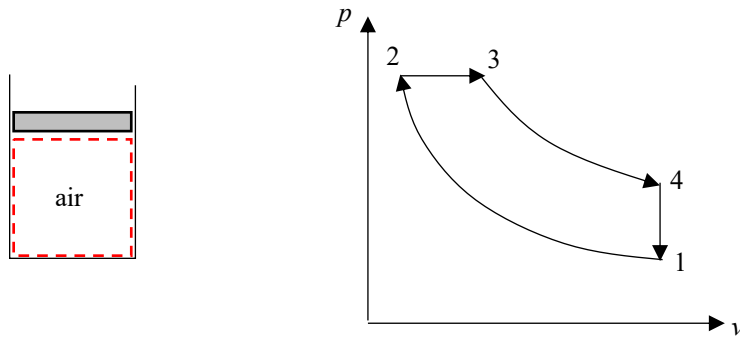


The displacement volume of an internal combustion engine is 3 L. The processes within each cylinder of the engine are modeled as an air-standard Diesel cycle with a cutoff ratio of 2.5. The state of the air at the beginning of compression is fixed by  $p_1 = 95$  kPa (abs),  $T_1 = 22^\circ\text{C}$ , and  $V_1 = 3.17$  L. Determine:

- a. the net work per cycle,
- b. the power developed by the engine if the cycle repeats 1000 times per minute,
- c. and the thermal efficiency of the cycle.

SOLUTION:



First, determine the mass of air in the cylinder using the ideal gas law,

$$m = \frac{p_1 V_1}{RT_1}, \quad (1)$$

Using the given values with  $R = 0.287 \text{ kJ/(kg.K)}$ ,

$$m = 3.5570 \cdot 10^{-3} \text{ kg.}$$

Now determine the properties at each state:

State 1:

$$p_1 = 95 \text{ kPa (abs)}, T_1 = 22^\circ\text{C} = 295 \text{ K}, \text{ and } V_1 = 3.17 \text{ L}$$

$$\Rightarrow u_1 = 210.5 \text{ kJ/kg} \text{ and } v_r(T_1 = 295 \text{ K}) = 647.9 \text{ (from the Ideal Gas Table (IGT) for air)}$$

State 2:

$$V_2 = V_1 - 3.0 \text{ L} = 0.17 \text{ L (given that the displacement volume is 3 L)}, \quad (2)$$

$$\frac{v_2}{v_1} = \frac{V_2}{V_1} = \frac{v_r(T_2)}{v_r(T_1)} \Rightarrow v_r(T_2) = v_r(T_1) \left( \frac{V_2}{V_1} \right), \quad (3)$$

where  $V_1 = 3.17 \text{ L}$ ,  $V_2 = 0.17 \text{ L}$ ,

$$\Rightarrow v_r(T_2) = 34.745 \Rightarrow T_2 = 896.15 \text{ K}, u_2 = 671.405 \text{ kJ/kg}, h_2 = 928.59 \text{ kJ/kg (interpolating in the IGT)}$$

The pressure may be found using the ideal gas law,

$$\Rightarrow p_2 = \frac{mRT_2}{V_2} \Rightarrow p_2 = 5381.37 \text{ kPa.} \quad (4)$$

State 3:

$$\text{The cut-off ratio is given as } r_c = 2.5 = V_3/V_2 = T_3/T_2 \Rightarrow T_3 = 2240.4 \text{ K}, V_3 = 0.425 \text{ L}, \quad (5)$$

$$\Rightarrow h_3 = 2553.87 \text{ kJ/kg}, u_3 = 1911.76 \text{ kJ/kg}, v_r(T_3) = 1.8925 \text{ (interpolating in the IGT)}$$

State 4:

$$\frac{v_4}{v_3} = \frac{V_4}{V_3} = \frac{v_r(T_4)}{v_r(T_3)} \Rightarrow v_r(T_4) = v_r(T_3) \left( \frac{V_4}{V_3} \right) = v_r(T_3) \left( \frac{V_4}{V_1} \cdot \frac{V_1}{V_2} \cdot \frac{V_2}{V_3} \right), \quad (6)$$

where  $V_4 = V_1$ ,  $V_1 = 3.17 \text{ L}$  (given),  $V_2 = 0.17 \text{ L}$  (Eq. (2)), and  $V_2/V_3 = 1/r_c = 1/2.5$  (Eq. (5)),

$$\Rightarrow v_r(T_4) = 14.1157 \Rightarrow T_4 = 1209.8 \text{ K} \text{ and } u_4 = 942.17 \text{ kJ/kg (interpolating in the IGT)}$$

The work into the air during the compression stroke is found by applying the 1<sup>st</sup> Law to the air (assuming negligible changes in KE and PE and an adiabatic process),

$$m(u_2 - u_1) = W_{in,12} \quad (7)$$

Using the previously calculated values,

$$W_{in,12} = 1.6394 \text{ kJ.}$$

Now calculate the work done by the air during the heat addition and power strokes using the 1<sup>st</sup> Law,

$$W_{out,23} = p_2(V_3 - V_2), \quad (8)$$

$$m(u_4 - u_3) = -W_{out,34} \quad (9)$$

Using the previously calculated values,

$$W_{out,23} = 1.3722 \text{ kJ and } W_{out,34} = 3.449 \text{ kJ}$$

The net work out is,

$$W_{out,net} = W_{out,23} + W_{out,34} - W_{in,12}, \quad (10)$$

$$\boxed{W_{out,net} = 3.18 \text{ kJ (This is the work over one cycle.)}}$$

Alternately, we could apply the 1<sup>st</sup> Law over the whole cycle, keeping in mind that the total energy does not change over the cycle,

$$0 = Q_{in,23} - Q_{out,41} + W_{in,12} - W_{out,23} - W_{out,34}, \quad (11)$$

$$0 = Q_{in,23} - Q_{out,41} - W_{out,net}, \quad (12)$$

$$W_{out,net} = Q_{in,23} - Q_{out,41}. \quad (13)$$

The heat transfer into the system during the combustion process is,

$$m(u_3 - u_2) = Q_{in,23} - p_2(V_3 - V_2), \text{ (noting that } p_3 = p_2), \quad (14)$$

$$Q_{in,23} = m(u_3 - u_2) + p_2(V_3 - V_2) = m(h_3 - h_2). \quad (15)$$

Using the previously calculated values,

$$Q_{in,23} = 5.7811 \text{ kJ.}$$

The heat transfer out of the system is,

$$m(u_4 - u_1) = -Q_{out,41}. \quad (16)$$

$$Q_{out,41} = 2.6025 \text{ kJ.}$$

Using the calculated heat values and Eq. (13),

$$W_{out,net} = 3.18 \text{ kJ, which is the same value found previously.}$$

The power is,

$$\dot{W}_{out,net} = \left( \frac{W_{out,net}}{1 \text{ cycle}} \right) \left( \frac{1000 \text{ cycle}}{1 \text{ min}} \right) \left( \frac{1 \text{ min}}{60 \text{ s}} \right), \quad (17)$$

$$\boxed{\dot{W}_{out,net} = 53.0 \text{ kJ/s} = 53.0 \text{ kW}.}$$

The thermal efficiency is,

$$\eta = \frac{W_{out,net}}{Q_{in}}, \quad (18)$$

Using  $W_{out,net} = 3.18 \text{ kJ}$  and  $Q_{in} = 5.7811 \text{ kJ}$ ,

$$\Rightarrow \boxed{\eta = 0.550 = 55.0\%}$$