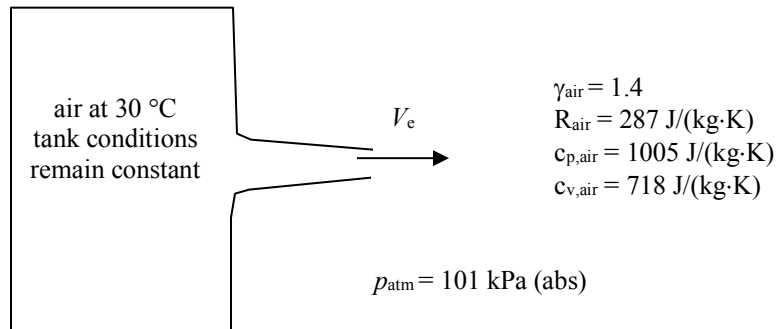


The large compressed-air tank shown in the figure exhausts from a nozzle at an exit velocity of  $V_e = 235$  m/s. Assuming isentropic flow, compute:

- the pressure in the tank
- the exit Mach number
- Now consider a case where the exit velocity is not given and the tank pressure is 300 kPa (abs). For these conditions, determine the exit flow speed,  $V_E$ .



SOLUTION:

First determine the exit Mach number using:

$$\text{Ma}_e = \frac{V_e}{c_e} \quad (1)$$

The exit speed of sound, assuming ideal gas behavior, is given by:

$$c_e = \sqrt{\gamma R T_e} \quad (2)$$

where, for an adiabatic flow:

$$T_0 = T_e + \frac{V_e^2}{2c_p} \quad (3)$$

Using the given data:

$$\begin{aligned} \gamma &= 1.4 \\ R &= 287 \text{ J/(kg}\cdot\text{K)} \\ T_0 &= 30 \text{ }^\circ\text{C} = 303 \text{ K} \\ V_e &= 235 \text{ m/s} \\ c_p &= 1005 \text{ J/(kg}\cdot\text{K)} \\ \Rightarrow T_e &= 276 \text{ K} \\ \Rightarrow c_e &= 333 \text{ m/s} \\ \Rightarrow \boxed{\text{Ma}_e = 0.71} \end{aligned}$$

Since the exit Mach number is subsonic, the exit pressure will be equal to the back pressure, *i.e.*

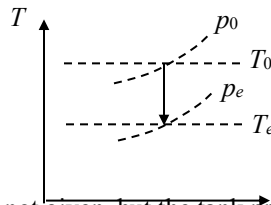
$$p_e = p_{\text{atm}} = 101 \text{ kPa (abs)}$$

Assuming isentropic flow:

$$\frac{p_e}{p_0} = \left(1 + \frac{\gamma - 1}{2} \text{Ma}_e^2\right)^{\frac{\gamma}{1-\gamma}} \quad (4)$$

Using the given data:

$$\Rightarrow \boxed{p_0 = 141 \text{ kPa (abs)}}$$



Now consider the case where the exit velocity is not given, but the tank pressure is given as  $p_0 = 300 \text{ kPa (abs)}$ . First determine whether or not the flow is choked. For a converging nozzle, the flow is choked if,

$$\frac{p_B}{p_0} \leq \frac{p^*}{p_0} = \left(1 + \frac{k-1}{2}\right)^{\frac{k}{1-k}} = 0.5283 \quad (5)$$

Using the given data ( $p_0 = 300 \text{ kPa (abs)}$  and  $p_B = 101 \text{ kPa (abs)}$ ),  $p_B/p_0 = 0.3367$ . Thus, the flow is choked for the given conditions and  $\text{Ma}_E = 1$ .

Since the exit is at sonic conditions, the speed of the flow there is,

$$V_E = V^* = c^* \underset{=1}{\text{Ma}^*} = \sqrt{kRT^*} \quad (6)$$

where

$$\frac{T^*}{T_0} = \left(1 + \frac{k-1}{2}\right)^{-1} \underset{k=1.4}{=} 0.8333 \quad (7)$$

Using the given data ( $T_0 = 303 \text{ K}$ ,  $k = 1.4$ ,  $R = 287 \text{ J/(kg}\cdot\text{K)}$ ),  $T^* = 253 \text{ K}$ , and  $\boxed{V_E = 319 \text{ m/s}}$ .